# An Experimental Investigation of Aerodynamic Drag on a Round Parachute Canopy 

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#### Abstract

The objective of this research was to experimentally study the connection between the shape of a parachute canopy during inflation and the aerodynamic forces on the canopy. This was done by comparing the aerodynamics of a series of rigid parachute models, which are similar in shape to the flexible inflating parachute, against unsteady aerodynamics of the flexible parachute during inflation. A series of rigid models were designed, manufactured and tested to see if they could replicate the aerodynamic drag forces on a flexible parachute model inflated under infinite-mass conditions (constant freestream velocity). Experimental results indicate that aerodynamic drag forces on the flexible canopy at specific time instances during the inflation process cannot be replicated using rigid canopy models. These findings suggest that the aerodynamic drag forces on an inflating flexible parachute under infinite-mass conditions are a result of the dynamic motion of the canopy.


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## 1. Introduction

### 1.1 Background

Round parachute canopies are bluff-body aerodynamic decelerators. Aerodynamic decelerator devices are used to primarily decelerate and/or stabilize an object in freefall, although they are also used in certain ground vehicle applications. Parachutes are used as deceleration devices for airdrop of personnel and equipment and recovery of missiles, rockets and spacecraft. They can also be used to stabilize and retard the delivery of ordinance or to orientate a body in freefall before the primary deceleration system deploys. Today, parachutes have many applications. Parachutes are used to land autonomous exploration rovers on other planets, deliver supplies to natural disaster victims, recover rocket engines from the space shuttle and precisely deliver troops onto the battlefield using steerable ram air canopies. The round parachute canopy has played an important role in US Army operations involving placement of personnel and equipment.

The idea of using a high drag device to slow an object moving through air has been around for centuries. Some of the earliest sketches for this type of device were drawn by Leonardo da Vinci around 1485. In 1783, Sebastien Lenormand gave the name parachute to such drag devices. Jean-Pierre Blanchard demonstrated jumping from a hot air balloon using these parachutes in 1793. In 1797, Andre-Jacques Garnerin jumped using a folded silk parachute. Until the beginning of the twentieth century, parachutes were not all that practical and were mostly used for entertainment purposes similar to earlier eighteenth century hot air balloon jumps (Desabrais, 2002).

However, with the development of flight vehicles in the early 1900s, it became apparent that parachutes could be used for more than just entertainment. As parachute usage increased, a
better understanding of the relationship between parachute dynamics and performance characteristics was desired. As a result, in order to refine design and analysis methods, formal studies into the dynamics of parachutes began.

Parachutes can inflate under two types of freestream velocity conditions. These conditions are characterized as either an infinite mass or finite mass condition. It is known that the connection between parachute canopy shape and size as well as the forces on the canopy depend on the freestream flow conditions (Knacke, 1992). Inflation under finite mass conditions means that the velocity of the parachute system decays as it inflates. Eventually after terminal velocity, the velocity of the parachute system becomes relatively constant. The canopy breathing does cause variations in drag forces and subsequently the velocity of the system. This condition is known as finite mass condition.

Parachutes also operate in the infinite mass condition during inflation for certain applications. For example, when a parachute is used for stabilization purposes, the velocity of the airflow over the canopy remains constant during inflation. The deceleration of the system due to the drag force of the stabilization canopy is small and can be neglected and therefore infinite mass conditions can be assumed.

Talyor (1963) performed early experiments using parachutes as air brakes for landing aircraft. Future studies involved investigating the inflation time of parachute canopies. Müller and others conducted experiments which concluded that for geometrically similar canopies, the distance over which the canopy takes to inflate will be the same (Müller, 1927; French 1963; Heinrich, 1969; Heinrich \& Noreen, 1970; Heinrich, 1972). French (1963) attempted to correlate the peak opening forces and the opening times of various parachute geometries. These theories helped designers determine the amount of time it takes a canopy to inflate. The initial
theories measured the drag force on the canopy using Newton's second law. The equations used both steady and unsteady mass terms to describe the drag force on the canopy during inflation.

## Finite-Mass Equation of Motion (French)

$$
\begin{equation*}
F=m a=m g \sin \theta-\frac{1}{2} \rho v^{2} C_{D} \pi r^{2} \tag{1}
\end{equation*}
$$

## Equation of Motion (Heinrich)

$$
\begin{equation*}
\left(\frac{d}{d t}\right)\left[\left(m_{s}+m_{p}+m_{i}+m_{a}\right) V\right]=\left(W_{s}+W_{p}\right) \sin \alpha-D_{s}-D_{p c} \tag{2}
\end{equation*}
$$

When the following assumptions are used:

- suspended weight is much greater than the weight of the parachute canopy
- $W_{s} \gg W_{p}$
- drag of the parachute canopy is much greater than the drag of the suspended weight - $D_{p c} \gg D_{s}$
- and when the angle to the horizontal is either zero or perpendicular
- $\alpha=0$ or $\alpha=\pi / 2$

The equation of motion simplifies to:

$$
\begin{equation*}
F=m_{s} \frac{d V}{d t}=m_{s} g-\frac{\rho}{2} C_{D} S V^{2}-V\left(\frac{d m_{i}}{d t}+\frac{d m_{a}}{d t}\right)-\left(m_{p}+m_{i}+m_{a}\right) \frac{d V}{d t} \tag{3}
\end{equation*}
$$

The infinite mass condition is approximated by setting $\mathrm{m}_{\mathrm{s}} /\left(\mathrm{m}_{\mathrm{p}}+\mathrm{m}_{\mathrm{i}}+\mathrm{m}_{\mathrm{a}}\right)=100$

$$
\begin{aligned}
& \mathrm{W}_{\mathrm{s}}=\text { supsended weight } \\
& \mathrm{W}_{\mathrm{p}}=\text { weight of parachute canopy } \\
& D_{\mathrm{pc}}=\text { drag of parachute canopy } \\
& D_{\mathrm{s}}=\text { drag of suspended weight } \\
& \rho=\text { density } \\
& \mathrm{S}=\text { canopy projected area } \\
& v=\text { instantaneous velocity }
\end{aligned}
$$

$\mathrm{m}_{\mathrm{s}}=$ mass of suspended weight
$\mathrm{m}_{\mathrm{p}}=$ mass of parachute
$\mathrm{m}_{\mathrm{i}}=$ mass of included air
$\mathrm{m}_{\mathrm{a}}=$ apparent mass
$\mathrm{V}=$ velocity
$C_{D}=$ drag coefficient
$r=$ instantaneous max. projected radius

Additionally, investigations aimed at describing the mass of the flow around a parachute canopy were conducted for finite mass inflation conditions (Ibrahim, 1967; Eaton, 1983; Yavuz, 1989). Much of this research has focused on using experimental data and analytical theories to predict important design parameters such as opening time and maximum opening force (French, 1964; Knacke, 1992). However, these theories used simplifications to avoid dealing with the complex fluid dynamics in the flow field near the canopy. Computational models of parachute canopies have been created to study flow around fully inflated canopies (Stein, 1999; Stein et al., 1999; Stein et al., 2000). Additionally, experiments have been conducted to determine drag coefficients for a number of bluff body objects. For example, Hoerner (1958), among others, has experimentally determined drag coefficients for a number of objects including shapes similar to those of an inflating parachute canopy such as disks, cylinders and cups. Other recent research related to the relationship between forces on the canopies and the surrounding flow field includes investigations into the study of flow in the near wake of a parachute canopy (Johari, et al., 2001; Desabrais, 2002)

It is evident that there has been a sustained interest in understanding the dynamics of parachute systems for almost a century. Since 1964, parachute researchers have gathered at the biennial AIAA (American Institute of Aeronautics and Astronautics) Aerodynamic Decelerator Systems Technology Conference to present and discuss developments in this field.

Successful parachute design requires tailoring important performance parameters to meet the mission requirements or design specifications. In the design of a parachute canopy, the most significant parameters are parachute stability, peak opening force, opening (or filling) time, and steady-state drag. "These parameters are typically obtained from full-scale testing of parachute prototypes. Full-scale testing for design purposes is time consuming and not cost effective."
(Johari, 2003) "Because of the complexity of the fluid dynamics involved, advances in parachute technology rely heavily on experimentation." Recent parachute research efforts have focused on gaining a better understanding of the parameters affecting parachute dynamics under inflation and fully inflated conditions. Driving this research is the desire to reduce the time, development costs and uncertainties associated with the design of new parachute systems.

At the present time, few studies have taken a comprehensive look at the aerodynamic forces on a parachute canopy and their relationship to the canopy shape. An investigation of the relationship between canopy shape and the aerodynamics on an inflating round parachute canopy may provide results useful to the effort aimed at enhancing how new parachute systems are designed.

### 1.2 Objective

The objective of this project was to experimentally study the connection between the shape of a parachute canopy during inflation and the aerodynamic forces on the canopy by comparing the aerodynamics of a series of rigid parachute models, which are similar in shape to the flexible inflating parachute, against unsteady aerodynamics of the flexible parachute during inflation.

To accomplish this objective, solid models of a round parachute canopy at different stages of inflation were designed, manufactured and tested to see if the drag forces on rigid models are comparable to the flexible canopy model loads during inflation.


Figure 1 - Drag force vs. time graph of a flexible canopy (constructed diameter = $\mathbf{3 0 . 5} \mathbf{~ c m}$ ) tested in a water tunnel with a freestream velocity of $20 \mathrm{~cm} / \mathrm{s}$.

This investigation will help determine whether the drag forces on a round parachute canopy are related to the actual shape of the canopy or if they are related to other factors such as the time rate of change of the canopy's shape during inflation.

### 1.3 Parachute Analysis

In the study of parachute aerodynamics, certain parameters are used to characterize the environment in which the parachute is operating and the resultant forces on the canopy. The Reynolds number is a parameter correlating the viscous behavior of Newtonian fluids.

Newtonian fluids exhibit a linear relationship between applied shear force and viscosity. The

Reynolds number is non-dimensional with a density, viscosity, velocity, and length scale parameter.

$$
\begin{equation*}
\operatorname{Re}_{D \max }=\frac{\rho U_{\infty} L}{\mu} \tag{4}
\end{equation*}
$$

The drag force on a bluff body such as a parachute canopy is usually normalized with dynamic pressure and a characteristic area for comparison purposes. This value is defined as a drag coefficient $C_{D}$.

$$
\begin{align*}
C_{D} & =\frac{D}{q A}  \tag{5}\\
q & =\frac{1}{2} \rho U^{2} \tag{6}
\end{align*}
$$

Drag coefficients for bluff bodies similar in shape to those of an inflating round parachute canopy include disks, cones, and cups. Hoerner reported drag measurements on a wide variety of bluff bodies. In his book Fluid-Dynamic Drag, Hoerner lists experimentally determined drag coefficients for 2-D and 3-D objects such as for caps, cones, cups and cylinders in flow characterized by Reynolds numbers between $10^{4}$ and $10^{6}$. The shape of an inflating parachute canopy is similar to these shapes at different stages during the inflation process. There is a drag curve for 3-D sheet-metal "caps" for varying $C_{D}$ for Re numbers between $10^{5}$ to $10^{6}$. These Reynolds numbers are scaled using the diameter of the cap, d.


Figure 2 - Fluid Dynamic Drag Summary

According to Hoerner, in this Reynolds number range, variations in drag coefficient values depend much more heavily on the ratio of height to diameter of the cup or cap than it does on the precise Re number value. These shapes also experience separated flow and as a result of the negative pressure on the side opposite to the freestream flow, drag coefficients are noticeably higher when the concave surface meets the freestream flow as opposed to having the convex surface placed normal to the flow. In addition to caps, Hoerner has drag coefficient data for cones.

In his dissertation entitled, Velocity Field Measurement in the Near Wake of a Parachute Canopy, Desabrais (2002) tested small circular parachute models in a water tunnel. Typically, parachutes are constructed by sewing individual panels (gores) together to form a circular geometry (Knacke, 1992). It was decided that constructing scale models this way would make the canopies too stiff. For Desabrais' experiments, two different size canopies were constructed from a circular sheet of $1.1 \mathrm{oz} / \mathrm{yd}^{2}$ rip-stop nylon. The single sheet of nylon cut in the shape of a circle forms the canopy. Two different canopy sizes were used in these experiments. Twentyfour suspension lines were then attached at evenly spaced intervals along the edge of the circular sheets.

Data from Desabrais' dissertation experiments contain image sets and corresponding drag data for canopies with constructed diameters of both 15.2 cm and 30.5 cm . Data collected for these canopies during inflation at $19.6 \mathrm{~cm} / \mathrm{s}$ freestream water tunnel velocity over a time interval of approximately 10 seconds consists of 293 images and 2250 drag force measurements. The Reynolds number corresponding to the 15.2 cm canopy was 29,800 scaled to the canopy constructed diameter and $\operatorname{Re}=55,900$ for the 30.5 cm canopy. The force versus time plots for both the 15.2 cm and 30.5 cm constructed diameter canopies share similar characteristics.


Figure 3-Opening force and diameter for a 15 cm canopy with freestream velocity of $20 \mathrm{~cm} / \mathrm{s}$ in a water tunnel.


Figure 4 - Opening force and diameter for a 30 cm canopy with freestream velocity of $20 \mathrm{~cm} / \mathrm{s}$ in a water tunnel.

For a direct comparison of drag forces on the proposed rigid parachute models to the drag forces on the flexible canopy models tested by Desabrais, it would be necessary to test models under dynamically similar conditions to the flexible water tunnel models. There are limitations imposed on this drag force comparison by testing in a wind tunnel as opposed to a water tunnel. First, in order to compare data, conditions must be dynamically similar. This similarity depends on Reynolds number, since Reynolds number is a characteristic of the flow. As long as Reynolds numbers are of the same order, a direct comparison of normalized drag data can be considered reasonable.

In studies of drag on rigid bluff bodies by Hoerner, it is seen that drag coefficients for bluff bodies can be quite different depending on Reynolds number. For example, a plot of experimental drag coefficients against Reynolds number for a sphere (and other objects) shows that there are transitional points where the drag coefficient can change value rapidly. For a sphere, one very noticeable transition occurs at $\operatorname{Re}_{\mathrm{d}} \approx 4.7 \times 10^{5}-5.5 \times 10^{5}$ where $C_{D}$ drops from .8 to .2. Therefore, if the Reynolds number ranges of two different experiments differ, it must be assumed that there could be a possible Re range between these two experimental sets where drag coefficient could drastically change. Caution must be exercised when comparing two sets of experimental data when a gap exists.

Due to equipment availability at WPI, it was decided that the solid parachute canopy models would be tested in a closed-loop (full-return) wind tunnel. Closed-loop wind tunnels are generally considered best suited for obtaining aerodynamic data since the velocity distribution in the test section stays more uniform (Knacke, 1992). Drag forces would be measured by a linear displacement transducer dynamometer.

When testing in a wind tunnel, the amount of test section blockage generated by the test article must be taken into consideration when analyzing test data. In an unrestricted test section with no test article, flow is uniform ignoring boundary layer interactions at the walls. When a test article is placed in the test section, flow streamlines are deflected around the test body. However, the deflected flow is constricted by the test section walls, which interferes with the flow around the test body (Macha \& Buffington, 1989). As a result, the data must be corrected to account for this wall interaction.

As tunnel blockage increases, the effects due to the change in the streamlines and properties of the flow field can produce questionable test data. In order to avoid these effects, it is suggested that the test article not block more than $6 \%$ of the test section cross-sectional area (Knacke, 1992). However, a study published in 1989 from Sandia Labs suggests that it is possible to correct test data for parachutes tested in a wind tunnel with blockage ratios up to $30 \%$ (Macha \& Buffington, 1989).

According to this study, the results are believed to be applicable to any circular parachute. The results provide blockage correction factors based on Maskell correction methods for three-dimensional, non-lifting bluff bodies. According to Maskell, the effective increase in dynamic pressure due to blockage is given by

$$
\begin{equation*}
\frac{q}{q_{u}}=1+K_{M}+\frac{C_{D} S_{u}}{C} \tag{7}
\end{equation*}
$$

with C being the tunnel cross-sectional area, $C_{D} S_{u}$ being the model drag area (D/q), $K_{M}$ being the Maskell bluff-body blockage factor, and $q$ being the freestream dynamic pressure. The subscript u denotes an uncorrected value. This correction factor is based on the frontal area normal to the freestream flow.

## 2. Canopy Model Design Process

### 2.1 Canopy Model Design Constraints

It was previously mentioned that the purpose of this project was to compare the drag characteristics of solid canopy models to the drag characteristics of the Desabrais (2002) flexible parachute canopies. To accomplish this task, two-dimensional digital images provided by Desabrais needed to be converted into solid models using imaging software, computer aided design software, and computer aided manufacturing software packages. Throughout this process, the design was constrained by the capabilities of the wind tunnel and drag force measurement equipment. These constraints were wind tunnel test section blockage, drag force measurement capabilities, and wind tunnel testing velocity. All design considerations needed to take into account the implications that they had on these three constraints.

The first constraint, wind tunnel test section blockage, is the ratio of the projected area of the model and the cross sectional area of the wind tunnel test section. It was known that any sized test article placed in a wind tunnel creates blockage effects that induce flow disturbances. This is a result of the flow going around the model and interfering with the test section walls. Ideally, tunnel blockage should be chosen to be less than five to six percent (Knacke, 1992). However, although tunnel correction methods have been shown to accurately correct data for blockages up to $30 \%$, Macha suggests that tunnel blockage be no more than $10 \%$ to ensure reliability when using this method (Macha \& Buffington, 1989). Therefore, this constraint limited the projected area of the model to $10 \%$ of the wind tunnel test section cross sectional area. In this case, the 2 ft X 2 ft test section had a cross-sectional area of $4 \mathrm{ft}^{2}$ resulting in a maximum model projected area of $0.4 \mathrm{ft}^{2}$, or a maximum canopy diameter of 8.56 in . Due to the
fact that both drag coefficient and Reynolds number are calculated using the diameter of the canopy, this limitation would have direct affects on those constraints as well.

The second constraint was the magnitude of drag that could be accurately measured by the dynamometer. After referencing the instruction manual for the dynamometer, it was found that the maximum drag force that could be measured without damage to the instrumentation was 7 lbf . It was also ascertained that it could accurately read $1 / 20^{\text {th }}$ of that maximum drag force, a value of 0.35 lbf . Therefore, the drag force measurement instrumentation could not be subjected to forces outside the range of $0.35-7 \mathrm{lbf}$ for accurate readings. This constraint also limited the size of the model, due to the fact that drag force is calculated using the projected area.

The final constraint on the model designs was the wind tunnel testing velocity. It was known that cup shaped bluff bodies tend to vibrate due to vortex shedding when subjected to the freestream. These shedding cycles tend to become more violent as the freestream velocity is increased. Therefore, to avoid damage to the models and wind tunnel equipment due to model or dynamometer failure, the wind tunnel velocity needed to be monitored. Changes in wind tunnel velocity directly affect the drag force measurements and Reynolds number calculations.

It can be seen that these three constraints are interrelated. A change in design to compensate for one limitation will directly affect one or both of the other restrictions. Therefore, this design progression was an iterative process, meaning that the sizing of the models was modified until all design boundaries were satisfied.

### 2.2 Drag Force Time History Curve Representation

With the constraints outlined, a method for creating three-dimensional models from the two-dimensional images could be formulated. The first design consideration was how to accurately represent the drag force time history plot using solid models. It was known that the curve was created from approximately 1,500 data points. To attempt to recreate the curve perfectly would require the production of 1,500 rigid models. This would prove to be an impossible task given the time constraints.

Therefore, it became necessary to determine a means for representing the complex curve by creating a feasible number of rigid models. One possible method for representing this curve was to create models at specific data point intervals. This would effectively reduce the number of rigid models. Another option was to choose points on the time history that, when plotted separately, could accurately reflect the dynamic nature of the inflating canopy. One final way to accurately reflect this curve was to create single models that could represent more than one data point on the time history plot. This could be accomplished by making each model adjustable in some way.

The problem with selecting models based upon data point intervals is that they may not accurately capture the key points on the time history plot. For example, if models were made at each second during the canopy inflation, the opening shock peak that can be seen at about 2.5 seconds would not be represented. The drawback of the second method was the number of models needed to represent the curve. Visual inspection of the curve determined that approximately 50 models would be needed. Lastly, the complexity of designing an adjustable rigid canopy model that accurately reflected the flexible canopy at different inflation times was deemed impractical. This was due to the fact that the design process of this single model would
take more effort than simply making a greater quantity of simpler models. Also, given the fact that no previous attempts have been made to manufacture such a model, it was unknown if it could accurately represent each canopy model as well as a set of rigid models could.

With these options exhausted, the research team decided it was important to highlight critical changes in drag during the inflation process. These critical changes were identified to be the beginning of inflation when the canopy experienced relatively constant drag forces, the opening shock peak, the point of canopy overexpansion, and the fully inflated region. These four points became the basis for rigid canopy models.


Figure 5- Four points critical in representing the force time history curve (points: just as canopy starts to inflate, peak force, over expanded, steady-state)

With the four points of greatest interest accounted for, it became necessary to determine the number of additional models that could be made to replicate the other parts of the curve.

Upon further inspection of the force time history, it was determined that a much better curve representation could be achieved using just four more points. One point was chosen in the region where the canopy has not begun to inflate, two additional points were selected before the peak force as the opening force is increasing, and one other point was selected after the peak force as the opening force is decreasing. It is important to note that the points chosen immediately before and after the peak force experience the same drag value. These points are shown in Figure 6.


Figure 6- Points selected for model creation along the force time history plot.

These eight points captured the main features of the force time history of the inflating flexible canopy. Clearly, more points could be used for a better approximation. However, this
analysis showed that the main features of the force time history could be captured using at a minimum of eight points.

### 2.3 Canopy Curve Estimation Using Digital Images

As mentioned earlier, two-dimensional digital images were provided to the research team. These digital images were instantaneous photographs of the canopy during the inflation process. Each digital image included the shape of the canopy, the time into the opening process, and the location of this time on the opening force time history graph. With the time and location of each image on the drag force time history plot known, it was possible to obtain the canopy image for each of the points chosen above. These images became the starting point for solid model generation, and the corresponding canopy shapes can be seen in Figure 7.
Shape 1


Figure 7 - Digital images of the canopy shapes chosen and their location on the drag force time history plot.

The method of creating these three-dimensional models from the images, however, presented another design challenge due to the fact that each picture only depicted an instantaneous image of one side of the canopy. Additionally, the gores on the canopy were not uniform, and the images only showed the curves for two of the twenty-four gores. It was also unknown if the gores depicted were offset from the visual plane.

Due to the complexity of the folding, and the lack of a $360^{\circ}$ view for each canopy shape, a method to approximate these gore curves was needed. With no imagery of the backside of each canopy, it was impossible to know exactly how the gores were shaped all around the parachute. Using the images available, the best approximation of the gore curves was to consider them symmetric, evenly spaced and planar with the images. This resulted in a canopy consisting of twenty-four symmetric gores spaced $15^{\circ}$ from midpoint to midpoint. Even if the centers of the gores depicted were not perfectly planar with the images, the maximum the centers could be offset was $7.5^{\circ}$. Considering that the canopy gores change at each time instance during the inflation process, the gores are never exactly the same shape. Therefore, any offsets are negligible because of the variation of each gore shape.

Further work with the digital images required the selection of an image software package. Various software packages allow the user to retrieve information about pixels in digital images. These include Microsoft Paint, Adobe Photoshop, and MATLAB.

After experimentation with each of the aforementioned software packages, Microsoft Paint was chosen for its ease of use and well-suited coordinate system, a characteristic not shared by the other, more cumbersome, software packages. The images could be aligned with an $\mathrm{X}-\mathrm{Y}$ origin, allowing the $\mathrm{X}-\mathrm{Y}$ coordinates of any pixel to be recorded. The X direction was in the radial direction, and the Y direction was in the axial direction. Each canopy outline was traced
one half at a time with a high zoom percentage to see the exact edge of the shape. The curves of the canopy were determined by finding the X coordinates of points using predetermined Y coordinates. Due to the inconsistency of the curve from the right side to the left side, the two sets of points were averaged to create one smooth curve for the parachute canopy. This smooth curve was necessary for solid modeling using a computer aided design (CAD) package and for a better approximation of the gore shape. In total, each canopy shape was determined by finding approximately 135 points for each outside curve.

A similar procedure was used to estimate the inside curves resulting from the canopy gores. The transparency of the canopy fabric allowed for the determination of the depth of the gore folds. The image below shows a line transposed onto the image estimating the gore folding. The points of these curves were recorded, and once more averaged over each side of the canopy. Again, these gores were estimated using the same assumptions as the outside gore curves.


Figure 8 - Digital image of canopy 8 showing outside curve and gore curve

### 2.4 Sizing and Scaling of Canopies from Curve Estimations

With the pixel coordinates for each canopy curve determined, it became necessary to convert them into length coordinates. The length scale chosen determined the size of the models. Several interrelated design considerations were then encountered. As previously mentioned, the size of the models directly affected the wind tunnel blockage, the drag force measurements, and the Reynolds number calculations. This is because each of these factors is based on the frontal area. The scaling of these models was bounded by $10 \%$ tunnel blockage, $0.35-7.0 \mathrm{lbf}$ drag measurements, and wind tunnel velocities suitable for testing.

To reduce the adverse effects of tunnel blockage on collected data, reduce the need for larger amounts of correction thereby maximizing the reliability of drag force data, the project team first attempted to scale the parachute models to five percent tunnel blockage. This meant that the projected area of the largest model could not exceed five percent of the test section area. This resulted in a maximum projected diameter of 6.056 in. for the largest model. In this case, model seven had the largest diameter. Using the $\mathrm{X}-\mathrm{Y}$ coordinates determined earlier, the largest value of X was found, and scaled to 6.056 in . This scaling factor was used for the rest of the points on shape seven, and also for all of the points of the remaining seven models.

With model seven scaled to five percent tunnel blockage and the other models scaled accordingly, the expected drag force measurements and corresponding velocities could be calculated for each model. This analysis was performed using Equation (4) and Equation (5) to determine estimated drag forces on each canopy for the range of possible tunnel testing velocities and a range of possible drag coefficients. Using Equation (4) for a given range of Reynolds numbers, the kinematic viscosity for air, and the scaled maximum projected diameter of each model, the freestream velocity could be calculated. Once the freestream velocity was known,
drag force was found using Equation (5) and values for estimated canopy drag coefficients, the calculated freestream velocity, projected area, and the density of air.

It was found from this analysis that scaling the models to five percent tunnel blockage resulted in extremely small drag forces for the smaller projected diameter models. The freestream tunnel velocity required to achieve measurable forces for these models needed to exceed $70 \mathrm{~m} / \mathrm{s}$. This speed is much greater than the tunnel speeds deemed safe for testing bluff bodies in the wind tunnel. Therefore, scaling to five percent tunnel blockage was not feasible.

A maximum tunnel blockage of ten percent for the rigid models was then investigated. Although not ideal, correction methods for blockages of ten percent or less exist and have been proven to successfully correct for tunnel blockage flow disturbances (Knacke, 1992). Applying the same analysis methods described above, it was determined that measurable drag force could be obtained for seven of the eight models using tunnel speeds between 15 and $30 \mathrm{~m} / \mathrm{s}$. Scaling the models to ten percent blockage resulted in a test velocity range of 15 to $25 \mathrm{~m} / \mathrm{s}$, and measurable drag forces for all but the smallest model. Shape one was chosen very early in the opening process, resulting in a relatively small projected area. This small projected area, in turn, produced little drag force at reasonable tunnel speeds. The tunnel blockage could be increased to include this model, however, it was decided not to do so because historically, tunnel blockages greater than ten percent have resulted in unacceptable flow disturbances (Knacke 1992). Although there are proven correction methods for flexible parachute canopies having tunnel blockages greater than $10 \%$, the accuracy achieved by applying these methods to rigid canopy models could not be guaranteed (Macha \& Buffington, 1989). See Appendix C for the estimated drag force analysis.

### 2.5 Computer Aided Design of Scaled Models

Once the scaled estimations of the canopy curves were determined, solid modeling of the shapes could begin. Pro/ENGINEER Wildfire 2.0 solid modeling software was utilized due to its availability on campus, and the project team's familiarity with its operation.

To create the solid model, the scaled $\mathrm{X}-\mathrm{Y}$ coordinates of the outside curve were imported. These points were used to create a datum curve. Next, the scaled inner curve X-Y coordinates were imported, and used to create a datum curve rotated $7.5^{\circ}$ along the center axis from the previous curve. The outside datum curve was then mirrored on the other side of the inside curve.


Figure 9 - Pro/E Datum Curves

This created the $15^{\circ}$ quilt of one gore section.


Figure 10 - Pro/E Drawing of Gore
This gore was revolved $360^{\circ}$ to form the round canopies. The 24 separate gore quilts were then merged together forming one quilt. The quilt was then thickened to 0.05 in ., the desired thickness for the canopy models. This thickness was chosen, because it is the thinnest that the Computer Numeric Control (CNC) machines can machine without damaging the model according to Dr. William Weir (personal communication, November, 2005). The model was then solidified to create a solid model.


Figure 11 - Pro/E Solid Canopy

Keeping in mind that machining was a likely method of construction, in addition to modeling the canopy shape, a fixture was needed for holding the model during machining. To reduce flow disturbance caused by fixtures, it was decided to attach the canopy to the dynamometer at its center on the apex of the canopy. Additionally, it was decided to mount the parachute model in front of the dynamometer, to reduce flow disturbance created by the dynamometer structures.

Next, an investigation into feasible holding methods was conducted. This investigation concluded that a cylindrical boss extending from the top of the parachute with a smooth round to the canopy outer curve would be the strongest holding method that would minimize changes in canopy aerodynamics for drag testing. After investigating the collet holders available in the WPI Washburn machine shop for work holding, a diameter of one inch was chosen for the boss. Oneinch collets were available on all of the lathes and mills, allowing smooth transfer from one machine to another during manufacturing. Bosses were then modeled and added to the Pro/E canopies.


Figure 12 - Pro/E Solid Canopy with Boss

### 2.5 Dynamometer Sting Design

With the final solid models completed for each canopy shape, various analyses could be performed within Pro/ENGINEER to determine the models' masses, and centers of gravity. This information was needed for the sting design. The weight of the model hanging on the sting and the drag force on the model during testing created a moment about the dynamometer. To ensure the sensitive electronics within the dynamometer were not damaged, an analysis of induced moments would be needed.

The weight of the model could be found by multiplying its mass by gravity. The moment arm was calculated by choosing a sting length from the dynamometer to the top of the boss, and then adding the distance from the top of the boss to the center of gravity. Sting lengths of one inch to twelve inches were investigated for each model. An oscillatory moment was accounted for in these calculations due to expected vortex shedding. This moment was estimated to be an additional one third of the total drag force on the model (Johari, personal communication, 2005). Maximum moments found for aluminum and acetal when oscillatory forces are included were $6.13 \mathrm{ft}-\mathrm{lbs}$ and $6.28 \mathrm{ft}-\mathrm{lbs}$, respectively. These calculations can be found in Appendix A. The manufacturer of the dynamometer was then contacted to determine if these moments were too great for the dynamometer to withstand. According to Kurt Banaszynski at Engineering Laboratory Design (personal communication, 2005), the moments expected in this experiment were insignificant relative to the capabilities of the dynamometer.

The only remaining design criteria to determine were the diameter, threading, length, and material of the sting. Because weight was not determined to be an issue, a steel rod was used due to its strength. Later physical tests with the heaviest model showed that the diameter of the rod for shapes two through eight should be no less than $5 / 16$ in. to maintain sting rigidity. A
larger sting diameter could be used to reduce vibrations, but in the interest of causing the least flow disturbance, the $5 / 16 \mathrm{in}$. rod was selected. Model one was light enough to only require a $1 / 4 \mathrm{in}$. rod. One end of the rod was threaded with 1.5 inches of $1 / 4-20 \mathrm{in}$. thread for the dynamometer, and the other was threaded with one inch of 5/16-18 in. thread for the model bosses. Three stings were made from $5 / 16$ in. rod for models $2-8$ with varying lengths. The lengths of the non-threaded part of these stings were 2 in., 5 in., and 8 in.

Hardware required for mounting the sting to the model boss and dynamometer were one 5/16-18 nut, one 5/16 in. lock washer, two 1/4-20 in. nuts, and two $1 / 4 \mathrm{in}$. lock washers. The 5/16-18 in nut and 5/16 in. lock washer were used to tighten against the boss, reducing the chances of the model vibrating off of the threads. Similarly, the two $1 / 4-20 \mathrm{in}$. nuts and $1 / 4 \mathrm{in}$. lock washers were required to prevent the sting from unthreading from the dynamometer. The completed sting design can be seen in Figure 13.


Figure 13: Completed sting design mounted to dynamometer.

## 3. Construction

### 3.1 Construction Options

There were a number of construction methods available for building the rigid parachute models. The selection criteria for choosing a construction method was based on facility availability, quality of construction, ease of manufacturing, construction cost, and manufacturing time. Construction methods considered included CNC (computer numeric control) machining, rapid prototyping, fiberglass construction using machined molds and MonoKote covered wire frames. The construction method was chosen based on these criteria.

The construction methods of CNC machining and stereolithography rapid prototyping use 3-D CAD (computer aided design) models to machine or prototype physical models. The CNC machining would involve cutting models out of large pieces of stock to create models while stereolithography prototypes are created layer by layer with a photosensitive resin. The creation of molds to create fiberglass models would also be achieved through a CAD model. Molds would then be made by a CNC machine or through stereolithography and then used to shape the fiberglass. Another construction method considered was building models using frames covered in MonoKote, a durable heat shrinkable film. The frame could be machined from a solid piece of material or assembled from individually machined parts.

The first criterion for choosing a construction method was facility availability. CNC machining facilities in the WPI Washburn Shops and mold making facilities in the Metals Processing Institute were available for student use. Also, the shops had well qualified staff and professional machinists. In addition, the Natick Soldier Center (NSC) Air Drop/Aerial Delivery Group had access to an in-house rapid prototyping facility. All construction options except for
stereolithography could be done at WPI facilities. Therefore, facility availability would not be an obstacle in the construction of the rigid parachute models.

The next consideration was the quality of construction. With CNC machining and stereolithography, the models would be directly based on the generated CAD models. These manufacturing methods can easily hold tolerances to a few thousandths of an inch. Under conditions where part vibration is held to a minimum, a smooth surface finish can be achieved using machine tools. With stereolithography, it is relatively simple to obtain a polished or paintable surface finish (SLA Finish Levels, 2005). If molds were used to shape fiberglass, it would be difficult to hold the tolerances achieved through CNC machining or stereolithography. The molding process could leave an imprecise shape around the canopy skirt. In addition, the complex canopy geometries would require a two piece mold to allow the finished model to be separated after forming. When the frame method was analyzed, it was also determined that it would be very difficult to design a frame that could mimic a parachute canopy's complex geometry as accurately as the project specifications required.

In terms of manufacturability, the preliminary estimates for creating a mold showed that it would take almost the same amount of time to machine a mold as it would to machine an actual model. Again, manufacturing time estimates showed that it would probably take more time to machine a skeleton frame from a piece of solid stock than it would to machine an actual model. If instead, individual parts of the frame were machined and then put together, this would require building an assembly jig for each model. After this initial comparison, CNC machining and stereolithography rapid prototyping appeared to be the best manufacturing options when compared to forming fiberglass in molds or using MonoKote covered frames.

In terms of ease and convenience of manufacturing, the use of CNC machining was considered to be less desirable than stereolithography due to the intricate geometry of the parachute models. This complex model geometry would require multiple nonstandard part orientations, intricate machine setup and specialized tooling. There would also be problems associated with holding stock material in the CNC machines to minimize part vibrations during the cutting process.

Also, project specifications required that the model walls be as thin as possible. When machining a part, the less material there is to support the cutting surface, the more prone it becomes to vibrations and cracking. As a result, tailoring CNC G-Code and/or designing fixtures to hold and support thin, nonstandard shapes in order to minimize vibrations becomes a major manufacturing consideration. Trying to modify a CNC program to minimize vibrations for these complex model shapes is difficult for even experienced machinists. Creating fixturing to support these complex shapes is another design project in and of itself.

Therefore, to avoid the costly and time intensive process of designing and building support fixtures, it was realized that manufacturing these parachute models would require a machinist's attention while the CNC programs were running. This monitoring was necessary to make slight adjustments in cutter speeds and feed-rates.

Stereolithography is used to create a three-dimensional part directly from a CAD file. The only intermediate process is converting the CAD file to a .STL file. In Pro/E, this is a simple "save as" option. This file breaks the CAD model into slices.


Figure 14 - Pro/E Converted STL model
This file is then used by the stereolithography machine to build the model. Although the models would not be extremely durable, construction time was estimated to be about three weeks. The autonomy of a stereolithography machine and the speed at which it could build these models makes it a better choice than CNC machining. The simplicity of stereolithography however, comes with a sacrifice. The cost of a single model using stereolithography at the NSC was estimated to be $\$ 750-\$ 1000$ (personal communication Lee, 2005). The research budget allocated for this project could not cover this expense.

Based on ease of manufacturing, construction time, quality of construction, and facility availability, stereolithography was determined to be the best manufacturing method for meeting project design objectives and schedule requirements. However, project budget restrictions prevented the use of this ideal manufacturing option. Based on selection criteria, the next best option was to use CNC machining to build the eight rigid parachute models.

### 3.2 Material Selection

With the manufacturing method chosen, it was necessary to choose a stock material. The model material was selected based on availability, cost, cut time and material properties including achievable surface finish, durability and strength.

Commonly machined materials include metals and plastics. Aluminum is the most commonly machined metal. Compared to steel, it is light weight, costs less and is easy to machine. After speaking with machinists and material supply companies, it was determined that a type of plastic called acetal, a commonly machined plastic, could satisfy the parachute model thin wall requirements while maintaining structural rigidity. Acetal is more commonly known by the brand name Delrin, a trademark of the DuPont Company. These materials were also chosen because the machinists in the Washburn Shops had experience machining these materials.

Design requirements dictated that the selected material be able to withstand aerodynamic and dynamometer (sting) attachment forces during wind tunnel testing. The density of the material would also have to be considered, as the weight of the parachute must not damage the transducers within the dynamometer. Based upon the previous force and moment analysis and contact with Kurt Banaszynski of Engineering Laboratory Design Inc., the manufacturer of the dynamometer and the wind tunnel, it was confirmed that using either material would not damage the transducers within the dynamometer (personal communication, 2005). This analysis can be found in Appendix A.

Next, the cost of using acetal versus aluminum was assessed. After obtaining quotes from Yarde Metals, Tri Star Plastics and Plastics Unlimited, the total estimated material cost for eight models using aluminum was approximately $\$ 1,300$ and for acetal, $\$ 1,600$ (Yarde Metals personal communication, 2005; TriStar Plastics personal communication, 2005; Plastics

Unlimited personal communication, 2005) Refer to Appendix E for stock sizes and pricing quotes.

With a limited amount of manufacturing time and the complex nature of the CNC machining processes, selecting a material with the shortest cut time was a major consideration. Upon consultation with Steve Derosier and Matt Munyon at the Washburn Shops, it was determined that the cut time for aluminum would be at a minimum four times that of the cut time for acetal.

Although aluminum is stronger and costs less than acetal, it is approximately twice as heavy as acetal. With a much faster cut time and suitable strength and rigidity properties, acetal was chosen as the material for the solid parachute models.

### 3.3 Construction Economic Considerations

The cost of construction became the most significant factor driving construction method choice. If the budget for this project had been able to support stereolithography manufacturing, models could have been made in approximately three weeks as opposed to six months. In addition, models could have been made without the boss required for holding the model during machining.

Cost also played a significant role in material selection. However, the other factors mentioned above superseded cost when it came to choosing a stock material. From the construction method decision analysis, it is seen that economic and time constraints make significant contributions when determining how to meet design requirements and project objectives.

## 4. Manufacturing

### 4.1 Design of Manufacturing Process

With a manufacturing method chosen, it was necessary to develop a machining process to make the rigid models. Preliminary discussions with machinists and manufacturing experts at WPI showed that their experience was limited making parts with the complex rounded geometries and thin walls present in the canopy models.

The part most similar to these rigid canopy models which was manufactured at the Washburn Shops was a smaller simplified parachute model that lacked the feature of the canopy curving back into itself. That machining was done by clamping a single block of aluminum to the worktable of a vertical CNC mill and surfacing the entire inside of the canopy as if it were upside down. After that process, the piece was flipped right side up and fitted onto a spherical mount for support and vibration suppression during the surfacing operation of the outer side of the canopy.

Some of the rigid models designed for this research have a skirt diameter that is less than the maximum parachute diameter. This feature is necessary to accurately represent the flexible canopy shape, however, it creates difficulties for interior machining operations. These difficulties were not encountered by the Washburn Shop when they made the smaller parachute model. In addition, the method used by the shop to make the small parachute model required making complicated custom fixturing. In order to make the rigid canopy models, the method the shop used either needed to be adapted or rethought completely.

In an effort to reduce the amount of custom fixturing needed to machine the models, it was determined that a method had to be found for removing as much stock material as possible without the use of custom fixturing. The round geometry of the canopies made using a CNC lathe the ideal choice for removing the bulk of the stock material. The CNC lathe would be able
to remove a majority of the stock material on the outside of the canopy as well as bore out a majority of the stock material inside the models. Although lathe operations were adequate for making models 1 and 2 , the gore geometry could not be created with the lathe.

This problem was solved by combining both lathe and milling operations. It was also determined that the addition of a fourth and fifth axis for the CNC mill would allow the machining of the canopies where the skirt curves back into the middle of the model.

The milling processes required for creating the exterior and interior gore geometry involved multiple part orientations. One orientation needed was to align the canopy model so that its central axis would be horizontal and the mill would be able to surface the top outer gores. The fifth axis would then index the piece about its central axis to enable further surfacing of the outer gores. The surfacing operation for the inside of the canopy models would entail the rotation of the fourth axis to stand the canopy upside down with its leading edge pointed upward. The surfacing operation would then proceed. If the model contained the inner curve back feature, the fourth axis would be rotated so the piece is at a $45^{\circ}$ angle. This would allow room for the tools to move about within the canopy. The combination of the lathe and mill processes as well as the addition of the fifth axis made the manufacturing of these models a possibility.

### 4.2 Manufacturing Procedure

The HAAS CNC machines in the Washburn Shops are controlled using G-Code.
GibbsCAM software was used to generate the G-Code needed to machine the parachute models. For models 1 and 2, only the lathe processes were need due to their simple shape. Only two GibbsCAM G-Code programs were needed to make models 1 and 2. For models 3 through 8,
the lathe and mill processes required using four different GibbsCAM G-Code programs to make each model. Each time a new G-Code program was used, machine fixturing had to be adjusted, the G-Code file had to be loaded, and work piece and tool offsets had to be set.


Figure 15 - Example of G-Code on VF-4
The first step in making a model was cutting the needed material from the lengths of acetal stock. This was done on the horizontal band saw. The next step was to turn the stock to the rough outside shape of the parachute model on the HAAS TL-1 tool room lathe. G-Code for this process was produced in GibbsCAM by telling the lathe to follow the outer curve to a tolerance of .001 inches. The turning of the outer curve took approximately a half hour for model 1,1 hour for model 2 and 1.5-3 hours for models 3-8. These fast cut times were achieved because the stock was supported at both ends. This made the part stiff and eliminated noticeable vibration, which produced a clean cut. Once outside turning was complete, the turned piece was cut away from the remaining stock.

Next, the boss on the turned model was clamped into the manual lathe and a 2 inch diameter hole was drilled into the model. The depth of this hole was determined based on the distance the boring bar in the second machining operation was required to go inside the model.

The second process again used the TL-1, but this time, a boring bar was used to remove material from the inside of the parachute model. G-Code for this process was produced in GibbsCAM by telling the lathe to follow the inner curve to a tolerance of .001 inch. Because the entire model was now only being held at one end by the boss, structural rigidity was reduced and severe vibrations began to occur in all models. As a result, this operation required constant monitoring. Vibrations were minimized by careful manipulation of feed and spindle rates. These adjustments induced a higher chip load, which helped stabilize the work piece. Although these adjustments were necessary, they also increased the amount of time it took to bore out the inside of each model. Cut time for the inside of models 2-8 varied from 4-9 hours.

After these two lathe processes, models $1 \& 2$ were complete. Models 3-8 required milling work to create gore details on the outside, inside and skirt edge surfaces.


Figure 16 - Model 5 in TL-1 Interior Boring Operation


Figure 17 - Model 5 in TL-1


Figure 18-Close-up of Interior Boring

Milling operations were done on the HAAS VF-4, 5-axis vertical milling station. For models 3-8, the third G-Code process consisted of creating the outside gores and shaping the skirt edge surfaces. Due to unique model geometry, turned models were held in the TC-5 fifth axis using a one inch diameter compressed air collet clamp system. The fourth and fifth axes were used to index and rotate the model for the outside and inside surfacing operation.


Figure 19 - Model 5 in VF-4 Work Holding Setup
To begin the outside surfacing operations, the turned piece was placed in the collet clamp and probed to set tool and work piece offsets. The G-Code for this process was written so that three gores would be surfaced at a time, after which the machine would cut the skirt shape on these three gores. Cutting more than three gores at a time made the ball endmill cut into the finished portion of the model. The $5^{\text {th }}$ axis was then manually indexed $45^{\circ}$, and three more gores would be surfaced. This process was repeated seven more times until all twenty-four gores were completed.


Figure 20 - Model 8 Outer Surfacing Operation and Indexing

The fourth process involved surfacing the inside of the parachute model. For models that did not have interior geometry that curved back on itself interior milling operations could be conducted with the piece rotated as shown in Figure 22.


Figure 21 - Model 4 Example of Interior Curve
As seen in Figure 22, the piece was rotated up to a vertical position, with the open cup pointed straight up. Pieces which had gores that curved back into the piece needed to be set at a $45^{\circ}$ angle to cut the interior curve from the skirt to the apex. Figure 21 shows an example of a model with gores curving back toward the centerline of the canopy. However, probing these canopy models tilted at a $45^{\circ}$ angle to set CNC machine coordinates and offsets could not be done precisely. Therefore, the CNC machine was programmed to cut the inside of the canopy with the part oriented in the vertical position. As a result, the inside milling operation only surfaced a thin lip at the edges to keep the pieces from becoming too thin and losing their stiffness and strength and also to prevent the finished portions of the model from being cut away due to lack of tool clearance. If too thin and subjected to too strong of a cutting force, the piece would become more susceptible to breaking apart. Therefore, this operation also needed constant monitoring to ensure that the change in model structure from removing material did not
induce vibrations that could break the model. Including setup time, the third and fourth operations took approximately 15 hours each.


Figure 22 - Model 5 in Vertical Position

The models were then hand finished using a Dremel rotary tool and 500-grit sandpaper to remove slight imperfections in the surface finish. The boss on each model was also drilled and tapped to accept the sting allowing for attachment to the dynamometer in the wind tunnel.


Figure 23 - Hand Sanding


Figure 24 - Final Models


Figure 25 - All Models (Excluding 6) Side by Side
Due to time constraints and machine workspace limitations, the manufacturing of model six was not completed. As a result, the milling processes were not finished on this model.

### 4.3 Economic Considerations

The price of the manufacturing these eight models at the Washburn shops was considerably less than the price of using the rapid prototyping method at NSC. This is because WPI covers the cost of student project teams using the Washburn Shop CNC machines. However, if these models were machined at an outside machine shop, completion of the models would take approximately two months to complete (Munyon personal communication, 2006). The average hourly rate for CNC machining in the United States is $\$ 70.00$ per hour (Derosier personal communication, 2006). It was estimated that for models 3-8, a highly qualified machinist could make one model in 40 hours. For models 1 and 2, it would take an experienced machinist about 15 hours to make a model. This includes G-code generation using a software tool such as GibbsCAM, machine setup time, cut time, and finishing processes. This would be $\$ 2,800$ each for models 3-8 and \$1,050 each for models 1 and 2. This is a total of $\$ 18,900$ for all models. This does not include the cost of materials. Although the machining costs associated with using the WPI Washburn Shops are not directly charged to this project, the total cost of

CNC machining at an outside machine shop, including materials, is at least four times the cost of using the Natick Soldier Center stereolithography facilities.

## 5. Testing Procedure

### 5.1 Facilities and Equipment Used

The wind tunnel used for this research was a $2 \times 2 \times 8 \mathrm{ft}$ test section, re-circulating wind tunnel manufactured by Engineering Laboratory Design Inc. Data acquisition was performed with an ELD Inc. dynamometer and Digital Readout.


Figure 26 - ELD Digital Readout
LabVIEW 7.1 software in conjunction with a National Instruments DAQ-PAD USB 6020-E were used to read dynamometer voltages from the analog output on the ELD Digital Readout Box. These voltages vary linearly with the force on the dynamometer. This linearity was verified during dynamometer calibration. A LabVIEW VI was created to capture the
voltage readings from the dynamometer analog output readout box. Shown below are the front panel and the block diagram of the LabVIEW VI.


Figure 27 - LabVIEW VI Front Panel.


Figure 28: LabVIEW VI Block Diagram.
It can be seen from the VI block diagram that data is averaged, displayed on a waveform chart, and exported to a spreadsheet along with the real time of data acquisition. The control in the bottom right of the while loop sets the number of times the VI will iterate. This allows the user to control the number of data points acquired. Due to the linearity of the voltages to corresponding weight values, the averaged data was multiplied by a slope and summed with an intercept. This was done to scale the voltage values to a weight value. The slope and intercept values input into LabVIEW were obtained using a dynamometer calibration procedure described in the following section. The calibration procedure was used to find the relationship between known weights and the dynamometer output voltages. The slope and intercept could be input into the VI front panel. This was all designed to save time later when analyzing and reducing the
data points. It also allowed the user to directly compare force measurements between the VI and the ELD digital readout box in real time.

Voltage data coming into the VI was averaged using a For loop and the mean function. This combination was used to remove signal noise from the system. The For loop was programmed to execute four times, meaning that four data points were collected. Those four data points were then sent to the mean function, where they were averaged. This process continued until the condition of the while loop became false i.e. after the programmed number of iterations. Due to the fact that all of the drag force measurements were averaged later on using Excel, this method of averaging does not corrupt the data. The data points would have been averaged eventually; however, this allowed the project team to read more stable drag values for direct comparison with the ELD digital readout box. The block diagram for the mean function can be seen below.


Figure 29: Block Diagram of the Mean Function

Inputs were placed within the VI to collect a variety of information. An "append to file" function was used to export the dynamometer forces calculated by the VI from the voltages to an Excel file. The temperature of the wind tunnel test section, pitot static pressure, and the time of each sample were also exported to this excel sheet. One data spreadsheet was generated for each
model at each test velocity. With each spreadsheet generated, various calculations were made to process the data. The drag force measurements were averaged and converted to pounds of force. It is important to note that this calculation was also completed for the dynamometer and sting structure with no model attached. The average drag force for the dynamometer and sting attachment was subtracted from all of the other averages to determine the drag on the models alone. The standard deviation of each set of drag force measurements was calculated, and used to calculate error. A sample of these processed data spreadsheets can be found in Appendix D. The error calculations are summarized in the results section of this document.

### 5.2 Dynamometer Calibration

Before wind tunnel testing could begin, it was necessary to calibrate the wind tunnel dynamometer and ELD data readout system and LabVIEW VI. To do this, the dynamometer was first taken out of the wind tunnel test section, and clamped to a workbench. Preliminary investigation showed that the dynamometer was set to read drag forces in kilograms. The potentiometers on the digital readout were zeroed and a known weight of 0.5 kg was hung from the end of the dynamometer. The span of the potentiometers was then adjusted to display 0.5 kg on the dynamometer box digital readout. This process was repeated using weight values of 0.1 $\mathrm{kg}, 1 \mathrm{~kg}$, and 1.5 kg . This procedure ensured that the ELD readout box and corresponding output voltages were calibrated properly.

Next, the slope and intercept values needed to calibrate the LabVIEW VI were determined. For each mass ( $.1 \mathrm{~kg}, .5 \mathrm{~kg}, 1 \mathrm{~kg}, 1.5 \mathrm{~kg}$ ), the VI was run and set to record 100 voltage data points. The voltage data points and the known masses were exported to a
spreadsheet. Once these tests were complete, the masses were plotted as a function of voltage. A linear trend-line was added to determine the slope and intercept of the data. A least squares value was also calculated to determine the accuracy of the trend-line. The calibration curve used during wind tunnel testing on 2/23/06 is shown in Figure 30.


Figure 30-Calibration curve used to convert voltages from the dynamometer readout to force for rigid model testing on $2 / 23 / 06$. The linear curve fit trend line for the calibration data has an $R^{2}$ value of .99.

Using a least squares test, it was shown that the linear trend was a very good fit for the data. It is also seen that for a constant force on the dynamometer, the variation in voltages recorded by the virtual instrument was on the order of only a few millivolts. With a slope and intercept value input into the VI front panel, the readouts from the VI and ELD readout box were again checked using all four weights to verify that they were both reading the same force values.

Once the calibration was determined to be satisfactory, the dynamometer was mounted in the wind tunnel test section.

### 5.3 Pressure Calibration

The freestream stagnation pressure within the test section was required to verify the freestream velocity in the wind tunnel. To determine the pressure, a pitot tube was placed in the test section. The pitot tube measured the difference between the stagnation pressure in the freestream and the static pressure inside the test section. This pressure measurement was recorded using the pressure transducer in the ELD readout box.

The pressure value on the readout box was set by ELD to read pressure in inches of water. It was calibrated by first placing the pitot probe a quarter of the way down into the test section from the top of the tunnel. The pressure readout was then zeroed. With only the dynamometer in the test section, the tunnel fan was set to run at 12.8 Hz . This fan speed corresponds to a freestream velocity of $10 \mathrm{~m} / \mathrm{s}$ according to the wind tunnel manual (ELD, 1998). Using Equation (8), the pressure difference was calculated to be 0.241 in $-\mathrm{H}_{2} 0$. The potentiometer span on the digital readout was then adjusted to read this value.

$$
\begin{equation*}
\Delta h=\frac{.5 \rho_{\text {air }} U_{\infty}^{2}}{\rho_{H_{2} O} g} \tag{8}
\end{equation*}
$$

This calculation was repeated at tunnel speeds of $18 \mathrm{~m} / \mathrm{s}$ and $25 \mathrm{~m} / \mathrm{s}$, resulting in values of 0.782 and 1.505 in $-\mathrm{H}_{2} 0$. The wind tunnel fan was then set to the frequency corresponding to each of these velocities to be sure that the pressure readouts matched their respective calculated values. The values did in fact match, resulting in a calibrated pressure measurement system.

As mentioned earlier, the pitot tube was essential for verifying the freestream velocity in the wind tunnel test section. This is due to the fact that the addition of a parachute model inside the wind tunnel test section constricts the flow within the wind tunnel. This creates a pressure difference, which in turn results in varying freestream velocities around the canopy depending on the size of the model for a set wind tunnel fan frequency. Rather than using the wind tunnel fan frequency to set the freestream velocity, the pressure reading was used. In doing so, freestream velocities could be matched to the three standard test velocities regardless of the size of the model.

The three velocities were based on earlier calculations of the expected drag forces that each model would experience at different drag coefficients and tunnel velocities within the range of forces the dynamometer could accurately measure. They were also chosen to make the Reynolds number range as broad as possible. The speeds selected were $10 \mathrm{~m} / \mathrm{s}, 18 \mathrm{~m} / \mathrm{s}$, and 25 $\mathrm{m} / \mathrm{s}$. The velocity of $10 \mathrm{~m} / \mathrm{s}$ was chosen because it was the slowest speed where measurable drag forces could be recorded. The velocity of $18 \mathrm{~m} / \mathrm{s}$ was selected because it was the fastest velocity for which it was expected that all models could be accurately measured by the dynamometer for a range of possible drag coefficients with magnitudes from 0.5 to 3 . Lastly, the velocity of 25 $\mathrm{m} / \mathrm{s}$ was selected because it was anticipated that drag coefficients of two or more were unrealistic, allowing for greater freestream velocities than $18 \mathrm{~m} / \mathrm{s}$. The tables used for determining these velocity ranges can be seen in Appendix C.

### 5.4 Drag Measurements

The wind tunnel and data acquisition system was set up as shown in the schematic shown below.


Figure 31 - Wind Tunnel Schematic
The data acquisition system was set up as shown in the picture below.


Figure 32 - Data Acquisition System

A model was attached to the sting through its boss and then placed on the dynamometer. This configuration is seen in the photo below.


Figure 33 - Testing Configuration (model 2 shown)

The wind tunnel fan was then adjusted until the pressure required for the desired freestream velocity was achieved. Upon reaching this freestream velocity, one minute was allowed to pass before recording data to make sure that any effects from ramping the tunnel speed had dissipated. Two hundred data points were then recorded at a sample rate of approximately 2 Hz . The pressure was then adjusted to reach the next freestream velocity. Again, approximately one minute after stabilization, the data acquisition procedure was repeated. Finally, this same procedure was used for the last freestream velocity. This completed the testing procedure for one model. Each of the remaining models followed the same procedure.

## 6. Results

The seven rigid canopy models were tested in one day following the testing procedure outlined above. Testing all models in one day had the benefits of using a single calibration, relatively constant atmospheric conditions, and minimized the error introduced by varying setups from day to day. These experiments were completed on February 23, 2006. This particular day was cloudy with intermittent showers. The atmospheric pressure was obtained from the Worcester Regional Airport at hourly intervals. Throughout this experiment this measurement remained constant at $404.454 \mathrm{in}-\mathrm{H}_{2} \mathrm{O}$.

It was noted that each model behaved differently under the same testing conditions. Due to the blunt shape of these models, vibrations were a serious concern. To determine the upper bound of freestream velocity, each model was placed in the wind tunnel to observe stability. The most unstable models were found to be models 5,7 and 8 . After preliminary testing of these models at varying tunnel speeds, it was determined that a freestream velocity of $25 \mathrm{~m} / \mathrm{s}$ was indeed a satisfactory upper bound velocity that the models could be safely tested without inducing serious vibrations.

At $10 \mathrm{~m} / \mathrm{s}$, vibrations were not visually noticeable for any model. Force variations, however, could be seen on the dynamometer output, and within the virtual instrument. This may be due to the fact that, at such a slow speed, vortex shedding is not present. It may also mean that the vortex shedding that is present is not strong enough to induce visually noticeable vibrations.

At a freestream velocity of $18 \mathrm{~m} / \mathrm{s}$, vibrations were more noticeable for models 3-8. The model movement tended to be horizontal rather than vertical or in a circular motion. Estimated deflections were approximately $1 / 4$ inch maximum at this speed. This motion is most likely due
to the interaction of vortex shedding cycles with the canopy model. Models 1 and 2 remained relatively stable, most likely due to the fact that they were the least blunt of all the models. It was noticed that the onset of vibration was a gradual process. If the vibrations were stopped by hand, they could be seen slowly building up to a maximum once again. Depending on model shape, the time to reach maximum vibration was approximately 2-10 seconds.

Similarly, for a freestream velocity of $25 \mathrm{~m} / \mathrm{s}$, oscillatory vibrations were seen in model 2 and became increasingly noticeable for models 3-8. At this velocity, the model movement tended to be in both the horizontal and vertical directions. Estimated maximum deflections from the model centerlines were approximately $1 / 2$ inch for models $4,5,7$ and 8 and $1 / 4$ inch for models 2 and 3. Model one remained relatively steady. Again, the onset of these oscillatory vibrations was gradual; however, the time it took for these models to reach maximum vibrations was reduced to about 1-5 seconds depending on model.

The oscillations and vortex shedding caused by flow separation created variation in the drag measurements. This could be seen both on the dynamometer output and the virtual instrument as recorded values did not stay constant. Analysis of the data collected quantified this variation in the form of percent error.

Table 1 summarizes the testing conditions, and results from processed test data.

Table 1 - Test Conditions and Data Summary

| Model | Freestream Velocity (m/s) | Tunnel Frequency (Hz) | Temperature ( ${ }^{\circ} \mathrm{F}$ ) | Atmospheric Pressure $\left.\mathrm{H}_{2} \mathrm{O}\right)$ | $\begin{gathered} \hline \text { Pressure } \\ \left(\mathrm{P}_{\mathrm{o}}-\mathrm{P}_{\text {staticic }}\right) \\ \left(\text { (in } \mathrm{H}_{2} \mathrm{O}\right) \\ \hline \end{gathered}$ | Average <br> Drag (lbs) | Standard Deviation (lbs) | Total Percent Error of Drag Calculations |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 10 | 14.6 | 79 | 404.454 | 0.241 | 0.0295 | 0.1327 | 137.38 |
|  | 18 | 25.1 | 80 | 404.454 | 0.782 | 0.1222 | 0.1376 | 85.35 |
|  | 25 | 34.2 | 83 | 404.454 | 1.505 | 0.2691 | 0.1317 | 24.45 |
| 2 | 10 | 14 | 73 | 404.454 | 0.241 | 0.2307 | 0.1420 | 10.22 |
|  | 18 | 24.5 | 75 | 404.454 | 0.782 | 0.8106 | 0.1487 | 3.22 |
|  | 25 | 33.5 | 77 | 404.454 | 1.505 | 1.6007 | 0.1601 | 1.74 |
| 3 | 10 | 14.3 | 79 | 404.454 | 0.241 | 0.3832 | 0.1233 | 6.29 |
|  | 18 | 24.6 | 79 | 404.454 | 0.782 | 1.2292 | 0.1579 | 2.35 |
|  | 25 | 33.5 | 82 | 404.454 | 1.505 | 2.4182 | 0.2925 | 1.97 |
| 4 | 10 | 13.9 | 76 | 404.454 | 0.241 | 0.5379 | 0.1229 | 5.31 |
|  | 18 | 23.9 | 77 | 404.454 | 0.782 | 1.7311 | 0.2222 | 2.31 |
|  | 25 | 32.8 | 81 | 404.454 | 1.505 | 3.5613 | 1.9355 | 8.15 |
| 5 | 10 | 13.5 | 76 | 404.454 | 0.241 | 0.8360 | 0.1209 | 4.63 |
|  | 18 | 23.2 | 76 | 404.454 | 0.782 | 2.7390 | 0.1876 | 1.63 |
|  | 25 | 31.8 | 78 | 404.454 | 1.505 | 5.2392 | 0.6097 | 1.83 |
| 7 | 10 | 12.9 | 79 | 404.454 | 0.241 | 0.9735 | 0.1162 | 4.47 |
|  | 18 | 22.9 | 80 | 404.454 | 0.782 | 3.4272 | 0.2244 | 1.60 |
|  | 25 | 30.1 | 81 | 404.454 | 1.505 | 5.9788 | 0.1527 | 0.76 |
| 8 | 10 | 13.7 | 80 | 404.454 | 0.241 | 0.8372 | 0.1308 | 4.71 |
|  | 18 | 23.3 | 80 | 404.454 | 0.782 | 2.5853 | 0.1697 | 1.60 |
|  | 25 | 32 | 82 | 404.454 | 1.505 | 5.0371 | 0.2068 | 0.90 |

The results in Table 1 show that the percent error is larger for the smaller models. It is believed that this is due to the relatively low drag force measurements for these models. There was a small amount of "noise" within the system when no measurements were being taken. This could be seen on the digital readout when the tunnel was not running. This noise was estimated to be approximately $+/-0.02 \mathrm{~kg}$. Average drag measurements for model one ranged from 0.013 to 0.122 kg . Although this noise was averaged out within the virtual instrument, it still accounted for a great percent of the drag readout for the smaller models. This explains the large percent error for the small models.

The same reasoning explains why the percent error decreased as the freestream velocity increased. Slower freestream velocities again resulted in lower drag forces, which were more influenced by system noise. Faster freestream velocities resulted in higher drag forces, which were more able to overwhelm the system noise, resulting in less error.

Plots of the measured drag force on each model at $10 \mathrm{~m} / \mathrm{s}, 18 \mathrm{~m} / \mathrm{s}$ and $25 \mathrm{~m} / \mathrm{s}$ are shown in Figure 34, Figure 35, and Figure 36. Table 2 summarizes drag forces obtained at each
freestream test velocity. These results show that drag force is directly proportional to the model area normal to the freestream flow in the wind tunnel test section.

Table 2 - Measured Drag Forces

| Model <br> Number | Frontal Area <br> $\left(\mathbf{m}^{\mathbf{2}}\right)$ | Model Drag <br> $\mathbf{( \mathbf { I b } ) \mathbf { 1 0 } \mathbf { ~ m / s }}$ | Model Drag <br> $\mathbf{( \mathbf { l b } ) \mathbf { 1 8 } \mathbf { m } / \mathbf { s }}$ | Model Drag <br> $\mathbf{( \mathbf { I b } ) \mathbf { 2 5 } \mathbf { ~ m / s }}$ |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 0.00083 | 0.0137 | 0.0228 | 0.0762 |
| 2 | 0.01021 | 0.2148 | 0.7112 | 1.4078 |
| 3 | 0.01523 | 0.3674 | 1.1297 | 2.2254 |
| 4 | 0.02577 | 0.5221 | 1.6317 | 3.3685 |
| 5 | 0.03658 | 0.8202 | 2.6396 | 5.0464 |
| 7 | 0.04264 | 0.9576 | 3.3277 | 5.7860 |
| 9 | 0.03390 | 0.8213 | 2.4858 | 4.8443 |



Figure 34 - Drag Force of Various Models at 10 m/s


Figure 35 - Drag Force of Various Models at 18 m/s


Figure 36 - Drag Force of Various Models at 25 m/s

The average drag values for each model at each freestream velocity were found by averaging the 200 drag values recorded by the virtual instrument. The standard deviation was then calculated for each of these averages.

$$
\begin{equation*}
S t d D e v=\sqrt{\frac{\sum(x-\bar{x})^{2}}{(n-1)}} \tag{9}
\end{equation*}
$$

where $\bar{x}$ is the arithmetic mean, x is a sample point and n is the number of sample points. Uncertainty was found using the least squares uncertainty method.

$$
\begin{equation*}
\omega=\left[\sum\left(\frac{\partial R}{\partial x} \omega_{i}\right)^{2}\right]^{\frac{1}{2}} \tag{10}
\end{equation*}
$$

where $\omega$ is the total uncertainty, $\frac{\partial R}{\partial x}$ is the change in the results with respect to the measured variables and $\omega_{\mathrm{i}}$ is the uncertainty for each result. The total error was based upon the uncertainties in pressure and drag measurements. The uncertainty for pressure was estimated to be +/- .01 inches of $\mathrm{H}_{2} \mathrm{O}$ due to oscillations on the dynamometer digital readout. The uncertainty for drag was calculated using the standard deviation for the drag measurements, using the following formula.

$$
\begin{equation*}
\omega=\frac{2 S t d D e v}{\sqrt{n}} \tag{11}
\end{equation*}
$$

To calculate percent error, the uncertainty is divided by the results.

$$
\begin{equation*}
\text { PercentError }=\frac{\omega}{R} \times 100 \tag{12}
\end{equation*}
$$

Uncorrected drag coefficients were then calculated based upon the average drag, freestream dynamic pressure and maximum projected frontal area using Equation (5) and Equation (6). Drag coefficients using this average drag data were calculated for each model at each tunnel test speed. These results are summarized in the table below.

Table 3 - Uncorrected Drag Coefficients and Reynolds Number for Rigid Models

| Model | Freestream Velocity (m/s) | Uncorrected C | Reynolds Number |
| :---: | :---: | :---: | :---: |
| 1 | 10 | 1.219 | 21637 |
|  | 18 | 0.628 | 38946 |
|  | 25 | 1.088 | 54092 |
| 2 | 10 | 1.553 | 76006 |
|  | 18 | 1.586 | 136811 |
|  | 25 | 1.628 | 190015 |
| 3 | 10 | 1.779 | 92835 |
|  | 18 | 1.689 | 167102 |
|  | 25 | 1.725 | 232086 |
| 4 | 10 | 1.495 | 120759 |
|  | 18 | 1.442 | 217366 |
|  | 25 | 1.543 | 301897 |
| 5 | 10 | 1.654 | 143875 |
|  | 18 | 1.643 | 258975 |
|  | 25 | 1.628 | 359688 |
| 7 | 10 | 1.657 | 149053 |
|  | 18 | 1.777 | 268296 |
|  | 25 | 1.602 | 372633 |
| 8 | 10 | 1.787 | 155341 |
|  | 18 | 1.669 | 279613 |
|  | 25 | 1.686 | 388352 |

These uncorrected drag coefficients do not account for wall interference effects within the wind tunnel. As a result, the values of the drag coefficients were larger than they should be. Recall that the pressure increases around the model due to the flow constriction. This effect must be accounted for within the drag coefficient by correcting the dynamic pressure.

The method used for correcting tunnel data for wall interference effects was the Maskell Correction method for bluff bodies adapted by Macha and Buffington for circular parachutes under test in a closed-loop wind tunnel. Using test-section cross sectional area C , the model drag area $\mathrm{C}_{\mathrm{D}} \mathrm{S}_{\mathrm{u}}$ and the freestream dynamic pressure $\mathrm{q}_{\mathrm{u}}$, the adapted Maskell equation for the dynamic pressure correction at the model is

$$
\begin{equation*}
\frac{q}{q_{u}}=1+K_{M}+\frac{C_{D} S_{u}}{C} \tag{7}
\end{equation*}
$$

The results of Macha and Buffington's experiments provide blockage correction factors based on Maskell correction methods for three-dimensional, non-lifting bluff bodies. According to these experiments, Macha and Buffington determined that for a single round parachute canopy, a Maskell bluff-body blockage factor of 1.85 accurately accounts for the effective increase in dynamic pressure due to the presence of a round parachute canopy in the tunnel. It was also found that this correction factor is independent of canopy porosity and can be applied to circular canopies in general (Macha \& Buffington, 1989). See Appendix B for correction calculations.

This correction method was applied to the calculated drag coefficients. Since the correction method accounts for the increase in velocity around the model, the corrected dynamic pressure will increase. As a result, the values of the drag coefficients will decrease. The corrected drag coefficients and corresponding Reynolds numbers are shown in the following table.

Table 4 - Corrected Drag Coefficients and Reynolds Numbers for Rigid Models

| Model | Freestream Velocity (m/s) | Corrected C | Reynolds Number |
| :---: | :---: | :---: | :---: |
| 1 | 10 | 1.213 | 21691 |
|  | 18 | 0.626 | 38996 |
|  | 25 | 1.083 | 54213 |
| 2 | 10 | 1.439 | 78947 |
|  | 18 | 1.468 | 142218 |
|  | 25 | 1.504 | 197718 |
| 3 | 10 | 1.568 | 98899 |
|  | 18 | 1.497 | 18567 |
|  | 25 | 1.525 | 246794 |
| 4 | 10 | 1.254 | 131829 |
|  | 18 | 1.217 | 36228 |
|  | 25 | 1.288 | 330427 |
| 5 | 10 | 1.271 | 164119 |
|  | 18 | 1.265 | 295185 |
|  | 25 | 1.256 | 409558 |
| 7 | 10 | 1.226 | 173292 |
|  | 18 | 1.290 | 314855 |
|  | 25 | 1.195 | 431350 |
| 8 | 10 | 1.373 | 177226 |
|  | 18 | 1.302 | 316564 |
|  | 25 | 1.313 | 440167 |

Table 5 - Corrected Drag Coefficients and Reynolds Numbers for the Flexible Canopy

> Flexible Canopy - Water Tunnel
( $C_{D}$ based on frontal area and corrected for wall interference)

| Corresponding <br> Solid Model | $\mathbf{D}(\mathbf{c m})$ | $\mathbf{F}(\mathbf{N})$ | $\mathbf{C}_{\mathbf{D}}$ | $\mathbf{F}(\mathbf{l b s})$ | $\mathbf{R E}$ \# |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 3.62 | 0.0193 | 0.9762 | 0.0043 | 6886 |
| 2 | 12.51 | 0.0966 | 0.4091 | 0.0217 | 24191 |
| 3 | 15.44 | 1.2243 | 3.4039 | 0.2752 | 29547 |
| 4 | 20.05 | 2.8345 | 4.6732 | 0.6372 | 38435 |
| 5 | 23.46 | 5.2926 | 6.3738 | 1.1898 | 45792 |
| 6 | 24.36 | 2.8909 | 3.2289 | 0.6499 | 47440 |
| 7 | 25.36 | 1.1512 | 1.1864 | 0.2588 | 49441 |
| 8 | 22.61 | 1.5585 | 2.0206 | 0.3504 | 44085 |

With drag coefficients corrected for wall interference effects, it was necessary to determine how these drag coefficients varied over the range of tunnel test velocities. For each model, the three experimentally determined drag coefficients were plotted against corresponding Reynolds number. Error bars were included for each point based on the percent error from drag and pressure measurements. The average of the three experimentally determined drag coefficients for each model was then calculated. Plots of this data are shown below.


Figure $37-C_{D}$ vs. Re: Model 1


Figure 38- $\mathrm{C}_{\mathrm{D}}$ vs. Re: Model 2


Figure 39- C ${ }_{\text {D }}$ vs. Re: Model 3


Figure $40-C_{D}$ vs. Re: Model 4


Figure 41- $\mathrm{C}_{\mathrm{D}}$ vs. Re: Model 5


Figure 42 - $\mathbf{C}_{\text {D }}$ vs. Re: Model 7


Figure 43- C $_{\text {D }}$ vs. Re: Model 8

It is clear that the calculated drag coefficient for each model remains relatively constant over the range of Reynolds numbers achieved during testing. The averaged drag coefficient falls
within the range of error or very near to the range of error for all models. Transitions in flow characteristics causing the drag coefficient to drastically change are not noticed in this Reynolds number range.

After determining that the averaged $C_{D}$ values for each model appropriately represented the $C_{D}$ values for the range of Reynolds numbers seen during this testing, these values were plotted against $C_{D}$ values from the flexible canopy model tested in a water tunnel. The Reynolds number range under which the 30.5 cm constructed diameter flexible canopy water tunnel test was conducted are not the same as the Reynolds numbers achieved with wind tunnel testing for this research.

Reynolds numbers for the flexible canopy tests were on the order of $10^{4}$ whereas the Reynolds numbers for the solid model tests were on the order of $10^{5}$. Although this difference is only one order of magnitude, a direct comparison of these separate results can only be made if it known that the slope of the drag coefficient vs. Reynolds number plot is relatively constant in and between these two Reynolds number ranges.

A direct comparison of flexible canopy and solid model test data can be done with confidence for two reasons. First, a comparison of drag coefficient data for shapes similar to the flexible canopy and solid models showed that for these Reynolds number ranges, the slope of the drag coefficient with Reynolds number curve is very small and remains constant (Hoerner, 1958). Therefore, it is not expected that an abrupt change in slope of the drag coefficient will occur in the experimental data between the flexible and solid canopy model data. Secondly, it is seen in Figure 44 that the drag coefficients for models 1 and 8 are nearly equal to the drag coefficients calculated for the flexible canopy. Therefore, since both these models represent steady-state conditions, this result again helps validate this comparison.

The results suggest that the opening force on the flexible canopy is not directly related to the shape of the canopy. Therefore, other factors, such as the dynamic time rate of change of the canopy shape, must contribute to the drag forces seen during inflation. In the charts that follow, drag coefficients for the 30.5 cm flexible canopy model are plotted against normalized time. Also plotted are the drag coefficients for the solid models that correspond to the shape of the flexible canopy at times corresponding to specific points of interest on the force vs. time graph for the flexible canopy. The Reynolds numbers are scaled to the model's projected area normal to the freestream flow.


Figure 44-Calculated drag coefficients ( $\mathrm{C}_{\mathrm{D}}$ ) of rigid models (red) and the flexible canopy model (blue). The time average drag coefficient (black) and corresponding drag coefficient range (black range bar) for the flexible canopy model under steady-state conditions is also shown


Figure 45 - Calculated drag coefficients of rigid models (red) and the flexible canopy model (blue) against normalized time. The average drag coefficient (black) and corresponding drag coefficient range (black range bar) for the flexible canopy model under steady-state conditions is also shown

It can be seen in the figure above that the $C_{D}$ values for models $1,2,7$ and 8 are relatively close to the $\mathrm{C}_{\mathrm{D}}$ values of the flexible canopy models. The other models, however, do not compare with the flexible canopy. It is important to note that the flexible canopy $C_{D}$ for model number 8 can fall within the error bar depicted in the graph. Due to the breathing cycle of the canopy in steady state, the projected area is constantly changing. This area change directly affects the $C_{D}$ value. Therefore, based on the time chosen during steady state, the $C_{D}$ will vary slightly. It can be seen that the results from this experiment fall within the error bar for the flexible canopy as well.

## 7. Conclusions

Based upon the data presented in this report, several conclusions can be drawn. First, it was shown that the drag coefficients for each model remained relatively constant over the range of Reynolds numbers tested.

Table 6- Reynolds Number Ranges for Rigid Parachute Models

| Experimental Reynolds Number Ranges For Each Model |  |  |  |  |  |  |  |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model Number | 1 | 2 | 3 | 4 | 5 | 7 | 8 |
| Reynolds <br> Number Range <br> (Re $\times 10^{3}$ ) | $21-54$ | $78-197$ | $98-246$ | $131-330$ | $164-409$ | $173-431$ | $177-440$ |

Secondly, it was determined that the drag coefficients calculated in this experiment for models 2-7 do not match the drag coefficients found for a flexible model parachute canopy in the water tunnel even though the geometries are nearly identical. The fact that the drag coefficients for each model at each freestream velocity were found to be constant within a narrow margin of error shows very good experimental technique.

The variation in drag coefficient values between the flexible parachute canopy and the solid canopy models suggest that factors other than canopy geometry are related to the drag forces measured. The drag coefficients in this experiment remained relatively constant on the order of 1-1.4. The drag coefficients for the flexible canopy, however, ranged from about 0.46.4 , with a sharp spike at models 4,5 , and 6 . This spike is perhaps due to the time rate of change of the canopy opening.

## 8. Recommendations

This research project was the culmination of four years of education, encompassing a broad range of engineering topics. The design, construction, and testing procedures used in this experiment came both from prior experience, as well as hands-on trial and error. By no means was this group expert in all of these areas. Throughout the course of this project, many new concepts were learned. From learning new software, to becoming CNC machine operators, this project has welcomed many new experiences. From these experiences, this project group has recommendations to ease the creation and testing of similar models.

To begin, the machining of these models was incredibly difficult. The models needed to be very thin, yet precise. There is really no ideal material for this situation. Aluminum would have been more stable during the machining process, but it would have taken at least four times as long to cut compared to acetal. Due to the fact that acetal took nearly 45 hours per model to cut, this would be a substantial time investment. It is important to note that this is strictly cut time. This estimate does not consider the amount of time required to write the code, design the work holding, or set-up the machines, each of which was a very experimental process. Acetal also had its problems, as it became very unstable during machining. The consequence was broken models and/or tools resulting in lost time and money.

Due to this extreme difficulty, this project team recommends using stereolithography. Although in this academic setting, the availability of labor and machining facilities at no cost to the project budget made the use stereolithography uneconomical, it would have been much more cost effective and a wiser use of resources to use stereolithography if this project was done in a commercial setting. Using rapid prototyping, models can be constructed much more quickly, with very thin and accurate surfaces. At an estimated cost of $\$ 750-\$ 1000$ per model, the final
investment would be less than half of what was predicted for CNC machining in local machine shops.

However, if future models with thin walls and complex geometry are to be made using CNC machining, this project team has recommendations to make the process slightly more streamlined. To begin, be sure that the material stock ordered can fit and be held within all machines used in the manufacturing process. In several instances, unforeseen machine set-up configurations resulted in machine clearances that would not accept the size of stock ordered. This resulted in large time investments to alter the stock and/or become creative in the work holding techniques.

Secondly, the procedure outlined in the construction section of this document worked very well, provided someone could monitor the CNC machines at all times. Spindle speeds and feed rates needed constant monitoring due to extreme model vibrations, as the acetal became thinner and thinner. Contrary to normal convention, increasing the feed rate by 30 to $40 \%$ reduced vibrations at the same spindle speeds. It was found that this was due to increasing the chip load, giving the tool less chance of chattering out of the material.

Also, it is recommended to work very closely with the professional machinists. Countless hours were spent by the staff in the WPI machine shop helping this project team learn CAM (Computer Aided Machining) software, and teaching the operation of the CNC machines. By listening carefully, checking for machine availability, being responsible, cleaning the work area and following the rules, trust can be earned from the staff. Such work ethic earned this team great respect in the machine shop, allowing for increased CNC machine privileges with less direct supervision. This is essential when approaching deadlines need to be met.

It is also recommended that when testing in the WPI closed-loop wind tunnel, great attention be paid to the drag measurement readouts taken from the tunnel dynamometer. During preliminary testing of the dynamometer setup, it was noticed that the adjustment mechanism on the drag transducer can be moved to touch the inside of the transducer. When this happens, the transducer does not function properly and fails to accurately measure drag forces. Therefore, it is suggested that after the drag adjustment wheel is fine-tuned and secured with the set screw, it is checked to make sure that the adjustable part of the transducer is not touching the inside of the main transducer body. Also, it should be checked again after the dynamometer is installed in the test section.

## 9. Recommendations for Future Study

Great care was taken to ensure that the canopy models in this project accurately reflected the shape of the flexible canopy models at each time instance chosen. From the imaging and design, to the manufacturing and finishing processes, many hours were spent making sure that every detail was as accurate as possible. Therefore, these models should serve as the basis for future projects related to rigid canopy models.

These models would be an excellent starting point for the study of vortex shedding on solid models. This phenomenon was witnessed by the project team, both visually and quantitatively through the use of the LabVIEW VI. Flow characteristics surrounding each shape could also be analyzed to determine what wake effects may exist. These experiments would be even further validated by the fact that the canopy models were designed so that no mounting apparatus is subjected to the freestream flow in front of the model.

The data presented in this report could also be used as a comparison for flexible canopy experiments within the wind tunnel. The flexible canopy could be scaled to match the size of the models used here, and run at the same Reynolds numbers to double check the findings on differing drag coefficients for solid versus flexible canopies.

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## Appendices

Appendix A: Estimated dynamometer moment calculations.

ALUMINUM
DENSITY=0.0975 (LB/N ${ }^{\wedge} 3$ )


MOMENTS ABOUT DYNAMOMETER DISPLACEMENT TRANSDUCER

Max Torque in-lb
71.461

Max Torque ft-lb
5.955

Worst Case Values: Expected
Drag Force on models taken at
$28 \mathrm{~m} / \mathrm{s}$ assuming a Cd of 1.5
$28 \mathrm{~m} / \mathrm{s}$ assuming a Cd of 1.5
for all models. $28 \mathrm{~m} / \mathrm{s}$ is the
highest tunnel velocity where
expected drag forces on all
models are less than 7 lbs, the
models are less than 7 lbs, the
maximum drag force the
dynamometer is able to
measure

| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model | Torque in-lbs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.396 | 0.811 | 1.226 | 1.641 | 2.056 | 2.471 | 2.886 | 3.301 | 3.716 | 4.131 | 4.546 | 4.961 |
| 2 | 18.089 | 17.788 | 17.487 | 17.186 | 16.885 | 16.584 | 16.283 | 15.982 | 15.681 | 15.380 | 15.079 | 14.778 |
| 3 | 25.685 | 24.855 | 24.025 | 23.195 | 22.365 | 21.535 | 20.705 | 19.875 | 19.045 | 18.215 | 17.385 | 16.555 |
| 4 | 44.560 | 43.550 | 42.540 | 41.530 | 40.520 | 39.510 | 38.500 | 37.490 | 36.480 | 35.470 | 34.460 | 33.450 |
| 5 | 63.718 | 62.468 | 61.218 | 59.968 | 58.718 | 57.468 | 56.218 | 54.968 | 53.718 | 52.468 | 51.218 | 49.968 |
| 6 | 70.085 | 68.986 | 67.887 | 66.788 | 65.689 | 64.590 | 63.491 | 62.392 | 61.293 | 60.194 | 59.095 | 57.996 |
| 7 | 71.461 | 70.271 | 69.081 | 67.891 | 66.701 | 65.511 | 64.321 | 63.131 | 61.941 | 60.751 | 59.561 | 58.371 |
| 8 | 60.540 | 59.572 | 58.604 | 57.636 | 56.668 | 55.700 | 54.732 | 53.764 | 52.796 | 51.828 | 50.860 | 49.892 |
| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| Model | Torque ft-libs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.033 | 0.068 | 0.102 | 0.137 | 0.171 | 0.206 | 0.240 | 0.275 | 0.310 | 0.344 | 0.379 | 0.413 |
| 2 | 1.507 | 1.482 | 1.457 | 1.432 | 1.407 | 1.382 | 1.357 | 1.332 | 1.307 | 1.282 | 1.257 | 1.231 |
| 3 | 2.140 | 2.071 | 2.002 | 1.933 | 1.864 | 1.795 | 1.725 | 1.656 | 1.587 | 1.518 | 1.449 | 1.380 |
| 4 | 3.713 | 3.629 | 3.545 | 3.461 | 3.377 | 3.293 | 3.208 | 3.124 | 3.040 | 2.956 | 2.872 | 2.788 |
| 5 | 5.310 | 5.206 | 5.101 | 4.997 | 4.893 | 4.789 | 4.685 | 4.581 | 4.476 | 4.372 | 4.268 | 4.164 |
| 6 | 5.840 | 5.749 | 5.657 | 5.566 | 5.474 | 5.383 | 5.291 | 5.199 | 5.108 | 5.016 | 4.925 | 4.833 |
| 7 | 5.955 | 5.856 | 5.757 | 5.658 | 5.558 | 5.459 | 5.360 | 5.261 | 5.162 | 5.063 | 4.963 | 4.864 |
| 8 | 5.045 | 4.964 | 4.884 | 4.803 | 4.722 | 4.642 | 4.561 | 4.480 | 4.400 | 4.319 | 4.238 | 4.158 |

## DENSITY=0.0975 (LB/N^^3)

| Model | Weight (lbs) | CG from boss (in) |
| :---: | :---: | :---: |
| 1 | 0.415 | 3.741 |
| 2 | 0.301 | 3.490 |
| 3 | 0.830 | 3.620 |
| 4 | 1.010 | 3.356 |
| 5 | 1.250 | 3.256 |
| 6 | 1.099 | 3.221 |
| 7 | 1.190 | 2.377 |
| 8 | 0.968 | 3.128 |


| Model | Drag Force (lbs) |
| :---: | :---: |
| 1 | 0.131 |
| 2 | 1.62 |
| 3 | 2.46 |
| 4 | 4.08 |
| 5 | 5.802 |
| 6 | 6.227 |
| 7 | 6.29 |
| 8 | 5.378 |

MOMENTS ABOUT DYNAMOMETER DISPLACEMENT TRANSDUCER INCLUDES OSCILLATING MOMENT EFFECTS

| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model | Torque in-lbs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.352 | 0.767 | 1.182 | 1.597 | 2.012 | 2.427 | 2.842 | 3.257 | 3.672 | 4.087 | 4.502 | 4.917 |
| 2 | 18.629 | 18.328 | 18.027 | 17.726 | 17.425 | 17.124 | 16.823 | 16.522 | 16.221 | 15.920 | 15.619 | 15.318 |
| 3 | 26.505 | 25.675 | 24.845 | 24.015 | 23.185 | 22.355 | 21.525 | 20.695 | 19.865 | 19.035 | 18.205 | 17.375 |
| 4 | 45.920 | 44.910 | 43.900 | 42.890 | 41.880 | 40.870 | 39.860 | 38.850 | 37.840 | 36.830 | 35.820 | 34.810 |
| 5 | 65.652 | 64.402 | 63.152 | 61.902 | 60.652 | 59.402 | 58.152 | 56.902 | 55.652 | 54.402 | 53.152 | 51.902 |
| 6 | 72.161 | 71.062 | 69.963 | 68.864 | 67.765 | 66.666 | 65.567 | 64.468 | 63.369 | 62.270 | 61.171 | 60.072 |
| 7 | 73.558 | 72.368 | 71.178 | 69.988 | 68.798 | 67.608 | 66.418 | 65.228 | 64.038 | 62.848 | 61.658 | 60.468 |
| 8 | 62.333 | 61.365 | 60.397 | 59.429 | 58.461 | 57.493 | 56.525 | 55.557 | 54.589 | 53.621 | 52.653 | 51.685 |
| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| Model | Torque ft-lbs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.029 | 0.064 | 0.098 | 0.133 | 0.168 | 0.202 | 0.237 | 0.271 | 0.306 | 0.341 | 0.375 | 0.410 |
| 2 | 1.552 | 1.527 | 1.502 | 1.477 | 1.452 | 1.427 | 1.402 | 1.377 | 1.352 | 1.327 | 1.302 | 1.276 |
| 3 | 2.209 | 2.140 | 2.070 | 2.001 | 1.932 | 1.863 | 1.794 | 1.725 | 1.655 | 1.586 | 1.517 | 1.448 |
| 4 | 3.827 | 3.743 | 3.658 | 3.574 | 3.490 | 3.406 | 3.322 | 3.238 | 3.153 | 3.069 | 2.985 | 2.901 |
| 5 | 5.471 | 5.367 | 5.263 | 5.158 | 5.054 | 4.950 | 4.846 | 4.742 | 4.638 | 4.533 | 4.429 | 4.325 |
| 6 | 6.013 | 5.922 | 5.830 | 5.739 | 5.647 | 5.555 | 5.464 | 5.372 | 5.281 | 5.189 | 5.098 | 5.006 |
| 7 | 6.130 | 6.031 | 5.932 | 5.832 | 5.733 | 5.634 | 5.535 | 5.436 | 5.337 | 5.237 | 5.138 | 5.039 |
| 8 | 5.194 | 5.114 | 5.033 | 4.952 | 4.872 | 4.791 | 4.710 | 4.630 | 4.549 | 4.468 | 4.388 | 4.307 |

6.130

Worst Case Values: Expected
Drag Force on models taken at
$28 \mathrm{~m} / \mathrm{s}$ assuming a Cd of 1.5
for all models. $28 \mathrm{~m} / \mathrm{s}$ is the
tor all models. $28 \mathrm{~m} / \mathrm{sis}$ the
highest tunnel velocity where
expected drag forces on all
models are less than 7 lbs, the
maximum drag force the
nometer is ab
measure

## DENSITY=0.0513 (LB/N^3)

| Model | Weight (lbs) | CG from boss (in) |
| :---: | :---: | :---: |
| 1 | 0.218 | 3.741 |
| 2 | 0.260 | 3.490 |
| 3 | 0.436 | 3.620 |
| 4 | 0.530 | 3.356 |
| 5 | 0.658 | 3.725 |
| 6 | 0.578 | 3.221 |
| 7 | 0.627 | 2.377 |
| 8 | 0.509 | 3.128 |


| Model | Drag Force (lbs) |
| :---: | :---: |
| 1 | 0.131 |
| 2 | 1.62 |
| 3 | 2.46 |
| 4 | 4.08 |
| 5 | 5.802 |
| 6 | 6.227 |
| 7 | 6.29 |
| 8 | 5.378 |

MOMENTS ABOUT DYNAMOMETER DISPLACEMENT TRANSDUCER

Worst Case Values:
Expected Drag Force on
models taken at $28 \mathrm{~m} / \mathrm{s}$
models. $28 \mathrm{~m} / \mathrm{s}$ is the
highest tunnel velocity where
expected drag forces on all
expected drag forces on all
the maximum drag force the
dynamometer is able to
measure

| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model | Torque in-lbs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.538 | 0.320 | 0.102 | 0.116 | 0.334 | 0.552 | 0.770 | 0.988 | 1.206 | 1.424 | 1.642 | 1.860 |
| 2 | 18.273 | 18.013 | 17.753 | 17.493 | 17.233 | 16.973 | 16.713 | 16.453 | 16.193 | 15.933 | 15.673 | 15.413 |
| 3 | 27.506 | 27.070 | 26.634 | 26.198 | 25.762 | 25.326 | 24.890 | 24.454 | 24.018 | 23.582 | 23.146 | 22.710 |
| 4 | 46.651 | 46.121 | 45.591 | 45.061 | 44.531 | 44.001 | 43.471 | 42.941 | 42.411 | 41.881 | 41.351 | 40.821 |
| 5 | 66.515 | 65.857 | 65.199 | 64.541 | 63.883 | 63.225 | 62.567 | 61.909 | 61.251 | 60.593 | 59.935 | 59.277 |
| 6 | 72.284 | 71.706 | 71.128 | 70.550 | 69.972 | 69.394 | 68.816 | 68.238 | 67.660 | 67.082 | 66.504 | 65.926 |
| 7 | 73.363 | 72.736 | 72.109 | 71.482 | 70.855 | 70.228 | 69.601 | 68.974 | 68.347 | 67.720 | 67.093 | 66.466 |
| 8 | 62.435 | 61.926 | 61.417 | 60.908 | 60.399 | 59.890 | 59.381 | 58.872 | 58.363 | 57.854 | 57.345 | 56.836 |
| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 |
| Model | Torque ft-lbs |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.045 | 0.027 | 0.009 | 0.010 | 0.028 | 0.046 | 0.064 | 0.082 | 0.100 | 0.119 | 0.137 | 0.155 |
| 2 | 1.523 | 1.501 | 1.479 | 1.458 | 1.436 | 1.414 | 1.393 | 1.371 | 1.349 | 1.328 | 1.306 | 1.284 |
| 3 | 2.292 | 2.256 | 2.219 | 2.183 | 2.147 | 2.110 | 2.074 | 2.038 | 2.001 | 1.965 | 1.929 | 1.892 |
| 4 | 3.888 | 3.843 | 3.799 | 3.755 | 3.711 | 3.667 | 3.623 | 3.578 | 3.534 | 3.490 | 3.446 | 3.402 |
| 5 | 5.543 | 5.488 | 5.433 | 5.378 | 5.324 | 5.269 | 5.214 | 5.159 | 5.104 | 5.049 | 4.995 | 4.940 |
| 6 | 6.024 | 5.976 | 5.927 | 5.879 | 5.831 | 5.783 | 5.735 | 5.687 | 5.638 | 5.590 | 5.542 | 5.494 |
| 7 | 6.114 | 6.061 | 6.009 | 5.957 | 5.905 | 5.852 | 5.800 | 5.748 | 5.696 | 5.643 | 5.591 | 5.539 |
| 8 | 5.203 | 5.160 | 5.118 | 5.076 | 5.033 | 4.991 | 4.948 | 4.906 | 4.864 | 4.821 | 4.779 | 4.736 |

DELRIN
DENSITY $=0.0513($ LB/N^3)

| Model | Weight (lbs) | CG from boss (in) |
| :---: | :---: | :---: |
| 1 | 0.218 | 3.741 |
| 2 | 0.260 | 3.490 |
| 3 | 0.436 | 3.630 |
| 4 | 0.530 | 3.356 |
| 5 | 0.658 | 3.725 |
| 6 | 0.578 | 3.221 |
| 7 | 0.627 | 2.377 |
| 8 | 0.509 | 3.128 |


| Model | Drag Force (lbs) |
| :---: | :---: |
| 1 | 0.131 |
| 2 | 1.62 |
| 3 | 2.46 |
| 4 | 4.08 |
| 5 | 5.802 |
| 6 | 6.227 |
| 7 | 6.29 |
| 8 | 5.378 |

MOMENTS ABOUT DYNAMOMETER DISPLACEMENT TRANSDUCER

NCLUDES OSCILLATING MOMENT EFFECTS

| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Model | Torque in-lbs |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.582 | 0.364 | 0.146 | 0.072 | 0.290 | 0.508 | 0.726 | 0.944 | 1.162 | 1.380 | 1.598 |
| 2 | 18.813 | 18.553 | 18.293 | 18.033 | 17.773 | 17.513 | 17.253 | 16.993 | 16.733 | 16.473 | 16.21 |
| 3 | 28.326 | 27.890 | 27.454 | 27.018 | 26.582 | 26.146 | 25.710 | 25.274 | 24.838 | 24.402 | 23.966 |
| 4 | 48.011 | 47.481 | 46.951 | 46.421 | 45.891 | 45.361 | 44.831 | 44.301 | 43.771 | 43.241 | 42.711 |
| 5 | 68.449 | 67.791 | 67.133 | 66.475 | 65.817 | 65.159 | 64.501 | 63.843 | 63.185 | 62.527 | 61.869 |
| 6 | 74.360 | 73.782 | 73.204 | 72.626 | 72.048 | 71.470 | 70.892 | 70.314 | 69.736 | 69.158 | 68.580 |
| 7 | 75.459 | 74.832 | 74.205 | 73.578 | 72.951 | 72.324 | 71.697 | 71.070 | 70.443 | 69.816 | 69.189 |
| 8 | 64.228 | 63.719 | 63.210 | 62.701 | 62.192 | 61.683 | 61.174 | 60.665 | 60.156 | 59.647 | 59.138 |
| Sting Length inches | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 |
| Model | Torque ft-lbs |  |  |  |  |  |  |  |  |  |  |
| 1 | 0.049 | 0.030 | 0.012 | 0.006 | 0.024 | 0.042 | 0.060 | 0.079 | 0.097 | 0.115 | 0.133 |
| 2 | 1.568 | 1.546 | 1.524 | 1.503 | 1.481 | 1.459 | 1.438 | 1.416 | 1.394 | 1.373 | 1.351 |
| 3 | 2.360 | 2.324 | 2.288 | 2.251 | 2.215 | 2.179 | 2.142 | 2.106 | 2.070 | 2.033 | 1.997 |
| 4 | 4.001 | 3.957 | 3.913 | 3.868 | 3.824 | 3.780 | 3.736 | 3.692 | 3.648 | 3.603 | 3.559 |
| 5 | 5.704 | 5.649 | 5.594 | 5.540 | 5.485 | 5.430 | 5.375 | 5.320 | 5.265 | 5.211 | 5.156 |
| 6 | 6.197 | 6.148 | 6.100 | 6.052 | 6.004 | 5.956 | 5.908 | 5.859 | 5.811 | 5.763 | 5.715 |
| 7 | 6.288 | 6.236 | 6.184 | 6.132 | 6.079 | 6.027 | 5.975 | 5.923 | 5.870 | 5.818 | 5.766 |
| 8 | 5.352 | 5.310 | 5.267 | 5.225 | 5.183 | 5.140 | 5.098 | 5.055 | 5.013 | 4.971 | 4.928 |

Max Torque in
75.459
ax Torque ft-lb
6.288

Worst Case Values:
Expected Drag Force on
assuming a Cd of 1.5 for al
assuming a Cd of 1.5 for
models. $28 \mathrm{~m} / \mathrm{s}$ is the
highest tunnel velocity where
expected drag forces on all
models are less than 7 lbs
the maximum drag force the dynamometer is able to
measure

## Appendix B: Wall Interference Correction Calculations

## METHOD OF WALL CORRECTIONS

Macha \& Buffington (AIAA Journal of Aircraft 1989)
Wall-Interference Corrections for Parachutes in Closed Wind Tunnel
"The results are believed to be applicable to any parachute or circular or conical constructions"

## Maskell Correction Method for Bluff Bodies

$$
\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}
$$

| Nomenclature |
| :--- |
| C tunnel cross-sectional area <br> $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ model drag area $\mathrm{D} / \mathrm{q}$ <br> $\mathrm{K}_{\mathrm{M}}$ Maskell bluff-body blockage factor <br> q freestream dynamic pressure |

subscript
u uncorrected for wall interference

Method of Wall Correction
SINGLE PARACHUTE

| $\mathrm{K}_{\mathrm{M}}$ | Description | approx $C_{D}$ of parachutes tested by Macha |
| :---: | :---: | :---: |
| 1.85 | parachute (independent of porosity) | 1.35 corrected drag coefficient based on solid frontal area of inflated canopy |
| 2.8 | nonporous disk |  |

## MODEL 1

WALL CORRECTIONS

$\left\lvert\,$| Model 7: Frontal Area | 0.00083 | $\mathrm{~m}^{2}$ |
| :--- | ---: | :---: | | Freestream Velocity: | $\mathbf{1 0} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 1 |  | Units |
| C | 0.0137 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{q}_{\mathrm{u}}$ | 1.85 |  |
| $\mathrm{~K}_{\mathrm{M}}$ | 0.0109 | $\mathrm{ft}^{2}$ |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | $\mathbf{1 . 2 6 5 7}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| q | $\mathbf{1 0 . 0 2 5 1}$ | $\mathrm{m} / \mathrm{s}$ |
| Resultant Tunnel Velocity <br> (above model $)$ | 0.0251 | $\mathrm{~m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | $\mathbf{1 . 2 1 9 2}$ |  |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected |  | $\mathbf{1 . 2 1 3 1}$ |
| $\mathbf{C}_{\mathrm{D}}$ corrected |  |  |\right.


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 1.26 | 10.01 | 0.01 | 0.1 |
| 1.5 | 1.26 | 10.02 | 0.02 | 0.2 |
| 1.85 | 1.27 | 10.03 | 0.03 | 0.3 |
| 2 | 1.27 | 10.03 | 0.03 | 0.3 |
| 3 | 1.27 | 10.04 | 0.04 | 0.4 |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |  |
| :---: | :---: | :---: |
| Model 1 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.0228 | lbs |
| $\mathrm{qu}_{u}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{K}_{\text {M }}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.0056 | $\mathrm{ft}^{2}$ |
| q | 4.0909 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 18.0232 | m/s |
| $\Delta \mathrm{V}$ | 0.0232 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 0.6276 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 0.6260 |  |


\section*{| RE \# |
| :---: |
| 21636.74 |
| 21690.98 | <br> 21690.98}


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 4.09 | 18.01 | 0.01 | 0.1 |
| 1.5 | 4.09 | 18.02 | 0.02 | 0.1 |
| 1.85 | 4.09 | 18.02 | 0.02 | 0.1 |
| 2 | 4.09 | 18.03 | 0.03 | 0.1 |
| 3 | 4.10 | 18.04 | 0.04 | 0.2 |


| Freestream Velocity: | $\mathbf{2 5} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 1 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 0.0762 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 7.8711 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.0097 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{7 . 9 0 6 4}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{2 5 . 0 5 5 9}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V} \quad \mathrm{C}_{\mathrm{D}}$ uncorrected | 0.0559 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathbf{C}_{\mathrm{D}}$ corrected |  | $\mathbf{1 . 0 8 8 0}$ |

## RE \# <br> 54091.86

| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 7.89 | 25.03 | 0.03 | 0.1 |
| 1.5 | 7.90 | 25.05 | 0.05 | 0.2 |
| 1.85 | 7.9064 | 25.06 | 0.06 | 0.2 |
| 2 | 7.91 | 25.06 | 0.06 | 0.2 |
| 3 | 7.93 | 25.09 | 0.09 | 0.4 |

$$
\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}
$$

Nomenclature

| C | tunnel cross-sectional area |
| :---: | :--- |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | model drag area $\mathrm{D} / \mathrm{q}$ |
| $\mathrm{K}_{\mathrm{M}}$ | Maskell bluff-body blockage factor |
| q | freestream dynamic pressure |


| subscript |
| :---: |


| u |
| :---: |

ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :---: | :---: |
| Model 1 |  |
| Drag Force error +/- | 0.018769449 Ib |
| Total Error | 137.3795359\% |
| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| Model 1 |  |
| Drag Force error +/- | 0.019454502 lb |
| Total Error | 85.35205881 \% |
| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |
| Model 1 |  |
| Drag Force error +/- | 0.01863147 b |
| Total Error | $24.44827619 \%$ |

## MODEL 2

## WALL CORRECTIONS

| Model 2: Frontal Area | 0.01021 | $\mathrm{m}^{2}$ |
| :---: | :---: | :---: |
| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |  |
| Model 2 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.2148 | Ibs |
| $\mathrm{qu}_{4}$ | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| KM | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.1706 | $\mathrm{ft}^{2}$ |
| q | 1.3587 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 10.3870 | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 0.3870 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.5526 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.4391 |  |


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 2 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 0.7112 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.1743 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{4 . 4 0 9 3}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{1 8 . 7 1 1 4}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 0.7114 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathbf{C}_{\mathrm{D}}$ uncorrected | $\mathbf{1 . 5 8 6 4}$ |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected |  | $\mathbf{1 . 4 6 8 0}$ |


$\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}$

ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 2 |  |


| Drag Force error $+/-$ | $0.020086319 \mid \mathrm{lb}$ |
| :--- | :--- |


| Total Error | $10.22231855 \%$ |
| :--- | :--- |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 2 |  |


| Drag Force error +/- | 0.021034024 lb |
| :--- | :--- | :--- |


| Total Error | 3.222241252 |
| :--- | :--- |

## Freestream Velocity: <br> Model 2 <br> $25 \mathrm{~m} / \mathrm{s}$

Drag Force error +/- 0.02263605 Ib

| Total Error | $1.739754028 \%$ |
| :--- | :--- |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 4.26 | 18.39 | 0.39 |
| 1.5 | 4.35 | 18.58 | 0.58 |
| 1.85 | 4.41 | 18.71 | 0.71 |
| 2 | 4.44 | 18.77 | 0.77 |
| 3 | 4.61 | 19.14 | 1.14 |

RE \#
142218.1

| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 1.31 | 10.21 | 0.21 |
| 1.5 | 1.34 | 10.31 | 0.31 |
| 1.85 | 1.36 | 10.39 | 0.39 |
| 2 | 1.37 | 10.42 | 0.42 |
| 3 | 1.42 | 10.62 | 0.62 |

## $\begin{array}{r}\text { RE \# } \\ \hline 76006 \\ \hline 78947.33 \\ \hline\end{array}$ <br> 78947.33

| $\mathrm{K}_{M}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 8.22 | 25.55 | 0.55 |
| 1.5 | 8.40 | 25.82 | 0.82 |
| 1.85 | 8.5222 | 26.01 | 1.01 |
| 2 | 8.58 | 26.09 | 1.09 |
| 3 | 8.93 | 26.62 | 1.62 |

RE \#
190015

Nomenclature

| C | tunnel cross-sectional area model drag area D/q Maskell bluff-body blockage factoı freestream dynamic pressure |
| :---: | :---: |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ |  |
| $\mathrm{K}_{\text {M }}$ |  |
| q |  |
| subscript |  |
| u | uncorrected for wall interference |

## MODEL 3

WALL CORRECTIONS

| Model 7: Frontal Area | 0.01523 | $\mathrm{m}^{2}$ |
| :---: | :---: | :---: |
| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |  |
| Model 3 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.3674 | lbs |
| $\mathrm{q}_{4}$ | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| KM | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.2917 | $\mathrm{ft}^{2}$ |
| q | 1.4293 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 10.6532 | m/s |
| $\Delta \mathrm{V}$ | 0.6532 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.7795 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.5679 |  |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 1.35 | 10.36 | 0.36 |
| 1.5 | 1.40 | 10.53 | 0.53 |
| 1.85 | 1.43 | 10.65 | 0.65 |
| 2 | 1.44 | 10.70 | 0.70 |
| 3 | 1.53 | 11.04 | 1.04 |


\section*{| RE \# |
| :--- |
| 92834.58 |
| 98898 |}


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 3 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 1.1297 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.2769 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{4 . 6 0 2 9}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{1 9 . 1 1 7 8}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 1.1178 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathbf{C}_{\mathrm{D}}$ uncorrected | $\mathbf{1 . 6 8 9 0}$ |  |
| $\mathbf{C}_{\mathrm{D}}$ corrected | $\mathbf{1 . 4 9 7 3}$ |  |
|  |  |  |

RE \#
167102.2

177479


\section*{| RE \# |
| :---: |
| 232086.4 |
| 24679.5 | <br> 246794.5}

Nomenclature

| C | tunnel cross-sectional area |
| :---: | :--- |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | model drag area $\mathrm{D} / \mathrm{q}$ |
| $\mathrm{K}_{\mathrm{M}}$ | naskell bluff-body blockage factor |
| q | freestream dynamic pressure |


| subscript |  |
| :---: | :--- |
| U | uncorrected for wall interference |

ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 3 |  |


| Drag Force error $+/-$ | 0.017431498 | lb |
| :--- | :--- | :--- |


| Total Error | $6.292070952 \%$ |
| :--- | :--- |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 3 |  |


| Drag Force error $+/-$ | 0.022334747 lb |
| :--- | :--- | :--- |


| Total Error | $2.354513841 \%$ |
| :--- | :--- |


| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 3 |  |

Drag Force error +/- 0.04137108 Ib

| Total Error | $1.974223212 \%$ |
| :--- | :--- |

## MODEL 4

WALL CORRECTIONS

| Model 4: Frontal Area | 0.02577 | $\mathrm{m}^{2}$ |
| :---: | :---: | :---: |
| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |  |
| Model 4 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.5221 | Ibs |
| $\mathrm{q}_{4}$ | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{K}_{\text {M }}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.4146 | $\mathrm{ft}^{2}$ |
| q | 1.5008 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\begin{array}{\|l} \hline \text { Resultant Tunnel Velocity } \\ \text { (above model) } \\ \hline \end{array}$ | 10.9167 | m/s |
| $\triangle \mathrm{V}$ | 0.9167 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.4945 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.2541 |  |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 1.39 | 10.51 | 0.51 | 5.1 |
| 1.5 | 1.46 | 10.75 | 0.75 | 7.5 |
| 1.85 | 1.50 | 10.92 | 0.92 | 9.2 |
| 2 | 1.52 | 10.99 | 0.99 | 9.9 |
| 3 | 1.65 | 11.45 | 1.45 | 14.5 |


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model $\mathbf{4}$ |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 1.6317 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.3999 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{4 . 8 3 5 0}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{1 9 . 5 9 4 0}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 1.5940 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | $\mathbf{1 . 4 4 1 6}$ |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | $\mathbf{1 . 2 1 6 6}$ |  |


\section*{| RE \# |
| :--- |
| 120758.9 |
| 131828.6 |}


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 4.49 | 18.88 | 0.88 | 4.9 |
| 1.5 | 4.69 | 19.30 | 1.30 | 7.2 |
| 1.85 | 4.84 | 19.59 | 1.59 | 8.9 |
| 2 | 4.90 | 19.72 | 1.72 | 9.5 |
| 3 | 5.30 | 20.52 | 2.52 | 14.0 |


$\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}$

\section*{| RE \# |
| :--- |
| 217366.1 |
| 236614.7 |}


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 8.71 | 26.30 | 1.30 | 5.2 |
| 1.5 | 9.13 | 26.93 | 1.93 | 7.7 |
| 1.85 | 9.4290 | 27.36 | 2.36 | 9.5 |
| 2 | 9.56 | 27.55 | 2.55 | 10.2 |
| 3 | 10.40 | 28.73 | 3.73 | 14.9 |


\section*{| RE \# |
| :--- |
| 301897.3 | <br> 330426.7}

## Nomenclature

| C | tunnel cross-sectional area |
| :---: | :--- |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | model drag area $\mathrm{D} / \mathrm{q}$ |
| $\mathrm{K}_{\mathrm{M}}$ | Maskell bluff-body blockage factor |
|  | freestream dynamic pressure |

[^0]ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 4 |  |


| Drag Force error +/- | 0.0173816 | lb |
| :--- | :--- | :--- |


| Total Error | 5.306483627 \% |
| :--- | :--- |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 4 |  |


| Drag Force error $+/-$ | $0.031428119 \mid \mathrm{lb}$ |
| :--- | :--- |


| Total Error | 2.311941572 |
| :--- | :--- |


| Freestream Velocity: | $\mathbf{2 5 ~ m} / \mathbf{s}$ |
| :--- | :--- |
| Model 4 |  |
| Drag Force error +/- | $0.273717929 \mid \mathrm{lb}$ |


| Total Error | $8.152910024 \%$ |
| :--- | :--- |

## MODEL 5

WALL CORRECTIONS

$|$| Model 5: Frontal Area | 0.03658 | $\mathrm{~m}^{2}$ |
| :--- | ---: | :---: |
| Freestream Velocity: $\mathbf{1 0} \mathbf{~ m / s}$     <br> Model 5  Units    <br> C 4 $\mathrm{ft}^{2}$    <br> $\mathrm{D}=$ Drag Force 0.8202 lbs    <br> $\mathrm{q}_{\mathrm{u}}$ 1.2594 $\mathrm{lb} / \mathrm{ft}^{2}$    <br> $\mathrm{~K}_{\mathrm{M}}$ 1.85     <br> $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ 0.6513 $\mathrm{ft}^{2}$    <br> q $\mathbf{1 . 6 3 8 7}$ $\mathrm{lb} / \mathrm{ft}^{2}$    <br> Resultant Tunnel Velocity <br> (above model) $\mathbf{1 1 . 4 0 7 0}$ $\mathrm{m} / \mathrm{s}$    <br> $\Delta \mathrm{V}$ 1.4070 $\mathrm{~m} / \mathrm{s}$    <br> $\mathrm{C}_{\mathrm{D}}$ uncorrected    $\mathbf{1 . 6 5 4 0}$  <br> $\mathrm{C}_{\mathrm{D}}$ corrected  $\mathbf{1 . 2 7 1 2}$    |  |  |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 1.46 | 10.78 | 0.78 |
| 1.5 | 1.57 | 11.15 | 1.15 |
| 1.85 | 1.64 | 11.41 | 1.41 |
| 2 | 1.67 | 11.51 | 1.51 |
| 3 | 1.87 | 12.20 | 2.20 |


\section*{| RE \# |
| :---: |
| 143875.1 | <br> | 1638718.9 |
| :--- |}


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 5 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 2.6396 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.6469 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{5 . 3 0 1 2}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\begin{array}{l}\text { Resultant Tunnel Velocity } \\ \text { (above model) }\end{array}$ | $\mathbf{2 0 . 5 1 6 8}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 2.5168 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathbf{C}_{\mathrm{D}}$ uncorrected |  | $\mathbf{1 . 6 4 3 0}$ |$]$

## RE \# <br> 258975.2

295185.5

| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |  |
| :---: | :---: | :---: |
| Model 5 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 5.0464 | Ibs |
| $\mathrm{q}_{\mathrm{u}}$ | 7.8711 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.6411 | $\mathrm{ft}^{2}$ |
| q | 10.2051 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 28.4662 | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 3.4662 | $\mathrm{m} / \mathrm{s}$ |
| $C_{D}$ uncorrected | 1.6283 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.2559 |  |

$$
\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}
$$

ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 5 |  |


| Total Error | $4.628441646 \%$ |
| :--- | :--- |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 5 |  |


| Drag Force error $+/-$ | 0.02653258 | Ib |
| :--- | :---: | :--- |
| Total Error | $1.626536606 \%$ |  |


| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 5 |  |

Drag Force error +/- 0.086218022 Ib

| Total Error | $1.833158724 \%$ |
| :--- | :--- |

## MODEL 7

WALL CORRECTIONS

| Model 7: Frontal Area | 0.04264 | $\mathrm{m}^{2}$ |
| :---: | :---: | :---: |
| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |  |
| Model 7 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.9576 | lbs |
| $\mathrm{q}_{4}$ | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| KM | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.7604 | $\mathrm{ft}^{2}$ |
| q | 1.7023 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 11.6262 | m/s |
| $\triangle \mathrm{V}$ | 1.6262 | m/s |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.6567 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.2256 |  |


| $\mathrm{K}_{\text {M }}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 1.50 | 10.91 | 0.91 | 9.1 |
| 1.5 | 1.62 | 11.34 | 1.34 | 13.4 |
| 1.85 | 1.70 | 11.63 | 1.63 | 16.3 |
| 2 | 1.74 | 11.75 | 1.75 | 17.5 |
| 3 | 1.98 | 12.53 | 2.53 | 25.3 |


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 7 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 3.3277 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.8155 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{5 . 6 1 9 5}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{2 1 . 1 2 3 7}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 3.1237 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | $\mathbf{1 . 7 7 6 8}$ |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected |  | $\mathbf{1 . 2 9 0 2}$ |


\section*{| RE \# |
| :--- |
| 268295.6 |
| 31854.8 | <br> 314854.8}


| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |  |
| :---: | :---: | :---: |
| Model 7 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 5.7860 | Ibs |
| $\mathrm{q}_{4}$ | 7.8711 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{K}_{\text {M }}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.7351 | $\mathrm{ft}^{2}$ |
| q | 10.5471 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 28.9394 | m/s |
| $\Delta \mathrm{V}$ | 3.9394 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.6015 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.1952 |  |


\section*{| RE \# |
| :--- |
| 372632.8 |
| 431350.4 |}


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\%$ change |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 9.32 | 27.20 | 2.20 | 8.8 |
| 1.5 | 10.04 | 28.24 | 3.24 | 12.9 |
| 1.85 | 10.5471 | 28.94 | 3.94 | 15.8 |
| 2 | 10.76 | 29.24 | 4.24 | 16.9 |
| 3 | 12.21 | 31.14 | 6.14 | 24.6 |

$\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}$


ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 7 |  |


| Drag Force error $+/-$ | 0.016436322 | Ib |
| :--- | :--- | :--- |


| Total Error | $4.474496708 \%$ |
| :--- | :--- |


| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| :--- | :--- |
| Model 7 |  |


| Drag Force error +/- | 0.031732667 | lb |
| :--- | :--- | :--- |


| Total Error | $1.595174764 \%$ |
| :--- | :--- |


| Freestream Velocity: | $\mathbf{2 5 ~ m} / \mathrm{s}$ |
| :--- | :--- |
| Model 7 |  |
| Drag Force error $+/-$ 0.021588262 <br> lb  |  |

## MODEL 8

## WALL CORRECTIONS

| Model 8: Frontal Area | 0.03390 | $\mathrm{m}^{2}$ |
| :---: | :---: | :---: |
| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |  |
| Model 8 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 0.8213 | Ibs |
| $\mathrm{q}_{4}$ | 1.2594 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{K}_{\text {M }}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.6522 | $\mathrm{ft}^{2}$ |
| q | 1.6392 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity (above model) | 11.4089 | m/s |
| $\triangle \mathrm{V}$ | 1.4089 | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.7870 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.3729 |  |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 1.46 | 10.78 | 0.78 |
| 1.5 | 1.57 | 11.16 | 1.16 |
| 1.85 | 1.64 | 11.41 | 1.41 |
| 2 | 1.67 | 11.52 | 1.52 |
| 3 | 1.88 | 12.20 | 2.20 |


| Freestream Velocity: | $\mathbf{1 8} \mathbf{~ m / s}$ |  |
| :--- | ---: | :---: |
| Model 8 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| $\mathrm{D}=$ Drag Force | 2.4858 | lbs |
| $\mathrm{q}_{\mathrm{u}}$ | 4.0804 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{~K}_{\mathrm{M}}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.6092 | $\mathrm{ft}^{2}$ |
| q | $\mathbf{5 . 2 3 0 1}$ | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| Resultant Tunnel Velocity <br> (above model) | $\mathbf{2 0 . 3 7 8 7}$ | $\mathrm{m} / \mathrm{s}$ |
| $\Delta \mathrm{V}$ | 2.3787 | $\mathrm{~m} / \mathrm{s}$ |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | $\mathbf{1 . 6 6 9 3}$ |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected |  | $\mathbf{1 . 3 0 2 4}$ |


| RE \# |
| :--- |
| 155340.7 |
| 177226.4 |


| $\mathrm{K}_{\mathrm{M}}$ | $\mathrm{q}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ | $\mathrm{V}(\mathrm{m} / \mathrm{s})$ | $\Delta \mathrm{V}(\mathrm{m} / \mathrm{s})$ |
| :---: | :---: | :---: | :---: |
| 1 | 4.70 | 19.32 | 1.32 |
| 1.5 | 5.01 | 19.95 | 1.95 |
| 1.85 | 5.23 | 20.38 | 2.38 |
| 2 | 5.32 | 20.56 | 2.56 |
| 3 | 5.94 | 21.73 | 3.73 |


| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |  |
| :---: | :---: | :---: |
| Model 8 |  | Units |
| C | 4 | $\mathrm{ft}^{2}$ |
| D = Drag Force | 4.8443 | Ibs |
| $\mathrm{q}_{\mathrm{u}}$ | 7.8711 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| K ${ }^{\text {m }}$ | 1.85 |  |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | 0.6155 | $\mathrm{ft}^{2}$ |
| q | 10.1116 | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\begin{array}{\|l} \hline \begin{array}{l} \text { Resultant Tunnel Velocity } \\ \text { (above model) } \end{array} \\ \hline \end{array}$ | 28.3356 | m/s |
| $\Delta \mathrm{V}$ | 3.3356 | m/s |
| $\mathrm{C}_{\mathrm{D}}$ uncorrected | 1.6864 |  |
| $\mathrm{C}_{\mathrm{D}}$ corrected | 1.3128 |  |

## RE \# <br> 388351.8 <br> 440166.9

$$
\frac{q}{q_{u}}=1+K_{M} \frac{C_{D} S_{u}}{C}
$$

Nomenclature

| C | tunnel cross-sectional area |
| :---: | :--- |
| $\mathrm{C}_{\mathrm{D}} \mathrm{S}$ | model drag area $\mathrm{D} / \mathrm{q}$ |
| $\mathrm{K}_{\mathrm{M}}$ | Maskell bluff-body blockage factoı |
| q | freestream dynamic pressure |


| subscript |  |
| :--- | :--- |
| u | uncorrected for wall interference |

ERROR

| Freestream Velocity: | $10 \mathrm{~m} / \mathrm{s}$ |
| :---: | :---: |
| Model 8 |  |
| Drag Force error +/- | 0.018501385 \|lb |
| Total Error | $4.706347692 \%$ |
| Freestream Velocity: | $18 \mathrm{~m} / \mathrm{s}$ |
| Model 8 |  |
| Drag Force error +/- | 0.023994115 \|lb |
| Total Error | 1.602168526\% |
| Freestream Velocity: | $25 \mathrm{~m} / \mathrm{s}$ |
| Model 8 |  |
| Drag Force error +/- | $0.029245565 \square$ |
| Total Error | $0.897753905 \%$ |

Appendix C: Estimated expected drag forces for rigid canopy models for varying tunnel speeds and drag coefficients.
drag force calculations for various wind tunnel freestream velocities and drag coefficients.
IMAGE 1
blockage Less than $10 \%$

| Diameter Dprojected $(\mathrm{m})$ | 0.03245 |
| :--- | ---: |
| Area Projected $\left(\mathrm{m}^{2}\right)$ | 0.00083 |
| Density $\left(\mathrm{Kg} / \mathrm{m}^{3}\right)$ | 1.2 |
| Mu $\left(\mathrm{N}\right.$ s $\left./ \mathrm{m}^{2}\right)$ | $1.80 \mathrm{E}-05$ |
| Kinematic Viscosity | $1.50 \mathrm{E}-05$ |


| Kinematic Viscosity | 1.50 |  |  | Cd |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | 0.5 | 0.7 | 0.8 | 0.85 | 0.9 | 1 | 1.1 | 1.2 | 1.3 | 1.4 | 1.5 | 1.6 | 2 | 2.5 | 3 | 3.5 |
| $\frac{\text { Tunnel Speed }}{\mathrm{m} / \mathrm{s}}$ | MPH | Hz | RE\# | $\frac{\text { Force }}{\text { lbs }}$ | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | libs | lbs | lbs | lbs | lbs | lbs |
| 0 | 0.0 |  | $0.00 \mathrm{E}+00$ | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 |
| 1 | 2.2 |  | ${ }^{2.16 E+03}$ | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 |
| 2 | 4.5 | 2 | $4.33 E+03$ | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.002 |
| 3 | 6.7 | 2 | $6.49 \mathrm{E}+03$ | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.001 | 0.002 | 0.002 | 0.002 | 0.003 | 0.003 | 0.004 |
| 4 | 8.9 | 4 | $8.65 \mathrm{E}+03$ | 0.001 | 0.001 | 0.001 | 0.002 | 0.002 | 0.002 | 0.002 | 0.002 | 0.002 | 0.002 | 0.003 | 0.003 | 0.004 | 0.004 | 0.005 | 0.006 |
| 5 | 11.2 |  | $1.08 \mathrm{E}+04$ | 0.001 | 0.002 | 0.002 | 0.002 | 0.003 | 0.003 | 0.003 | 0.003 | 0.004 | 0.004 | 0.004 | 0.004 | 0.006 | 0.007 | 0.008 | 0.010 |
| 6 | 13.4 |  | 1.30E+04 | 0.002 | 0.003 | 0.003 | 0.003 | 0.004 | 0.004 | 0.004 | 0.005 | 0.005 | 0.006 | 0.006 | 0.006 | 0.008 | 0.010 | 0.012 | 0.014 |
|  | 15.7 |  | $1.51 \mathrm{E}+04$ | 0.003 | 0.004 | 0.004 | 0.005 | 0.005 | 0.005 | 0.006 | 0.007 | 0.007 | 0.008 | 0.008 | 0.009 | 0.011 | 0.014 | 0.016 | 0.019 |
| 8 | 17.9 |  | $1.73 E+04$ | 0.004 | 0.005 | 0.006 | 0.006 | 0.006 | 0.007 | 0.008 | 0.009 | 0.009 | 0.010 | 0.011 | 0.011 | 0.014 | 0.018 | 0.021 | 0.025 |
| 9 | 20.1 |  | $1.95 \mathrm{E}+04$ | 0.005 | 0.006 | 0.007 | 0.008 | 0.008 | 0.009 | 0.010 | 0.011 | 0.012 | 0.013 | 0.014 | 0.014 | 0.018 | 0.023 | 0.027 | 0.032 |
| 10 | 22.4 |  | $2.16 \mathrm{E}+04$ | 0.006 | 0.008 | 0.009 | 0.009 | 0.010 | 0.011 | 0.012 | 0.013 | 0.015 | 0.016 | 0.017 | 0.018 | 0.022 | 0.028 | 0.033 | 0.039 |
| 11 | 24.6 |  | $2.38 \mathrm{E}+04$ | 0.007 | 0.009 | 0.011 | 0.011 | 0.012 | 0.013 | 0.015 | 0.016 | 0.018 | 0.019 | 0.020 | 0.022 | 0.027 | 0.034 | 0.040 | 0.047 |
| 12 | 26.8 |  | $2.60 \mathrm{E}+04$ | 0.008 | 0.011 | 0.013 | 0.014 | 0.014 | 0.016 | 0.018 | 0.019 | 0.021 | 0.022 | 0.024 | 0.026 | 0.032 | 0.040 | 0.048 | 0.056 |
| 13 | 29.1 | 15 | $2.81 \mathrm{E}+04$ | 0.009 | 0.013 | 0.015 | 0.016 | 0.017 | 0.019 | 0.021 | 0.023 | 0.025 | 0.026 | 0.028 | 0.030 | 0.038 | 0.047 | 0.057 | 0.066 |
| 14 | 31.3 |  | $3.03 E+04$ | 0.011 | 0.015 | 0.017 | 0.019 | 0.020 | 0.022 | 0.024 | 0.026 | 0.028 | 0.031 | 0.033 | 0.035 | 0.044 | 0.055 | 0.066 | 0.077 |
| 15 | 33.6 |  | 3.25E+04 | 0.013 | 0.018 | 0.020 | 0.021 | 0.023 | 0.025 | 0.028 | 0.030 | 0.033 | 0.035 | 0.038 | 0.040 | 0.050 | 0.063 | 0.075 | 0.088 |
| 16 | 35.8 |  | 3.46E+04 | 0.014 | 0.020 | 0.023 | 0.024 | 0.026 | 0.029 | 0.031 | 0.034 | 0.037 | 0.040 | 0.043 | 0.046 | 0.057 | 0.071 | 0.086 | 0.100 |
| 17 | 38.0 | 20 | $3.68 \mathrm{E}+04$ | 0.016 | 0.023 | 0.026 | 0.027 | 0.029 | 0.032 | 0.035 | 0.039 | 0.042 | 0.045 | 0.048 | 0.052 | 0.064 | 0.081 | 0.097 | 0.113 |
| 18 | 40.3 |  | 3.89E+04 | 0.018 | 0.025 | 0.029 | 0.031 | 0.033 | 0.036 | 0.040 | 0.043 | 0.047 | 0.051 | 0.054 | 0.058 | 0.072 | 0.090 | 0.108 | 0.127 |
| 19 | 42.5 |  | $4.11 \mathrm{E}+04$ | 0.020 | 0.028 | 0.032 | 0.034 | 0.036 | 0.040 | 0.044 | 0.048 | 0.052 | 0.056 | 0.060 | 0.064 | 0.081 | 0.101 | 0.121 | 0.141 |
| 20 | 44.7 |  | $4.33 \mathrm{E}+04$ | 0.022 | 0.031 | 0.036 | 0.038 | 0.040 | 0.045 | 0.049 | 0.054 | 0.058 | 0.062 | 0.067 | 0.071 | 0.089 | 0.112 | 0.134 | 0.156 |
| 21 | 47.0 |  | $4.54 \mathrm{E}+04$ | 0.025 | 0.034 | 0.039 | 0.042 | 0.044 | 0.049 | 0.054 | 0.059 | 0.064 | 0.069 | 0.074 | 0.079 | 0.098 | 0.123 | 0.148 | 0.172 |
| 22 | 49.2 | 25 | $4.76 \mathrm{E}+04$ | 0.027 | 0.038 | 0.043 | 0.046 | 0.049 | 0.054 | 0.059 | 0.065 | 0.070 | 0.076 | 0.081 | 0.086 | 0.108 | 0.135 | 0.162 | 0.189 |
| 23 | 51.4 |  | $4.98 \mathrm{E}+04$ | 0.030 | 0.041 | 0.047 | 0.050 | 0.053 | 0.059 | 0.065 | 0.071 | 0.077 | 0.083 | 0.089 | 0.094 | 0.118 | 0.148 | 0.177 | 0.207 |
| 24 | 53.7 |  | $5.19 \mathrm{E}+04$ | 0.032 | 0.045 | 0.051 | 0.055 | 0.058 | 0.064 | 0.071 | 0.077 | 0.084 | 0.090 | 0.096 | 0.103 | 0.129 | 0.161 | 0.193 | 0.225 |
| 25 | 55.9 |  | $5.41 \mathrm{E}+04$ | 0.035 | 0.049 | 0.056 | 0.059 | 0.063 | 0.070 | 0.077 | 0.084 | 0.091 | 0.098 | 0.105 | 0.112 | 0.139 | 0.174 | 0.209 | 0.244 |
| 26 | 58.2 | 30 | $5.62 \mathrm{E}+04$ | 0.038 | 0.053 | 0.060 | 0.064 | 0.068 | 0.075 | 0.083 | 0.090 | 0.098 | 0.106 | 0.113 | 0.121 | 0.151 | 0.189 | 0.226 | 0.264 |
| 27 | 60.4 |  | $5.84 \mathrm{E}+04$ | 0.041 | 0.057 | 0.065 | 0.069 | 0.073 | 0.081 | 0.089 | 0.098 | 0.106 | 0.114 | 0.122 | 0.130 | 0.163 | 0.203 | 0.244 | 0.285 |
| 28 | 62.6 |  | $6.06 \mathrm{E}+04$ | 0.044 | 0.061 | 0.070 | 0.074 | 0.079 | 0.087 | 0.096 | 0.105 | 0.114 | 0.122 | 0.131 | 0.140 | 0.175 | 0.219 | 0.262 | 0.306 |
| 29 | 64.9 |  | 6.27E+04 | 0.047 | 0.066 | 0.075 | 0.080 | 0.084 | 0.094 | 0.103 | 0.113 | 0.122 | 0.131 | 0.141 | 0.150 | 0.188 | 0.235 | 0.281 | 0.328 |
| 30 | 67.1 |  | $6.49 \mathrm{E}+04$ | 0.050 | 0.070 | 0.080 | 0.085 | 0.090 | 0.100 | 0.110 | 0.120 | 0.131 | 0.141 | 0.151 | 0.161 | 0.201 | 0.251 | 0.301 | 0.351 |
| 31 | 69.3 | 35 | $6.71 \mathrm{E}+04$ | 0.054 | 0.075 | 0.086 | 0.091 | 0.096 | 0.107 | 0.118 | 0.129 | 0.139 | 0.150 | 0.161 | 0.172 | 0.214 | 0.268 | 0.322 | 0.375 |
| 32 | 71.6 |  | $6.92 \mathrm{E}+04$ | 0.057 | 0.080 | 0.091 | 0.097 | 0.103 | 0.114 | 0.126 | 0.137 | 0.149 | 0.160 | 0.171 | 0.183 | 0.228 | 0.286 | 0.343 | 0.400 |
| 33 | 73.8 |  | $7.14 \mathrm{E}+04$ | 0.061 | 0.085 | 0.097 | 0.103 | 0.109 | 0.121 | 0.134 | 0.146 | 0.158 | 0.170 | 0.182 | 0.194 | 0.243 | 0.304 | 0.364 | 0.425 |
| 34 | 76.1 |  | $7.36 \mathrm{E}+04$ | 0.064 | 0.090 | 0.103 | 0.110 | 0.116 | 0.129 | 0.142 | 0.155 | 0.168 | 0.181 | 0.193 | 0.206 | 0.258 | 0.322 | 0.387 | 0.451 |
| 35 | 78.3 | 40 | $7.57 \mathrm{E}+04$ | 0.068 | 0.096 | 0.109 | 0.116 | 0.123 | 0.137 | 0.150 | 0.164 | 0.178 | 0.191 | 0.205 | 0.219 | 0.273 | 0.342 | 0.410 | 0.478 |
| 36 | 80.5 |  | $7.79 \mathrm{E}+04$ | 0.072 | 0.101 | 0.116 | 0.123 | 0.130 | 0.145 | 0.159 | 0.173 | 0.188 | 0.202 | 0.217 | 0.231 | 0.289 | 0.361 | 0.434 | 0.506 |
| 37 | 82.8 |  | $8.00 \mathrm{E}+04$ | 0.076 | 0.107 | 0.122 | 0.130 | 0.137 | 0.153 | 0.168 | 0.183 | 0.199 | 0.214 | 0.229 | 0.244 | 0.305 | 0.382 | 0.458 | 0.535 |
| 38 | 85.0 |  | $8.22 \mathrm{E}+04$ | 0.081 | 0.113 | 0.129 | 0.137 | 0.145 | 0.161 | 0.177 | 0.193 | 0.209 | 0.226 | 0.242 | 0.258 | 0.322 | 0.403 | 0.483 | 0.564 |
| 39 | 87.2 |  | $8.44 \mathrm{E}+04$ | 0.085 | 0.119 | 0.136 | 0.144 | 0.153 | 0.170 | 0.187 | 0.204 | 0.221 | 0.238 | 0.255 | 0.271 | 0.339 | 0.424 | 0.509 | 0.594 |
| 40 | 89.5 | 45 | $8.65 E+04$ | 0.089 | 0.125 | 0.143 | 0.152 | 0.161 | 0.178 | 0.196 | 0.214 | 0.232 | 0.250 | 0.268 | 0.286 | 0.357 | 0.446 | 0.535 | 0.625 |
| 41 | 91.7 |  | $8.87 \mathrm{E}+04$ | 0.094 | 0.131 | 0.150 | 0.159 | 0.169 |  | 0.206 | 0.225 | 0.244 | 0.263 | 0.281 | 0.300 | 0.375 | 0.469 | 0.563 | 0.656 |
| 42 | 94.0 |  | $9.09 \mathrm{E}+04$ | 0.098 | 0.138 | 0.157 | 0.167 | 0.177 | 0.197 | 0.216 | 0.236 | 0.256 | 0.275 | 0.295 | 0.315 | 0.394 | 0.492 | 0.590 | 0.689 |
| 43 | 96.2 |  | $9.30 \mathrm{E}+04$ | 0.103 | 0.144 | 0.165 | 0.175 | 0.186 | 0.206 | 0.227 | 0.248 | 0.268 | 0.289 | 0.309 | 0.330 | 0.413 | 0.516 | 0.619 | 0.722 |
| 44 | 98.4 | 50 | $9.52 \mathrm{E}+04$ | 0.108 | 0.151 | 0.173 | 0.184 | 0.194 | 0.216 | 0.238 | 0.259 | 0.281 | 0.302 | 0.324 | 0.346 | 0.432 | 0.540 | 0.648 | 0.756 |
| 45 | 100.7 |  | $9.74 \mathrm{E}+04$ | 0.113 | 0.158 | 0.181 | 0.192 | 0.203 | 0.226 | 0.248 | 0.271 | 0.294 | 0.316 | 0.339 | 0.361 | 0.452 | 0.565 | 0.678 | 0.791 |
| 46 | 102.9 |  | $9.95 \mathrm{E}+04$ | 0.118 | 0.165 | 0.189 | 0.201 | 0.212 | 0.236 | 0.260 | 0.283 | 0.307 | 0.330 | 0.354 | 0.378 | 0.472 | 0.590 | 0.708 | 0.826 |
| 47 | 105.1 |  | $1.02 \mathrm{E}+05$ | 0.123 | 0.172 | 0.197 | 0.209 | 0.222 | 0.246 | 0.271 | 0.296 | 0.320 | 0.345 | 0.370 | 0.394 | 0.493 | 0.616 | 0.739 | 0.862 |
| 48 | 107.4 |  | $1.04 \mathrm{E}+05$ | 0.129 | 0.180 | 0.206 | 0.218 | 0.231 | 0.257 | 0.283 | 0.308 | 0.334 | 0.360 | 0.386 | 0.411 | 0.514 | 0.643 | 0.771 | 0.900 |
| 49 | 109.6 | 55 | $1.06 E+05$ | 0.134 | 0.187 | 0.214 | 0.228 | 0.241 | 0.268 | 0.295 | 0.321 | 0.348 | ${ }_{0}^{0.375}$ | 0.402 | 0.429 | 0.536 | 0.670 | 0.804 | 0.937 |
| 50 | 111.8 |  | $1.08 \mathrm{E}+05$ | 0.139 | 0.195 | 0.223 | 0.237 | 0.251 | 0.279 | 0.307 | 0.335 | 0.363 | 0.390 | 0.418 | 0.446 | 0.558 | 0.697 | 0.837 | 0.976 |
| 51 | 114.1 |  | ${ }^{1.10 E+05}$ | 0.145 | 0.203 | 0.232 | 0.247 | 0.261 | 0.290 | 0.319 | 0.348 | 0.377 | 0.406 | 0.435 | 0.464 | 0.580 | 0.725 | 0.870 | 1.016 |
| 52 | 116.3 |  | ${ }^{1.122}+05$ | 0.151 | 0.211 | 0.241 | 0.256 | 0.271 | 0.302 | 0.332 | 0.362 | 0.392 | 0.422 | 0.452 | 0.483 | ${ }^{0.603}$ | 0.754 | 0.905 | ${ }_{1}^{1.056}$ |
| 53 | 118.6 | 60 | $1.15 \mathrm{E}+05$ | 0.157 | 0.219 | 0.251 | 0.266 | 0.282 | 0.313 | 0.345 | 0.376 | 0.407 | 0.439 | 0.470 | 0.501 | 0.627 | 0.783 | 0.940 | 1.097 |
| 54 | 120.8 |  | 1.17E+05 | 0.163 | 0.228 | 0.260 | 0.276 | 0.293 | 0.325 | 0.358 | 0.390 | 0.423 | 0.455 | 0.488 | 0.520 | 0.651 | 0.813 | 0.976 | 1.139 |



BLOCKAGE LESS THAN $10^{\circ}$


BLOCKAGE LESS THAN $10 \%$

bLOCKAGE LESS than $10 \%$

BLOCKAGE LESS THAN $10 \%$

| Diameter Dprojected $(\mathrm{m})$ | $\mathbf{0 . 2 2 3 5 7}$ |
| :--- | ---: |
| Area Projected $\left.\mathrm{m}^{2}\right)$ | $\mathbf{0 . 0 3 9 2 6}$ |
| Density $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ | $\mathbf{1 . 2}$ |
| Mu $\left(\mathrm{N}\right.$ - $\left./ \mathrm{m}^{2}\right)$ | $\mathbf{1 . 8 0 E - 0 5}$ |
| Kinematic Viscosity | $1.50 \mathrm{E}-05$ |


| Kinematic Viscosity |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | 0.5 | 0.7 | 0.8 | 0.85 | 0.9 | 1 | 1.1 | 1.2 | 1.3 | 1.4 | 1.5 | 1.6 | 2 | 2.5 | 3 | 3.5 |
| $\frac{\text { Tunnel Speed }}{\mathrm{m} / \mathrm{s}}$ |  |  |  | Force |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| m/s | MPH | Hz | RE\# | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs |
| 1 | 0.0 |  | ${ }^{0}$ | 0.000 | $\frac{0.000}{0.004}$ | $\stackrel{0.000}{0.004}$ | $\stackrel{0.000}{0.005}$ | $\frac{0.000}{0.005}$ | $\stackrel{0.000}{0.005}$ | ${ }_{0}^{0.000}$ | ${ }_{0}^{0.0000}$ | ${ }_{0}^{0.000}$ | 0.000 0.007 | $\stackrel{0}{0.000}$ | $\stackrel{0.008}{0.000}$ | $\frac{0.000}{0.011}$ | ${ }_{0}^{0.000}$ | 0.000 | 0.000 |
| 2 | 4.5 | 2 | $2.98 \mathrm{E}+04$ | 0.011 | 0.015 | 0.017 | 0.018 | 0.019 | 0.021 | 0.023 | 0.025 | 0.028 | 0.030 | 0.032 | 0.034 | 0.042 | 0.053 | 0.064 | 0.074 |
| 3 | 6.7 | 2 | $4.47 \mathrm{E}+04$ | 0.024 | 0.033 | 0.038 | 0.041 | 0.043 | 0.048 | 0.052 | 0.057 | 0.062 | 0.067 | 0.071 | 0.076 | 0.095 | 0.119 | 0.143 | 0.167 |
| 4 | 8.9 | 4 | $5.96 \mathrm{E}+04$ | 0.042 | 0.059 | 0.068 | 0.072 | 0.076 | 0.085 | 0.093 | 0.102 | 0.110 | 0.119 | 0.127 | 0.136 | 0.169 | 0.212 | 0.254 | 0.297 |
| 5 | 11.2 |  | $7.45 \mathrm{E}+04$ | 0.066 | 0.093 | 0.106 | 0.113 | 0.119 | 0.132 | 0.146 | 0.159 | 0.172 | 0.185 | 0.199 | 0.212 | 0.265 | 0.331 | 0.397 | 0.463 |
|  | 13.4 |  | $8.94 \mathrm{E}+04$ | 0.095 | 0.133 | 0.153 | 0.162 | 0.172 | 0.191 | 0.210 | 0.229 | 0.248 | 0.267 | 0.286 | 0.305 | 0.381 | 0.477 | 0.572 | 0.667 |
| 7 | 15.7 |  | $1.04 \mathrm{E}+05$ | 0.130 | 0.182 | 0.208 | 0.221 | 0.234 | 0.259 | 0.285 | 0.311 | 0.337 | 0.363 | 0.389 | 0.415 | 0.519 | 0.649 | 0.778 | 0.908 |
| 8 | 17.9 |  | 1.19E+05 | 0.169 | 0.237 | 0.271 | 0.288 | 0.305 | 0.339 | ${ }^{0.373}$ | 0.407 | 0.441 | 0.474 | 0.508 | 0.542 | 0.678 | 0.847 | 1.017 | ${ }^{1.186}$ |
| 9 | 20.1 |  | $1.34 \mathrm{E}+05$ | 0.214 | 0.300 | 0.343 | 0.365 | ${ }^{0.386}$ | 0.429 | ${ }^{0.472}$ | 0.515 | 0.558 | 0.600 | 0.643 | 0.686 | 0.858 | 1.072 | 1.287 | 1.501 |
| 10 | 22.4 |  | $1.49 \mathrm{E}+05$ | 0.265 | 0.371 | 0.424 | 0.450 | 0.477 | 0.530 | 0.582 | 0.635 | 0.688 | 0.741 | 0.794 | 0.847 | 1.059 | 1.324 | 1.589 | 1.853 |
| 11 | 24.6 |  | $1.64 \mathrm{E}+05$ | 0.320 | 0.449 | 0.513 | 0.545 | 0.577 | 0.641 | 0.705 | 0.769 | 0.833 | 0.897 | 0.961 | 1.025 | 1.281 | 1.602 | 1.922 | 2.243 |
| 12 | 26.8 |  | 1.79E+05 | 0.381 | 0.534 | 0.610 | 0.648 | 0.686 | 0.763 | 0.839 | 0.915 | 0.991 | 1.068 | $\underline{1.144}$ | 1.220 | 1.525 | 1.906 | 2.288 | 2.669 |
| 13 | 29.1 | 15 | $1.94 \mathrm{E}+05$ | 0.447 | 0.626 | 0.716 | 0.761 | 0.805 | 0.895 | 0.984 | 1.074 | 1.163 | 1.253 | 1.342 | 1.432 | 1.790 | 2.237 | 2.685 | 3.132 |
| 14 | 31.3 |  | $2.09 E+05$ | 0.519 | 0.727 | 0.830 | 0.882 | 0.934 | 1.038 | ${ }^{1.142}$ | 1.245 | 1.349 | 1.453 | 1.557 | 1.661 | 2.076 | 2.595 | 3.114 | 3.633 |
| 15 | 33.6 |  | $2.24 \mathrm{E}+05$ | 0.596 | 0.834 | 0.953 | 1.013 | 1.072 | 1.191 | 1.311 | 1.430 | 1.549 | 1.668 | 1.787 | 1.906 | 2.383 | 2.979 | 3.574 | 4.170 |
| 16 | 35.8 |  | $2.38 \mathrm{E}+05$ | 0.678 | 0.949 | 1.084 | 1.152 | 1.220 | 1.356 | 1.491 | 1.627 | 1.762 | 1.898 | 2.033 | 2.169 | 2.711 | 3.389 | 4.067 | 4.744 |
| 17 | 38.0 | 20 | $2.53 \mathrm{E}+05$ | 0.765 | 1.071 | 1.224 | 1.301 | 1.377 | 1.530 | 1.683 | 1.836 | 1.989 | 2.142 | 2.295 | 2.448 | 3.061 | 3.826 | 4.591 | 5.356 |
| 18 | 40.3 |  | $2.68 \mathrm{E}+05$ | 0.858 | 1.201 | 1.373 | 1.458 | 1.544 | 1.716 | 1.887 | 2.059 | 2.230 | 2.402 | 2.573 | 2.745 | 3.431 | 4.289 | 5.147 | 6.005 |
| 19 | 42.5 |  | $2.83 \mathrm{E}+05$ | 0.956 | 1.338 | 1.529 | 1.625 | 1.720 | 1.912 | 2.103 | 2.294 | 2.485 | 2.676 | ${ }^{2.867}$ | ${ }^{3.059}$ | ${ }^{3.823}$ | 4.779 | 5.735 | 6.690 |
| 20 | 44.7 |  | $2.98 \mathrm{E}+05$ | 1.059 | 1.483 | 1.694 | 1.800 | 1.906 | 2.118 | 2.330 | 2.542 | 2.754 | 2.965 | 3.177 | 3.389 | 4.236 | 5.295 | 6.354 | 7.413 |
| 21 | 47.0 |  | $3.13 E+05$ | 1.168 | 1.635 | 1.868 | 1.985 | 2.102 | 2.335 | 2.569 | 2.802 | 3.036 | 3.269 | 3.503 | 3.736 | 4.670 | 5.838 | 7.006 | 8.173 |
| 22 | 49.2 | 25 | $3.28 \mathrm{E}+05$ | 1.281 | 1.794 | 2.050 | 2.178 | 2.307 | 2.563 | 2.819 | 3.075 | 3.332 | 3.588 | 3.844 | 4.101 | 5.126 | 6.407 | 7.689 | 8.970 |
| ${ }^{23}$ | 51.4 |  | $3.43 \mathrm{E}+05$ | 1.401 | 1.961 | 2.241 | 2.381 | 2.521 | 2.801 | ${ }^{3.081}$ | ${ }^{3.361}$ | ${ }^{3.642}$ | 3.922 | 4.202 | 4.482 | 5.602 | 7.003 | 8.403 | ${ }^{9.804}$ |
| 24 | 53.7 |  | $3.58 \mathrm{E}+05$ | 1.525 | 2.135 | 2.440 | 2.593 | 2.745 | 3.050 | 3.355 | 3.660 | 3.965 | 4.270 | 4.575 | 4.880 | 6.100 | 7.625 | 9.150 | 10.675 |
| 25 | 55.9 |  | $3.73 \mathrm{E}+05$ | 1.655 | 2.317 | 2.648 | 2.813 | 2.979 | 3.309 | 3.640 | 3.971 | 4.302 | 4.633 | 4.964 | 5.295 | 6.619 | 8.274 | 9.928 | 11.583 |
| 26 | 58.2 | 30 | $3.88 \mathrm{E}+05$ | 1.790 | 2.506 | 2.864 | ${ }^{3.043}$ | 3.222 | 3.580 | 3.938 | 4.295 | 4.653 | 5.011 | 5.369 | 5.727 | 7.159 | 8.949 | 10.739 | 12.528 |
| 27 | 60.4 |  | $4.02 \mathrm{E}+05$ | 1.930 | 2.702 | 3.088 | 3.281 | 3.474 | 3.860 | ${ }^{4.246}$ | 4.632 | 5.018 | 5.404 | 5.790 | 6.176 | 7.720 | 9.650 | 11.581 | 13.511 |
| 28 | 62.6 |  | $4.17 \mathrm{E}+05$ | 2.076 | 2.906 | 3.321 | 3.529 | 3.736 | 4.151 | 4.567 | 4.982 | 5.397 | 5.812 | 6.227 | 6.642 | 8.303 | 10.379 | 12.454 | 14.530 |
| 29 | 64.9 |  | $4.32 \mathrm{E}+05$ | 2.227 | 3.117 | ${ }^{3.563}$ | 3.785 | 4.008 | 4.453 | 4.899 | 5.344 | 5.789 | 6.235 | 6.880 | 7.125 | 8.907 | 11.133 | 13.360 | 15.586 |
| 30 | 67.1 |  | $4.47 \mathrm{E}+05$ | 2.383 | ${ }^{3.336}$ | 3.813 | 4.051 | 4.289 | 4.766 | ${ }^{5.242}$ | 5.719 | 6.195 | 6.672 | 7.149 | 7.625 | ${ }^{9.531}$ | 11.914 | 14.297 | 16.680 |
| 31 | 69.3 | 35 | $4.62 \mathrm{E}+05$ | 2.544 | 3.562 | 4.071 | 4.325 | 4.580 | 5.089 | 5.598 | 6.106 | 6.615 | 7.124 | 7.633 | 8.142 | 10.177 | 12.722 | 15.266 | 17.810 |
| 32 | 71.6 |  | $4.77 \mathrm{E}+05$ | 2.711 | 3.796 | 4.338 | 4.609 | 4.880 | 5.422 | 5.965 | 6.507 | 7.049 | 7.591 | 8.133 | 8.676 | 10.845 | 13.556 | 16.267 | 18.978 |
| 33 | 73.8 |  | $4.92 \mathrm{E}+05$ | 2.883 | 4.037 | 4.613 | 4.901 | 5.190 | 5.766 | 6.343 | 6.920 | 7.496 | 8.073 | 8.650 | 9.226 | 11.533 | 14.416 | 17.299 | 20.183 |
| 34 | 76.1 |  | $5.07 \mathrm{E}+05$ | 3.061 | 4.285 | 4.897 | 5.203 | 5.509 | 6.121 | ${ }^{6.733}$ | 7.345 | 7.958 | 8.570 | 9.182 | 9.794 | 12.242 | 15.303 | 18.364 | 21.424 |
| 35 | 78.3 | 40 | $5.22 \mathrm{E}+05$ | 3.243 | 4.541 | 5.189 | 5.514 | 5.838 | 6.487 | ${ }^{6} .135$ | 7.784 | 8.433 | 9.081 | 9.730 | 10.379 | 12.973 | 16.217 | 19.460 | 22.703 |
| 36 | 80.5 |  | $5.37 \mathrm{E}+05$ | 3.431 | 4.804 | 5.490 | 5.833 | 6.176 | 6.863 | 7.549 | 8.235 | 8.921 | 9.608 | 10.294 | 10.980 | 13.725 | 17.156 | 20.588 | 24.019 |
| 37 | 82.8 |  | $5.51 \mathrm{E}+05$ | 3.625 | 5.074 | 5.799 | 6.162 | ${ }^{6.524}$ | 7.249 | ${ }^{7.974}$ | 8.699 | 9.424 | 10.149 | 10.874 | 11.599 | ${ }^{14.498}$ | $\frac{18.123}{}$ | 21.747 | 25.372 |
| 38 | 85.0 |  | $5.66 \mathrm{E}+05$ | 3.823 | 5.352 | $\underline{6.117}$ | 6.499 | 6.882 | ${ }^{7.646}$ | ${ }^{8.411}$ | $\underline{9.176}$ | 9.940 | 10.705 | 11.469 | 12.234 | ${ }^{15.293}$ | ${ }^{19.116}$ | 22.939 | 26.762 |
|  | 87.2 |  | $5.81 \mathrm{E}+05$ | 4.027 | 5.638 | 6.443 | 6.846 | 7.249 | 8.054 | 8.859 | 9.665 | 10.470 | 11.276 | 12.081 | 12.886 | 16.108 | 20.135 | 24.162 | 28.189 |
| 40 | 89.5 | 45 | $5.96 E+05$ | 4.236 | 5.931 | 6.778 | 7.201 | 7.625 | 8.472 | 9.320 | 10.167 | 11.014 | 11.861 | 12.708 | 13.556 | 16.945 | 21.181 | 25.417 | 29.653 |
| 41 | 91.7 |  | $6.11 \mathrm{E}+05$ | 4.451 | 6.231 | 7.121 | ${ }^{7.566}$ | 8.011 | 8.901 | 9.791 | 10.681 | 11.572 | 12.462 | ${ }^{13.352}$ | 14.242 | 17.802 | ${ }^{22.253}$ | 26.704 | 31.154 |
| 42 | 94.0 |  | $6.26 E+05$ | 4.670 | 6.539 | ${ }^{7.473}$ | 7.940 | 8.407 | 9.341 | ${ }^{10.275}$ | 11.209 | 12.143 | ${ }^{13.077}$ | ${ }^{14.011}$ | 14.945 | 18.681 | ${ }^{23.352}$ | 28.022 | 32.693 |
| 43 | 96.2 |  | $6.41 \mathrm{E}+05$ | 4.895 | 6.854 | 7.833 | 8.322 | 8.812 | 9.791 | 10.770 | 11.749 | 12.728 | 13.707 | 14.686 | 15.665 | 19.582 | 24.477 | 29.372 | 34.268 |
| 44 | 98.4 | 50 | $6.56 \mathrm{E}+05$ | 5.126 | 7.176 | 8.201 | 8.714 | 9.226 | 10.251 | 11.277 | 12.302 | 13.327 | 14.352 | 15.377 | 16.402 | 20.503 | 25.629 | 30.754 | 35.880 |
| 45 | 100.7 |  | $6.71 \mathrm{E}+05$ | 5.361 | 7.506 | 8.578 | 9.114 | 9.650 | 10.723 | 11.795 | 12.867 | 13.940 | 15.012 | 16.084 | 17.156 | 21.446 | 26.807 | 32.168 | 37.530 |
| 46 | 102.9 |  | $6.86 \mathrm{E}+05$ | 5.602 | 7.843 | 8.964 | 9.524 | 10.084 | 11.205 | ${ }^{12.325}$ | ${ }^{13.446}$ | 14.566 | 15.686 | 16.807 | 17.927 | 22.409 | 28.012 | 33.614 | 39.216 |
| 47 | 105.1 |  | $7.01 \mathrm{E}+05$ | 5.849 | 8.188 | 9.358 | 9.943 | 10.527 | 11.697 | 12.867 | 14.036 | 15.206 | 16.376 | 17.546 | 18.715 | 23.394 | 29.243 | 35.091 | 40.940 |
| 48 | 107.4 |  | $7.15 \mathrm{E}+05$ | 6.100 | 8.540 | 9.760 | ${ }^{10.370}$ | 10.980 | 12.200 | 13.420 | 14.640 | 15.860 | 17.080 | 18.300 | 19.520 | 24.400 | 30.500 | 36.600 | 42.700 |
| 49 | 109.6 | 55 | $7.30 \mathrm{E}+05$ | 6.357 | 8.900 | 10.171 | 10.807 | 11.442 | 12.714 | ${ }^{13.985}$ | ${ }^{15.257}$ | 16.528 | 17.799 | 19.071 | 20.342 | 25.428 | 31.784 | 38.141 | 44.498 |
| 50 | 111.8 |  | $7.45 \mathrm{E}+05$ | 6.619 | 9.267 | 10.590 | 11.252 | 11.914 | ${ }^{13.238}$ | 14.562 | 15.886 | 17.209 | 18.533 | 19.857 | 21.181 | 26.476 | 33.095 | 39.714 | 46.333 |
| 51 | 114.1 |  | $7.60 \mathrm{E}+05$ | 6.886 | 9.641 | 11.018 | 11.707 | 12.396 | ${ }^{13.773}$ | ${ }^{15.150}$ | 16.527 | 17.905 | 19.282 | 20.659 | 22.036 | 27.546 | 34.432 | 41.318 | 48.205 |
| 52 | 116.3 |  | $7.75 \mathrm{E}+05$ | 7.159 | 10.023 | 11.455 | 12.170 | 12.886 | 14.318 | 15.750 | 17.182 | 18.614 | 20.045 | 21.477 | 22.909 | 28.636 | 35.796 | 42.955 | 50.114 |
| 53 | 118.6 | 60 | $7.90 \mathrm{E}+05$ | 7.437 | 10.412 | 11.899 | 12.643 | 13.387 | 14.874 | 16.362 | 17.849 | 19.336 | 20.824 | 22.311 | 23.799 | 29.748 | 37.185 | 44.623 | 52.060 |
| 54 | 120.8 |  | 8.05E+05 | 7.720 | 10.809 | 12.353 | 13.125 | 13.897 | 15.441 | 16.985 | 18.529 | 20.073 | 21.617 | 23.161 | 24.705 | 30.882 | 38.602 | 46.322 | 54.043 |

BLOCKAGE LESS THAN $10 \%$

| Diameter Dprojected $(\mathrm{m})$ | $\mathbf{0 . 2 3 3 0 1}$ |
| :--- | ---: |
| Area Projected $\left(\mathrm{m}^{2}\right)$ | $\mathbf{0 . 0 4 2 6 4}$ |
| Density $\left.\mathrm{kg} / \mathrm{m}^{3}\right)$ | 1.2 |
| Mu $\left(\mathrm{N}\right.$-s $\left./ \mathrm{m}^{2}\right)$ | $\mathbf{1 . 8 0 E - 0 5}$ |
| Kinematic Viscosity | $1.50 \mathrm{E}-05$ |


| .50E-05 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | 0.5 | 0.7 | 0.8 | 0.85 | 0.9 | 1 | 1.1 | 1.2 | 1.3 | 1.4 | 1.5 | 1.6 | 2 | 2.5 | 3 | 3.5 |
| Tunnel Speed |  |  |  | Force |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| m/s | MPH | Hz |  | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs |
| 1 | 0.0 |  | ${ }_{\text {0.00E+00 }}^{155 E+04}$ | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 |
| 2 | 4.5 | 2 | ${ }^{\text {a }}$ 3.11E $\mathrm{E}+04$ | 0.012 | 0.0016 | 0.0018 | 0.020 | 0.021 | 0.023 | 0.025 | 0.0028 | 0 | 0.032 | 0.035 | 0.037 | 0.046 | 0.058 | 0.069 | 0.081 |
| 3 | 6.7 | 2 | $4.66 \mathrm{E}+04$ | 0.026 | 0.036 | 0.041 | 0.044 | 0.047 | 0.052 | 0.057 | 0.062 | 0.067 | 0.072 | 0.078 | 0.083 | 0.104 | 0.129 | 0.155 | 0.181 |
| 4 | 8.9 | 4 | 6.21E+04 | 0.046 | 0.064 | 0.074 | 0.078 | 0.083 | 0.092 | 0.101 | 0.110 | 0.120 | 0.129 | 0.138 | 0.147 | 0.184 | 0.230 | 0.276 | 0.322 |
| 5 | 11.2 |  | 7.77E+04 | 0.072 | 0.101 | 0.115 | 0.122 | 0.129 | 0.144 | 0.158 | 0.173 | 0.187 | 0.201 | 0.216 | 0.230 | 0.288 | 0.359 | 0.431 | 0.503 |
| 6 | 13.4 |  | $9.32 \mathrm{E}+04$ | 0.104 | 0.145 | 0.166 | 0.176 | 0.186 | 0.207 | 0.228 | 0.248 | 0.269 | 0.290 | 0.311 | 0.331 | 0.414 | 0.518 | 0.621 | 0.725 |
| 7 | 15.7 |  | $1.09 \mathrm{E}+05$ | 0.141 | 0.197 | 0.225 | 0.240 | 0.254 | 0.282 | 0.310 | 0.338 | 0.366 | 0.395 | 0.423 | 0.451 | 0.564 | 0.705 | 0.846 | 0.986 |
| 8 | 17.9 |  | 1.24E+05 | 0.184 | 0.258 | 0.294 | ${ }^{0.313}$ | 0.331 | 0.368 | 0.405 | 0.442 | 0.479 | 0.515 | 0.552 | 0.589 | 0.736 | 0.920 | 1.104 | 1.288 |
| 9 | 20.1 |  | 1.40E+05 | 0.233 | 0.326 | 0.373 | 0.396 | 0.419 | 0.466 | 0.512 | 0.559 | 0.606 | 0.652 | 0.699 | 0.745 | 0.932 | 1.165 | 1.398 | 1.631 |
| 10 | 22.4 |  | $1.55 \mathrm{E}+05$ | 0.288 | 0.403 | 0.460 | 0.489 | 0.518 | 0.575 | 0.633 | 0.690 | 0.748 | 0.805 | 0.863 | 0.920 | 1.150 | 1.438 | 1.726 | 2.013 |
| 11 | 24.6 |  | ${ }^{1.71 E+05}$ | 0.348 | 0.487 | 0.557 | 0.592 | 0.626 | 0.696 | 0.766 | 0.835 | 0.905 | 0.974 | 1.044 | 1.114 | $\frac{1.392}{107}$ | 1.740 | 2.088 | 2.436 |
| 12 | 26.8 |  | $1.86 \mathrm{E}+05$ | 0.414 | 0.580 | 0.663 | 0.704 | 0.745 | 0.828 | 0.911 | 0.994 | 1.077 | 1.160 | 1.242 | 1.325 | 1.657 | 2.071 | 2.485 | 2.899 |
| 13 | 29.1 | 15 | $2.02 \mathrm{E}+05$ | 0.486 | 0.680 | 0.778 | 0.826 | 0.875 | 0.972 | 1.069 | ${ }^{1.166}$ | 1.264 | 1.361 | 1.458 | 1.555 | 1.944 | 2.430 | 2.916 | 3.402 |
| 14 | 31.3 |  | $2.17 \mathrm{E}+05$ | 0.564 | 0.789 | 0.902 | 0.958 | 1.015 | 1.127 | 1.240 | 1.353 | 1.466 | 1.578 | 1.691 | 1.804 | 2.255 | 2.818 | 3.382 | 3.946 |
| 15 | 33.6 |  | $2.33 \mathrm{E}+05$ | 0.647 | 0.906 | 1.035 | 1.100 | 1.165 | 1.294 | 1.424 | 1.553 | 1.682 | 1.812 | 1.941 | 2.071 | 2.588 | 3.235 | 3.882 | 4.530 |
| 16 | 35.8 |  | $2.49 \mathrm{E}+05$ | 0.736 | 1.031 | 1.178 | 1.252 | 1.325 | 1.472 | 1.620 | 1.767 | 1.914 | 2.061 | 2.209 | 2.356 | 2.945 | 3.681 | 4.417 | 5.154 |
| 17 | 38.0 | 20 | $2.64 \mathrm{E}+05$ | 0.831 | 1.164 | 1.330 | 1.413 | 1.496 | 1.662 | 1.828 | 1.995 | 2.161 | 2.327 | 2.493 | 2.660 | 3.325 | 4.156 | 4.987 | 5.818 |
| 18 | 40.3 |  | $2.80 \mathrm{E}+05$ | 0.932 | 1.305 | 1.491 | 1.584 | 1.677 | 1.864 | 2.050 | 2.236 | 2.423 | 2.609 | 2.795 | 2.982 | 3.727 | 4.659 | 5.591 | 6.523 |
| 19 | 42.5 |  | $2.95 \mathrm{E}+05$ | 1.038 | 1.453 | 1.661 | 1.765 | 1.869 | 2.076 | 2.284 | 2.492 | 2.699 | 2.907 | 3.115 | 3.322 | 4.153 | 5.191 | 6.229 | 7.267 |
| 20 | 44.7 |  | $3.11 \mathrm{E}+05$ | 1.150 | 1.611 | 1.841 | 1.956 | 2.071 | 2.301 | 2.531 | 2.761 | 2.991 | 3.221 | 3.451 | 3.681 | 4.601 | 5.752 | 6.902 | 8.053 |
| 21 | 47.0 |  | 3.26E+05 | 1.268 | 1.776 | 2.029 | 2.156 | 2.283 | 2.537 | 2.790 | 3.044 | 3.298 | 3.551 | 3.805 | 4.058 | 5.073 | 6.341 | 7.610 | 8.878 |
| 22 | 49.2 | 25 | 3.42E+05 | 1.392 | 1.949 | 2.227 | 2.366 | 2.505 | 2.784 | 3.062 | 3.341 | 3.619 | 3.897 | 4.176 | 4.454 | 5.568 | 6.960 | 8.352 | 9.744 |
| 23 | 51.4 |  | $3.57 \mathrm{E}+05$ | 1.521 | 2.130 | 2.434 | 2.586 | 2.738 | 3.043 | 3.347 | 3.651 | 3.956 | 4.260 | 4.564 | 4.868 | 6.085 | 7.607 | 9.128 | 10.649 |
| 24 | 53.7 |  | $3.73 \mathrm{E}+05$ | 1.657 | 2.319 | 2.650 | 2.816 | 2.982 | 3.313 | 3.644 | 3.976 | 4.307 | 4.638 | 4.970 | 5.301 | 6.626 | 8.283 | 9.939 | 11.596 |
| 25 | 55.9 |  | $3.88 \mathrm{E}+05$ | 1.797 | 2.516 | 2.876 | 3.056 | 3.235 | 3.595 | 3.954 | 4.314 | 4.673 | 5.033 | 5.392 | 5.752 | 7.190 | 8.987 | 10.785 | 12.582 |
| 26 | 58.2 | 30 | $4.04 \mathrm{E}+05$ | 1.944 | 2.722 | 3.111 | 3.305 | 3.499 | 3.888 | 4.277 | 4.666 | 5.055 | 5.444 | 5.832 | 6.221 | 7.776 | 9.721 | 11.665 | 13.609 |
| 27 | 60.4 |  | $4.19 \mathrm{E}+05$ | 2.097 | 2.935 | 3.354 | 3.564 | 3.774 | 4.193 | 4.612 | 5.032 | 5.451 | 5.870 | 6.290 | 6.709 | 8.386 | 10.483 | 12.579 | 14.676 |
| 28 | 62.6 |  | $4.35 \mathrm{E}+05$ | 2.255 | 3.157 | 3.608 | 3.833 | 4.058 | 4.509 | 4.960 | 5.411 | 5.862 | 6.313 | 6.764 | 7.215 | 9.019 | 11.274 | 13.528 | 15.783 |
| 29 | 64.9 |  | $4.50 \mathrm{E}+05$ | 2.419 | 3.386 | 3.870 | 4.112 | 4.354 | 4.837 | 5.321 | 5.805 | 6.288 | 6.772 | 7.256 | 7.740 | 9.675 | 12.093 | 14.512 | 16.930 |
| 30 | 67.1 |  | $4.66 \mathrm{E}+05$ | 2.588 | 3.624 | 4.141 | 4.400 | 4.659 | 5.177 | 5.694 | 6.212 | 6.730 | 7.247 | 7.765 | 8.283 | 10.353 | 12.942 | 15.530 | 18.118 |
| 31 | 69.3 | 35 | $4.82 \mathrm{E}+05$ | 2.764 | 3.869 | 4.422 | 4.698 | 4.975 | 5.527 | 6.080 | 6.633 | 7.186 | 7.738 | 8.291 | 8.844 | 11.055 | 13.819 | 16.582 | 19.346 |
| 32 | 71.6 |  | $4.97 \mathrm{E}+05$ | 2.945 | 4.123 | 4.712 | 5.006 | 5.301 | 5.890 | 6.479 | 7.068 | 7.657 | 8.246 | 8.835 | 9.424 | ${ }^{11.780}$ | ${ }^{14.725}$ | 17.670 | 20.614 |
| 33 | 73.8 |  | 5.13E+05 | 3.132 | 4.385 | 5.011 | 5.324 | 5.637 | $\underline{6.264}$ | $\underline{6.890}$ | 7.516 | 8.143 | 8.769 | 9.396 | 10.022 | 12.527 | 15.659 | 18.791 | 21.923 |
| 34 | 76.1 |  | $5.28 \mathrm{E}+05$ | 3.325 | 4.654 | 5.319 | 5.652 | 5.984 | 6.649 | 7.314 | 7.979 | 8.644 | 9.309 | 9.974 | 10.639 | 13.298 | 16.623 | 19.947 | 23.272 |
| 35 | 78.3 | 40 | ${ }^{5.44 E+05}$ | ${ }^{3.523}$ | 4.932 | 5.637 | 5.989 | ${ }^{6.341}$ | 7.046 | 7.751 | 8.455 | ${ }^{9.160}$ | 9.864 | 10.569 | 11.274 | 14.092 | 17.615 | 21.138 | 24.661 |
| 36 | 80.5 |  | ${ }^{5.59 E+05}$ | 3.727 | 5.218 | 5.963 | ${ }^{6.336}$ | $\frac{6.709}{}$ | ${ }^{7.454}$ | 8.200 | 8.945 | 9.691 | 10.436 | 11.182 | 11.927 | 14.909 | 18.636 | 22.363 | 26.090 |
| 37 | 82.8 |  | $5.75 \mathrm{E}+05$ | 3.937 | 5.512 | 6.299 | 6.693 | 7.087 | 7.874 | 8.662 | 9.449 | 10.236 | 11.024 | 11.811 | 12.599 | 15.748 | 19.686 | 23.623 | 27.560 |
| 38 | 85.0 |  | $5.90 \mathrm{E}+05$ | 4.153 | 5.814 | 6.644 | 7.060 | 7.475 | 8.306 | 9.136 | 9.967 | 10.797 | 11.628 | 12.458 | 13.289 | 16.611 | 20.764 | 24.917 | 29.070 |
| 39 | 87.2 |  | $6.06 \mathrm{E}+05$ | 4.374 | 6.124 | 6.999 | 7.436 | 7.874 | 8.748 | 9.623 | 10.498 | ${ }^{11.373}$ | 12.248 | 13.123 | 13.998 | 17.497 | 21.871 | 26.245 | 30.620 |
| 40 | 89.5 | 45 | $6.21 \mathrm{E}+05$ | 4.601 | 6.442 | 7.362 | 7.822 | 8.283 | 9.203 | 10.123 | 11.043 | 11.964 | 12.884 | 13.804 | 14.725 | 18.406 | 23.007 | 27.609 | 32.210 |
| 41 | 91.7 |  | $6.37 \mathrm{E}+05$ | 4.834 | 6.768 | 7.735 | 8.218 | 8.702 | 9.669 | 10.636 | 11.603 | 12.569 | 13.536 | 14.503 | 15.470 | 19.338 | 24.172 | 29.006 | 33.841 |
| 42 | 94.0 |  | ${ }^{6.52 E+05}$ | 5.073 | 7.102 | 8.117 | 8.624 | 9.132 | ${ }^{10.146}$ | 11.161 | 12.175 | 13.190 | 14.205 | 15.219 | 16.234 | 20.292 | 25.365 | 30.439 | 35.512 |
| 43 | 96.2 |  | 6.68E+05 | 5.318 | 7.445 | 8.508 | 9.040 | 9.572 | 10.635 | 11.699 | 12.762 | ${ }^{13.826}$ | 14.889 | 15.953 | 17.016 | 21.270 | 26.588 | 31.905 | 37.223 |
| 44 | 98.4 | 50 | $6.83 \mathrm{E}+05$ | 5.568 | 7.795 | 8.908 | 9.465 | ${ }^{10.022}$ | 11.135 | 12.249 | ${ }^{13.363}$ | ${ }^{14.476}$ | 15.590 | 16.703 | 17.817 | 22.271 | 27.839 | 33.406 | 38.974 |
| 45 | 100.7 |  | $6.99 \mathrm{E}+05$ | 5.824 | 8.153 | 9.318 | 9.900 | ${ }^{10.483}$ | 11.647 | 12.812 | 13.977 | 15.142 | 16.306 | 17.471 | 18.636 | 23.295 | 29.118 | 34.942 | 40.766 |
| 46 | 102.9 |  | $7.15 \mathrm{E}+05$ | 6.085 | 8.520 | 9.737 | 10.345 | 10.954 | ${ }^{12.171}$ | ${ }^{13.388}$ | 14.605 | ${ }^{15.822}$ | 17.039 | 18.256 | 19.473 | 24.342 | 30.427 | 36.512 | 42.598 |
| 47 | 105.1 |  | $7.30 \mathrm{E}+05$ | 6.353 | 8.894 | 10.165 | 10.800 | 11.435 | ${ }^{12.706}$ | ${ }^{13.976}$ | 15.247 | 16.517 | 17.788 | 19.059 | 20.329 | 25.411 | 31.764 | 38.117 | 44.470 |
| 48 | 107.4 |  | $7.46 \mathrm{E}+05$ | 6.626 | 9.277 | 10.602 | 11.264 | 11.927 | ${ }^{13.252}$ | 14.577 | 15.903 | ${ }^{17.228}$ | 18.553 | 19.878 | 21.203 | 26.504 | 33.130 | 39.756 | ${ }^{46.383}$ |
| 49 | 109.6 | 55 | $7.61 \mathrm{E}+05$ | 6.905 | 9.667 | 11.048 | 11.739 | 12.429 | ${ }^{13.810}$ | ${ }^{15.191}$ | 16.572 | ${ }^{17.953}$ | ${ }^{19.334}$ | 20.715 | ${ }^{22.096}$ | ${ }^{27.620}$ | ${ }^{34.525}$ | 41.430 | 48.335 |
| 50 | 111.8 |  | $7.77 \mathrm{E}+05$ | 7.190 | 10.066 | 11.504 | 12.223 | 12.942 | 14.380 | 15.817 | 17.255 | 18.693 | 20.131 | 21.569 | 23.007 | 28.759 | 35.949 | 43.139 | 50.328 |
| 51 | 114.1 |  | $7.92 \mathrm{E}+05$ | 7.480 | 10.472 | 11.968 | 12.716 | 13.464 | 14.960 | 16.456 | 17.953 | 19.449 | 20.945 | 22.441 | 23.937 | 29.921 | 37.401 | 44.881 | 52.362 |
| 52 | 116.3 |  | ${ }^{8.08 \mathrm{C}+05}$ | 7.776 | $\frac{10.887}{11810}$ | $\frac{12.442}{}$ | ${ }^{13.220}$ | $\frac{13.998}{14.541}$ | $\stackrel{15.553}{ }$ | ${ }^{17.108}$ | ${ }^{18.663}$ | 20.219 | 21.774 | 23.329 | $\stackrel{24.885}{ }$ | ${ }^{31.106}$ | 38.882 | 46.659 | - 54.435 |
| 53 | 118.6 | 60 | ${ }^{8.23 E+05}$ | 8.078 | $\frac{11.310}{11741}$ | $\frac{12.925}{13418}$ | ${ }^{13.733}$ | $\frac{14.541}{1505}$ | $\frac{16.157}{1672}$ | $\frac{17.772}{1849}$ | $\stackrel{19.388}{20.127}$ | $\frac{21.004}{21.804}$ | $\frac{22.620}{23.481}$ | $\stackrel{24.235}{25.158}$ | $\stackrel{25.851}{2683}$ | 32.314 | ${ }_{4}^{40.392}$ | $\stackrel{48.470}{50317}$ | 56.549 <br> 5803 |
| 54 | 120.8 |  | $8.39 \mathrm{E}+05$ | 8.386 | 11.741 | 13.418 | 14.256 | 15.095 | 16.772 | 18.449 | 20.127 | 21.804 | 23.481 | 25.158 | 26.836 | 33.545 | 41.931 | 50.317 | 58.703 |

BLOCKAGE LESS THAN $10 \%$

| Diameter Dprojected $(\mathrm{m})$ | $\mathbf{0 . 2 0 7 7 7}$ |
| :--- | ---: |
| Arae Projected $\left.\mathrm{m}^{2}\right)$ | $\mathbf{0 . 0 3 3 9 0}$ |
| Density $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ | $\mathbf{1 . 2}$ |
| Mu $\left(\mathrm{N}-\mathrm{s} / \mathrm{m}^{2}\right)$ | $\mathbf{1 . 8 0 E - 0 5}$ |
| Kinematic Viscosity | $\mathbf{1 . 5 0 \mathrm { E } - 0 5}$ |


| Kinematic Viscosity |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | 0.5 | 0.7 | 0.8 | 0.85 | 0.9 | 1 | 1.1 | 1.2 | 1.3 | 1.4 | 1.5 | 1.6 | 2 | 2.5 | 3 | 3.5 |
| $\frac{\text { Tunnel Speed }}{\mathrm{m} / \mathrm{s}}$ | MPH | Hz | RE\# | $\stackrel{\text { Force }}{\text { lbs }}$ | Ibs | Ibs | lbs | Ibs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs | lbs |
| 0 | 0.0 |  | $0.00 \mathrm{E}+00$ | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 |
| 1 | 2.2 |  | 1.39E+04 | 0.002 | 0.003 | 0.004 | 0.004 | 0.004 | 0.005 | 0.005 | 0.005 | 0.006 | 0.006 | 0.007 | 0.007 | 0.009 | 0.011 | 0.014 | 0.016 |
| 2 | 4.5 | 2 | $2.77 \mathrm{E}+04$ | 0.009 | 0.013 | 0.015 | 0.016 | 0.016 | 0.018 | 0.020 | 0.022 | 0.024 | 0.026 | 0.027 | 0.029 | 0.037 | 0.046 | 0.055 | 0.064 |
| 3 | 6.7 | 2 | $4.16 \mathrm{E}+04$ | 0.021 | 0.029 | 0.033 | 0.035 | 0.037 | 0.041 | 0.045 | 0.049 | 0.054 | 0.058 | 0.062 | 0.066 | 0.082 | 0.103 | 0.123 | 0.144 |
| 4 | 8.9 | 4 | $5.54 \mathrm{E}+04$ | 0.037 | 0.051 | 0.059 | 0.062 | 0.066 | 0.073 | 0.080 | 0.088 | 0.095 | 0.102 | 0.110 | 0.117 | 0.146 | 0.183 | 0.220 | 0.256 |
| 5 | 11.2 |  | $6.93 \mathrm{E}+04$ | 0.057 | 0.080 | 0.091 | 0.097 | 0.103 | 0.114 | 0.126 | 0.137 | 0.149 | 0.160 | 0.171 | 0.183 | 0.229 | 0.286 | 0.343 | 0.400 |
| 6 | 13.4 |  | 8.31E+04 | 0.082 | 0.115 | 0.132 | 0.140 | 0.148 | 0.165 | 0.181 | 0.198 | 0.214 | 0.230 | 0.247 | 0.263 | 0.329 | 0.412 | 0.494 | 0.576 |
| 7 | 15.7 |  | $9.70 \mathrm{E}+04$ | 0.112 | 0.157 | 0.179 | 0.190 | 0.202 | 0.224 | 0.246 | 0.269 | 0.291 | 0.314 | 0.336 | 0.359 | 0.448 | 0.560 | 0.672 | 0.784 |
| 8 | 17.9 |  | 1.11E+05 | 0.146 | 0.205 | 0.234 | 0.249 | 0.263 | 0.293 | 0.322 | 0.351 | 0.380 | 0.410 | 0.439 | 0.468 | 0.585 | 0.732 | 0.878 | 1.024 |
| 9 | 20.1 |  | $1.25 \mathrm{E}+05$ | 0.185 | 0.259 | 0.296 | 0.315 | 0.333 | 0.370 | 0.407 | 0.445 | 0.482 | 0.519 | 0.556 | 0.593 | 0.741 | 0.926 | 1.111 | 1.297 |
| 10 | 22.4 |  | 1.39E+05 | 0.229 | 0.320 | 0.366 | 0.389 | 0.412 | 0.457 | 0.503 | 0.549 | 0.595 | 0.640 | 0.686 | 0.732 | 0.915 | 1.143 | 1.372 | 1.601 |
| 11 | 24.6 |  | $1.52 \mathrm{E}+05$ | 0.277 | 0.387 | 0.443 | 0.470 | 0.498 | 0.553 | 0.609 | 0.664 | 0.719 | 0.775 | 0.830 | 0.885 | 1.107 | 1.383 | ${ }^{1.660}$ | 1.937 |
| 12 | 26.8 |  | $1.66 \mathrm{E}+05$ | 0.329 | 0.461 | 0.527 | 0.560 | 0.593 | 0.659 | 0.724 | 0.790 | 0.856 | 0.922 | 0.988 | 1.054 | 1.317 | 1.646 | 1.976 | 2.305 |
| 13 | 29.1 | 15 | $1.80 \mathrm{E}+05$ | 0.386 | 0.541 | 0.618 | 0.657 | 0.696 | 0.773 | 0.850 | 0.927 | 1.005 | 1.082 | 1.159 | 1.237 | 1.546 | 1.932 | 2.319 | 2.705 |
| 14 | 31.3 |  | $1.94 \mathrm{E}+05$ | 0.448 | 0.627 | 0.717 | 0.762 | 0.807 | 0.896 | 0.986 | 1.076 | 1.165 | 1.255 | 1.345 | 1.434 | 1.793 | 2.241 | 2.689 | 3.137 |
| 15 | 33.6 |  | $2.08 \mathrm{E}+05$ | 0.514 | 0.720 | 0.823 | 0.875 | 0.926 | 1.029 | 1.132 | 1.235 | 1.338 | 1.441 | 1.543 | 1.646 | 2.058 | 2.572 | 3.087 | 3.601 |
| 16 | 35.8 |  | $2.22 \mathrm{E}+05$ | 0.585 | 0.820 | 0.937 | 0.995 | 1.054 | 1.171 | 1.288 | 1.405 | 1.522 | 1.639 | 1.756 | 1.873 | 2.341 | 2.927 | 3.512 | 4.098 |
| 17 | 38.0 | 20 | $2.35 \mathrm{E}+05$ | 0.661 | 0.925 | 1.057 | 1.123 | 1.189 | 1.322 | 1.454 | 1.586 | 1.718 | 1.850 | 1.982 | 2.115 | 2.643 | 3.304 | 3.965 | 4.626 |
| 18 | 40.3 |  | $2.49 \mathrm{E}+05$ | 0.741 | 1.037 | 1.185 | 1.259 | 1.334 | 1.482 | 1.630 | 1.778 | 1.926 | 2.074 | 2.223 | 2.371 | 2.963 | 3.704 | 4.445 | 5.186 |
| 19 | 42.5 |  | $2.63 \mathrm{E}+05$ | 0.825 | 1.156 | 1.321 | 1.403 | 1.486 | 1.651 | 1.816 | 1.981 | 2.146 | 2.311 | ${ }^{2.476}$ | ${ }^{2.641}$ | 3.302 | 4.127 | 4.953 | 5.778 |
| 20 | 44.7 |  | $2.77 \mathrm{E}+05$ | 0.915 | 1.280 | 1.463 | 1.555 | 1.646 | 1.829 | 2.012 | 2.195 | 2.378 | 2.561 | 2.744 | 2.927 | 3.659 | 4.573 | 5.488 | 6.402 |
| 21 | 47.0 |  | $2.91 \mathrm{E}+05$ | 1.008 | 1.412 | 1.613 | 1.714 | 1.815 | 2.017 | 2.218 | 2.420 | 2.622 | 2.823 | 3.025 | 3.227 | 4.034 | 5.042 | 6.050 | 7.059 |
| 22 | 49.2 | 25 | $3.05 \mathrm{E}+05$ | 1.107 | 1.549 | 1.771 | 1.881 | 1.992 | 2.213 | 2.435 | 2.656 | 2.877 | 3.099 | 3.320 | 3.541 | 4.427 | 5.534 | 6.640 | 7.747 |
| 23 | 51.4 |  | 3.19E+05 | 1.210 | 1.693 | 1.935 | 2.056 | 2.177 | 2.419 | 2.661 | 2.903 | ${ }^{3.145}$ | 3.387 | ${ }^{3.629}$ | 3.871 | 4.838 | 6.048 | 7.258 | 8.467 |
| 24 | 53.7 |  | $3.32 \mathrm{E}+05$ | 1.317 | 1.844 | 2.107 | 2.239 | 2.371 | 2.634 | 2.898 | 3.161 | 3.424 | 3.688 | 3.951 | 4.215 | 5.268 | 6.585 | 7.902 | 9.220 |
| 25 | 55.9 |  | $3.46 \mathrm{E}+05$ | 1.429 | 2.001 | 2.287 | 2.430 | 2.572 | 2.858 | 3.144 | 3.430 | 3.716 | 4.002 | 4.287 | 4.573 | 5.717 | 7.146 | 8.575 | 10.004 |
| 26 | 58.2 | 30 | $3.60 \mathrm{E}+05$ | 1.546 | 2.164 | 2.473 | 2.628 | 2.782 | 3.091 | 3.401 | 3.710 | 4.019 | 4.328 | 4.637 | 4.946 | 6.183 | 7.729 | 9.274 | 10.820 |
| 27 | 60.4 |  | $3.74 \mathrm{E}+05$ | 1.667 | 2.334 | 2.667 | 2.834 | 3.000 | 3.334 | 3.667 | 4.001 | 4.334 | 4.667 | 5.001 | 5.334 | 6.668 | 8.335 | 10.002 | 11.669 |
| 28 | 62.6 |  | $3.88 \mathrm{E}+05$ | 1.793 | 2.510 | 2.868 | 3.048 | 3.227 | 3.585 | 3.944 | 4.302 | 4.661 | 5.020 | 5.378 | 5.737 | 7.171 | 8.963 | 10.756 | 12.549 |
| 29 | 64.9 |  | $4.02 \mathrm{E}+05$ | 1.923 | 2.692 | 3.077 | 3.269 | 3.461 | 3.846 | 4.231 | 4.615 | 5.000 | 5.384 | 5.769 | 6.154 | 7.692 | 9.615 | 11.538 | 13.461 |
| 30 | 67.1 |  | $4.16 \mathrm{E}+05$ | 2.058 | 2.881 | 3.293 | 3.498 | 3.704 | 4.116 | 4.527 | 4.939 | 5.351 | 5.762 | 6.174 | 6.585 | 8.232 | 10.290 | 12.348 | 14.406 |
| 31 | 69.3 | 35 | $4.29 \mathrm{E}+05$ | 2.197 | 3.076 | 3.516 | 3.736 | 3.955 | 4.395 | 4.834 | 5.274 | 5.713 | 6.153 | 6.592 | 7.032 | 8.790 | 10.987 | 13.185 | 15.382 |
| 32 | 71.6 |  | $4.43 \mathrm{E}+05$ | 2.341 | 3.278 | 3.746 | 3.981 | 4.215 | 4.683 | 5.151 | 5.620 | 6.088 | 6.556 | 7.024 | 7.493 | 9.366 | 11.707 | 14.049 | 16.390 |
| 33 | 73.8 |  | $4.57 \mathrm{E}+05$ | 2.490 | 3.486 | 3.984 | 4.233 | 4.482 | 4.980 | 5.478 | 5.976 | 6.474 | 6.972 | 7.470 | 7.968 | 9.960 | 12.451 | 14.941 | 17.431 |
| 34 | 76.1 |  | $4.711 \mathrm{E}+05$ | 2.643 | 3.701 | 4.229 | 4.494 | 4.758 | 5.287 | 5.815 | 6.344 | 6.873 | 7.401 | 7.930 | 8.459 | 10.573 | 13.217 | 15.860 | 18.503 |
| 35 | 78.3 | 40 | $4.85 \mathrm{E}+05$ | 2.801 | 3.922 | 4.482 | 4.762 | 5.042 | 5.602 | 6.162 | 6.723 | 7.283 | 7.843 | 8.403 | 8.963 | 11.204 | 14.005 | 16.807 | 19.608 |
| 36 | 80.5 |  | $4.99 \mathrm{E}+05$ | 2.963 | 4.149 | 4.741 | 5.038 | 5.334 | 5.927 | 6.520 | 7.112 | 7.705 | 8.298 | 8.890 | 9.483 | 11.854 | 14.817 | 17.781 | 20.744 |
| 37 | 82.8 |  | $5.12 \mathrm{E}+05$ | 3.130 | 4.382 | 5.009 | 5.322 | 5.635 | 6.261 | 6.887 | 7.513 | 8.139 | 8.765 | 9.391 | 10.017 | 12.521 | 15.652 | 18.782 | 21.912 |
| 38 | 85.0 |  | $5.26 \mathrm{E}+05$ | 3.302 | 4.623 | 5.283 | 5.613 | 5.943 | 6.604 | 7.264 | 7.924 | 8.585 | 9.245 | 9.906 | 10.566 | 13.207 | 16.509 | 19.811 | 23.113 |
| 39 | 87.2 |  | 5.40E+05 | 3.478 | 4.869 | 5.565 | 5.912 | 6.260 | 6.956 | 7.651 | 8.347 | 9.043 | 9.738 | 10.434 | 11.129 | 13.912 | 17.390 | 20.868 | 24.345 |
| 40 | 89.5 | 45 | $5.54 \mathrm{E}+05$ | 3.659 | 5.122 | 5.854 | 6.220 | 6.585 | 7.317 | 8.049 | 8.781 | 9.512 | 10.244 | 10.976 | 11.707 | 14.634 | 18.293 | 21.951 | 25.610 |
| 41 | 91.7 |  | $5.68 \mathrm{E}+05$ | 3.844 | 5.381 | 6.150 | 6.534 | $\frac{6.919}{7}$ | 7.688 | 8.456 | 9.225 | 9.994 | 10.763 | 11.531 | ${ }^{12.300}$ | ${ }^{15.375}$ | 19.219 | ${ }^{23.063}$ |  |
| 42 | 94.0 |  | $5.82 \mathrm{E}+05$ | 4.034 | 5.647 | 6.454 | 6.857 | 7.260 | 8.067 | 8.874 | 9.681 | 10.487 | 11.294 | 12.101 | 12.907 | 16.134 | 20.168 | 24.201 | 28.235 |
| 43 | 96.2 |  | $5.96 \mathrm{E}+05$ | 4.228 | 5.919 | 6.765 | 7.187 | 7.610 | 8.456 | 9.301 | 10.147 | 10.993 | 11.838 | 12.684 | 13.529 | 16.912 | 21.140 | 25.368 | 29.595 |
| 44 | 98.4 | 50 | $6.09 \mathrm{E}+05$ | 4.427 | 6.198 | 7.083 | 7.526 | 7.968 | 8.854 | 9.739 | 10.624 | 11.510 | 12.395 | 13.281 | 14.166 | 17.707 | 22.134 | 26.561 | 30.988 |
| 45 | 100.7 |  | ${ }^{6.23 E+05}$ | 4.630 | ${ }^{6.483}$ | 7.409 | 7.872 | 8.335 | 9.261 | 10.187 | $\frac{11.113}{}$ | 12.039 | 12.965 | 13.891 | 14.817 | ${ }^{18.521}$ | 23.152 | 27.782 | 32.413 |
| 46 | 102.9 |  | $6.37 \mathrm{E}+05$ | 4.838 | 6.774 | 7.742 | 8.225 | 8.709 | 9.677 | 10.645 | 11.612 | 12.580 | 13.548 | 14.515 | 15.483 | 19.354 | 24.192 | 29.031 | 33.869 |
| 47 | 105.1 |  | $6.51 \mathrm{E}+05$ | 5.051 | 7.072 | 8.082 | 8.587 | 9.092 | 10.102 | 11.112 | 12.123 | 13.133 | 14.143 | 15.153 | 16.164 | 20.204 | 25.256 | 30.307 | 35.358 |
| 48 | 107.4 |  | $6.65 \mathrm{E}+05$ | 5.268 | 7.376 | 8.429 | 8.956 | 9.483 | 10.537 | 11.590 | 12.644 | 13.698 | 14.751 | 15.805 | 16.859 | 21.073 | 26.342 | 31.610 | 36.878 |
| 49 | 109.6 | 55 | $6.79 \mathrm{E}+05$ | 5.490 | 7.686 | 8.784 | 9.333 | 9.882 | ${ }^{10.980}$ | 12.078 | ${ }^{13.176}$ | 14.274 | 15.372 | 16.470 | 17.568 | 21.961 | 27.451 | 32.941 | 38.431 |
| 50 | 111.8 |  | $6.93 \mathrm{E}+05$ | 5.717 | 8.003 | 9.146 | 9.718 | 10.290 | 11.433 | 12.576 | 13.720 | 14.863 | 16.006 | 17.150 | 18.293 | 22.866 | 28.583 | 34.299 | 40.016 |
| 51 | 114.1 |  | $7.06 \mathrm{E}+05$ | 5.947 | 8.326 | 9.516 | 10.111 | 10.705 | 11.895 | 13.084 | 14.274 | 15.463 | 16.653 | 17.842 | 19.032 | 23.790 | 29.737 | 35.685 | 41.632 |
| 52 | 116.3 |  | $7.20 \mathrm{E}+05$ | 6.183 | 8.656 | 9.893 | 10.511 | 11.129 | 12.366 | ${ }^{13.603}$ | 14.839 | 16.076 | ${ }^{17.312}$ | 18.549 | ${ }^{19.786}$ | 24.732 | 30.915 | 37.098 | 43.281 |
| 53 | 118.6 | 60 | $7.34 \mathrm{E}+05$ | 6.423 | 8.992 | 10.277 | 10.919 | 11.562 | 12.846 | 14.131 | 15.415 | 16.700 | 17.985 | 19.269 | 20.554 | 25.692 | 32.115 | 38.538 | 44.961 |
| 54 | 120.8 |  | $7.48 \mathrm{E}+05$ | 6.668 | 9.335 | 10.668 | 11.335 | 12.002 | 13.335 | 14.669 | 16.003 | 17.336 | 18.670 | 20.003 | 21.337 | 26.671 | 33.339 | 40.006 | 46.674 |

Appendix D: Sample of collected drag force data output from the LabVIEW virtual instrument for Model 8 at a freestream tunnel velocity of $\mathbf{1 8} \mathbf{~ m} / \mathrm{s}$. (Please refer to enclosed compact disk for complete data set.)

| Image 8 @ 18 m/s |  |  |
| :---: | :---: | :---: |
| Drag Force (Kg) | Time(s) | Pressure (inH20) |
| 2.173 | 0 | -0.782 |
| 2.402 | 0.638 | -0.782 |
| 2.246 | 1.222 | -0.782 |
| 2.125 | 1.811 | -0.782 |
| 2.161 | 2.398 | -0.782 |
| 2.39 | 2.985 | -0.782 |
| 2.342 | 3.57 | -0.782 |
| 2.294 | 4.16 | -0.782 |
| 2.258 | 4.749 | -0.782 |
| 2.306 | 5.337 | -0.782 |
| 2.21 | 5.927 | -0.782 |
| 2.318 | 6.514 | -0.782 |
| 2.354 | 7.103 | -0.782 |
| 2.137 | 7.693 | -0.782 |
| 2.222 | 8.282 | -0.782 |
| 2.234 | 8.872 | -0.782 |
| 2.161 | 9.462 | -0.782 |
| 2.234 | 10.054 | -0.782 |
| 2.246 | 10.644 | -0.782 |
| 2.161 | 11.236 | -0.782 |
| 2.21 | 11.827 | -0.782 |
| 2.137 | 12.417 | -0.782 |
| 2.27 | 13.009 | -0.782 |
| 2.234 | 13.6 | -0.782 |
| 2.198 | 14.192 | -0.782 |
| 2.222 | 14.783 | -0.782 |
| 2.246 | 15.375 | -0.782 |
| 2.414 | 15.968 | -0.782 |
| 2.234 | 16.559 | -0.782 |
| 2.185 | 17.152 | -0.782 |
| 2.342 | 17.745 | -0.782 |
| 2.45 | 18.338 | -0.782 |
| 2.173 | 18.93 | -0.782 |
| 2.258 | 19.524 | -0.782 |
| 2.246 | 20.117 | -0.782 |
| 2.149 | 20.711 | -0.782 |
| 2.21 | 21.302 | -0.782 |
| 2.27 | 21.898 | -0.782 |
| 2.185 | 22.491 | -0.782 |
| 2.258 | 23.086 | -0.782 |
| 2.198 | 23.678 | -0.782 |
| 2.366 | 24.273 | -0.782 |


| 2.366 | 24.867 | -0.782 |
| :---: | :---: | :---: |
| 2.185 | 25.46 | -0.782 |
| 2.234 | 26.053 | -0.782 |
| 2.246 | 26.647 | -0.782 |
| 2.33 | 27.243 | -0.782 |
| 2.234 | 27.838 | -0.782 |
| 2.318 | 28.432 | -0.782 |
| 2.246 | 29.027 | -0.782 |
| 2.366 | 29.622 | -0.782 |
| 2.39 | 30.217 | -0.782 |
| 2.258 | 30.811 | -0.782 |
| 2.366 | 31.407 | -0.782 |
| 2.366 | 32.001 | -0.782 |
| 2.173 | 32.596 | -0.782 |
| 2.222 | 33.189 | -0.782 |
| 2.173 | 33.786 | -0.782 |
| 2.137 | 34.383 | -0.782 |
| 2.27 | 34.976 | -0.782 |
| 2.354 | 35.574 | -0.782 |
| 2.294 | 36.17 | -0.782 |
| 2.33 | 36.765 | -0.782 |
| 2.306 | 37.361 | -0.782 |
| 2.234 | 37.956 | -0.782 |
| 2.354 | 38.552 | -0.782 |
| 2.318 | 39.147 | -0.782 |
| 2.294 | 39.742 | -0.782 |
| 2.354 | 40.338 | -0.782 |
| 2.294 | 40.933 | -0.782 |
| 2.366 | 41.53 | -0.782 |
| 2.161 | 42.127 | -0.782 |
| 2.089 | 42.722 | -0.782 |
| 2.258 | 43.319 | -0.782 |
| 2.149 | 43.917 | -0.782 |
| 2.125 | 44.512 | -0.782 |
| 2.185 | 45.109 | -0.782 |
| 2.234 | 45.707 | -0.782 |
| 2.294 | 46.302 | -0.782 |
| 2.246 | 46.9 | -0.782 |
| 2.366 | 47.498 | -0.782 |
| 2.306 | 48.094 | -0.782 |
| 2.185 | 48.691 | -0.782 |
| 2.149 | 49.289 | -0.782 |
| 2.234 | 49.886 | -0.782 |
| 2.198 | 50.483 | -0.782 |
| 2.234 | 51.08 | -0.782 |
| 2.198 | 51.676 | -0.782 |
| 2.246 | 52.274 | -0.782 |
| 2.27 | 52.87 | -0.782 |


| 2.198 | 53.468 | -0.782 |
| :---: | :---: | :---: |
| 2.222 | 54.064 | -0.782 |
| 2.258 | 54.661 | -0.782 |
| 2.258 | 55.259 | -0.782 |
| 2.282 | 55.855 | -0.782 |
| 2.27 | 56.451 | -0.782 |
| 2.402 | 57.05 | -0.782 |
| 2.282 | 57.646 | -0.782 |
| 2.27 | 58.244 | -0.782 |
| 2.27 | 58.841 | -0.782 |
| 2.246 | 59.438 | -0.782 |
| 2.173 | 60.036 | -0.782 |
| 2.246 | 60.633 | -0.782 |
| 2.282 | 61.231 | -0.782 |
| 2.438 | 61.83 | -0.782 |
| 2.21 | 62.427 | -0.782 |
| 2.294 | 63.024 | -0.782 |
| 2.234 | 63.622 | -0.782 |
| 2.294 | 64.219 | -0.782 |
| 2.198 | 64.817 | -0.782 |
| 2.089 | 65.415 | -0.782 |
| 2.354 | 66.011 | -0.782 |
| 2.173 | 66.609 | -0.782 |
| 2.198 | 67.207 | -0.782 |
| 2.354 | 67.803 | -0.782 |
| 2.21 | 68.403 | -0.782 |
| 2.294 | 69.002 | -0.782 |
| 2.246 | 69.598 | -0.782 |
| 2.21 | 70.196 | -0.782 |
| 2.354 | 70.794 | -0.782 |
| 2.462 | 71.391 | -0.782 |
| 2.258 | 71.989 | -0.782 |
| 2.222 | 72.586 | -0.782 |
| 2.246 | 73.183 | -0.782 |
| 2.258 | 73.778 | -0.782 |
| 2.318 | 74.38 | -0.782 |
| 2.258 | 74.978 | -0.782 |
| 2.294 | 75.574 | -0.782 |
| 2.366 | 76.173 | -0.782 |
| 2.234 | 76.772 | -0.782 |
| 2.282 | 77.371 | -0.782 |
| 2.426 | 77.969 | -0.782 |
| 2.258 | 78.568 | -0.782 |
| 2.27 | 79.167 | -0.782 |
| 2.161 | 79.764 | -0.782 |
| 2.198 | 80.364 | -0.782 |
| 2.185 | 80.963 | -0.782 |
| 2.125 | 81.561 | -0.782 |


| 2.282 | 82.16 | -0.782 |
| :---: | :---: | :---: |
| 2.161 | 82.758 | -0.782 |
| 2.185 | 83.356 | -0.782 |
| 2.282 | 83.955 | -0.782 |
| 2.378 | 84.554 | -0.782 |
| 2.414 | 85.153 | -0.782 |
| 2.306 | 85.751 | -0.782 |
| 2.246 | 86.35 | -0.782 |
| 2.258 | 86.948 | -0.782 |
| 2.378 | 87.547 | -0.782 |
| 2.294 | 88.147 | -0.782 |
| 2.282 | 88.746 | -0.782 |
| 2.318 | 89.345 | -0.782 |
| 2.222 | 89.945 | -0.782 |
| 2.198 | 90.545 | -0.782 |
| 2.234 | 91.145 | -0.782 |
| 2.234 | 91.75 | -0.782 |
| 2.294 | 92.349 | -0.782 |
| 2.185 | 92.948 | -0.782 |
| 2.137 | 93.549 | -0.782 |
| 2.21 | 94.148 | -0.782 |
| 2.173 | 94.747 | -0.782 |
| 2.45 | 95.346 | -0.782 |
| 2.198 | 95.945 | -0.782 |
| 2.077 | 96.546 | -0.782 |
| 2.21 | 97.144 | -0.782 |
| 2.306 | 97.744 | -0.782 |
| 2.306 | 98.343 | -0.782 |
| 2.21 | 98.943 | -0.782 |
| 2.366 | 99.544 | -0.782 |
| 2.234 | 100.143 | -0.782 |
| 2.318 | 100.743 | -0.782 |
| 2.21 | 101.344 | -0.782 |
| 2.198 | 101.945 | -0.782 |
| 2.234 | 102.545 | -0.782 |
| 2.234 | 103.144 | -0.782 |
| 2.33 | 103.744 | -0.782 |
| 2.354 | 104.345 | -0.782 |
| 2.246 | 104.947 | -0.782 |
| 2.161 | 105.548 | -0.782 |
| 2.234 | 106.148 | -0.782 |
| 2.27 | 106.748 | -0.782 |
| 2.294 | 107.348 | -0.782 |
| 2.258 | 107.949 | -0.782 |
| 2.234 | 108.55 | -0.782 |
| 2.198 | 109.15 | -0.782 |
| 2.258 | 109.752 | -0.782 |
| 2.077 | 110.351 | -0.782 |


| 2.173 | 110.952 | -0.782 |
| :---: | :---: | :---: |
| 2.137 | 111.553 | -0.782 |
| 2.294 | 112.154 | -0.782 |
| 2.222 | 112.755 | -0.782 |
| 2.246 | 113.355 | -0.782 |
| 2.246 | 113.954 | -0.782 |
| 2.234 | 114.556 | -0.782 |
| 2.161 | 115.157 | -0.782 |
| 2.354 | 115.759 | -0.782 |
| 2.258 | 116.359 | -0.782 |
| 2.185 | 116.96 | -0.782 |
| 2.173 | 117.561 | -0.782 |
| 2.149 | 118.162 | -0.782 |
| 2.198 | 118.763 | -0.782 |

## Appendix E: Stock Material Pricing

DELRIN (ACETAL)
Plastics Unlimited
10/31/05
Pricing on Natural Delrin (Round Stock)
10 " dia \$365.33/ft - Standard Length 36"
so on this you would rec. 1 pc 3 ft

* \& 1 pc 1 ft

6" dia x 18" long material in stock
\$181.20/pc
2" dia x 12" material in stock \$27.16/ft
Would have to order the 10 " dia in just 3-4 days to get.
Thanks again
Wendy

## TriStar Plastics

Brian,
Here is the information that you requested.
8" dia. Natural Acetal rod - \$258.00/ft.
9" dia. Natural Acetal rod - \$312.46/ft.
10" dia. Natural Acetal rod - \$382.70/ft.
I would have to order all three of these. It would take 2-3 days for it to hit my dock once we get it on order.

Please let me know if you have any questions or if you need anything else from me.
Thank you,
Brian

```
Brian Parath
Branch Manager
Tri Star Plastics Corp.
phone-800-323-3311, ext. }320
fax - 508-845-1200
cell -508-847-8645
```

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## ALUMINUM

Yarde Metals
11/1/05
Sales Quote:

thank you,
Dave Chisholm
Sales
Yarde Metals
David.Chisholm@Yarde.com
Ph: 800-376-2011
Fx; 603-635-1282

## Pricing from Drop Zone:

Aluminum 6061-T6 Round Stock

| $10.125^{\prime \prime} \times 30.5^{\prime \prime}$ | $\$ 426.70$ |
| :--- | ---: |
| $9.0^{\prime \prime} \times 27.5^{\prime \prime}$ | $\$ 292.40$ |
| $7.2^{\prime \prime} \times 7.375^{\prime \prime}$ | $\$ 49.50$ |

## Rough estimate of stock size needed:

| Stock <br> dia <br> (inches) | max dia. <br> (inches) | height (inches) | boss <br> height <br> (inches) | total <br> height <br> (inches) | stock height <br> rounded up <br> (inches) |
| :--- | :--- | :--- | :--- | :--- | :--- |
| 2 | 1.278 | 5.634 | 3 | 8.63 | 10 |
| 6 | 4.489 | 4.953 | 3 | 7.95 | 9 |
| 6 | 5.482 | 4.151 | 3 | 7.15 | 9 |
| 9 | 7.131 | 3.315 | 3 | 7.31 | 9 |
| 9 | 8.497 | 3.193 | 3 | 7.19 | 9 |
| 9 | 8.802 | 2.457 | 3 | 5.45 | 7 |
| 10 | 9.174 | 2.154 | 3 | 5.15 | 7 |
| 9 | 8.180 | 3.466 | 3 | 7.46 | 9 |

Delrin cost estimate: approx. $\mathbf{\$ 1 , 6 0 0}$
Aluminum cost estimate: approx. \$1,100
Material request correspondence with Dr. Kenneth Desabrais at the Natick Soldier Center:

```
Checking into now. I will get back to you guys in a few days with more
details.
Ken
```

-----Original Message-----
From: Day, Brian P [mailto:brianday@WPI.EDU]
Sent: Friday, November 04, 2005 3:33 PM
To: kenneth.desabrais@natick.army.mil
Cc: dropteam@WPI.EDU
Subject: Material and Tooling Request
Ken,
The machine shop gave us a request for tooling and we also have some initial
material requests so we can get going on our simplest shapes (shape 1 \& 2 ).
Could you order these for us?
The machine shop would like:
From MSC:
ER-11 Collet Extension
MSC \#: 84904523
Cost: \$105.95
Shank dia.: .625"
overall length: 6.67"
Material from Plastics Unlimited:
Sales Rep: Wendy
Natural Delrin
6" dia x 18" long material in stock \$181.20/pc
2" dia x 12" material in stock \$27.16/ft
NOTE: These prices do not include shipping
Thanks,
Brian
Justin
Matt
Guys,
Your first order from Plastics Unlimited is in ME office, I think. I also
have the collet extension from MSC. I'll leave it in the small office
tonight near the wind tunnel where I store all my stuff. Ask Prof. Johari to
let you in the office to get
it.
I'm going to put your second order from Plastics Unlimited in system today.

```
I'll be on-campus next week if you have any questions.
Have a good weekend.
Ken
-----Original Message-----
From: Day, Brian P [mailto:brianday@WPI.EDU]
Sent: Wednesday, November 16, 2005 4:29 PM
To: kenneth.desabrais@natick.army.mil
Cc: dropteam@WPI.EDU
Subject: Additional Material Request
Ken,
In addition to the material you have already ordered, we will need the
following:
Acetal (delrin or equivalent)
ITEM 1:
3 ft length of 9" dia Acetal $312.46/ft
From Tri Star Plastics in Shrewsbury
If you order this size from Plastics Unlimited it will cost $343.22/ft
ITEM 2:
1 ft length 10" dia Acetal $365.33/ft
From Plastics Unlimited in Worcester
If you order this size from Tri Star it will cost $382.70/ft
According to these companies, Acetal stock of this size does not come in
square blocks.
Thanks,
Justin, Matt, Brian
```


[^0]:    $\square$ uncorrected for wall interference

